

# Pumps (and Piping)

Design, Performance and Commissioning Issues

Parallel Pumps and Pump/Motor Interactions



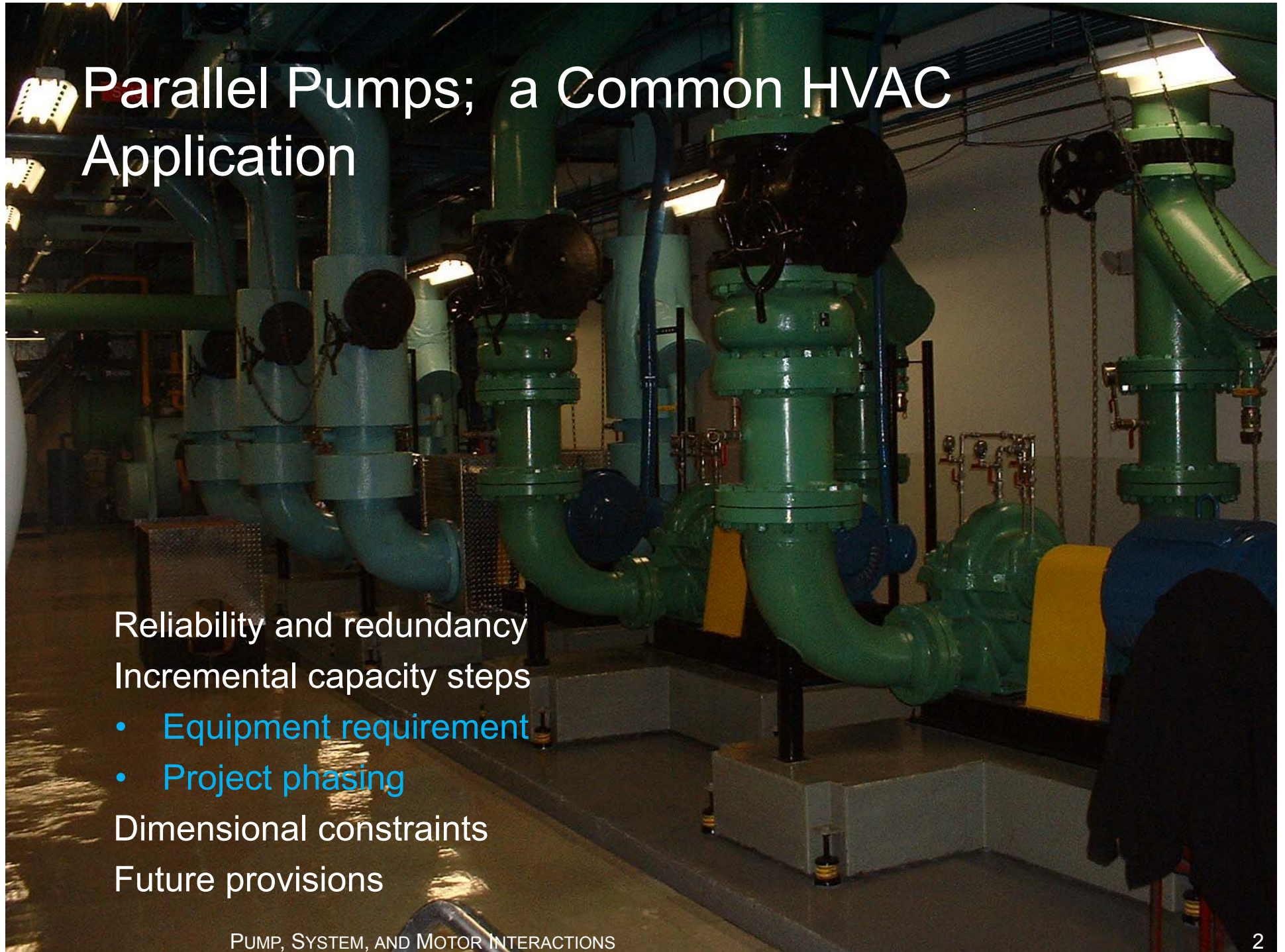
**Instructor:**

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Senior Engineer

Facility Dynamics Engineering

April 4, 2017



# Parallel Pumps; a Common HVAC Application

Reliability and redundancy

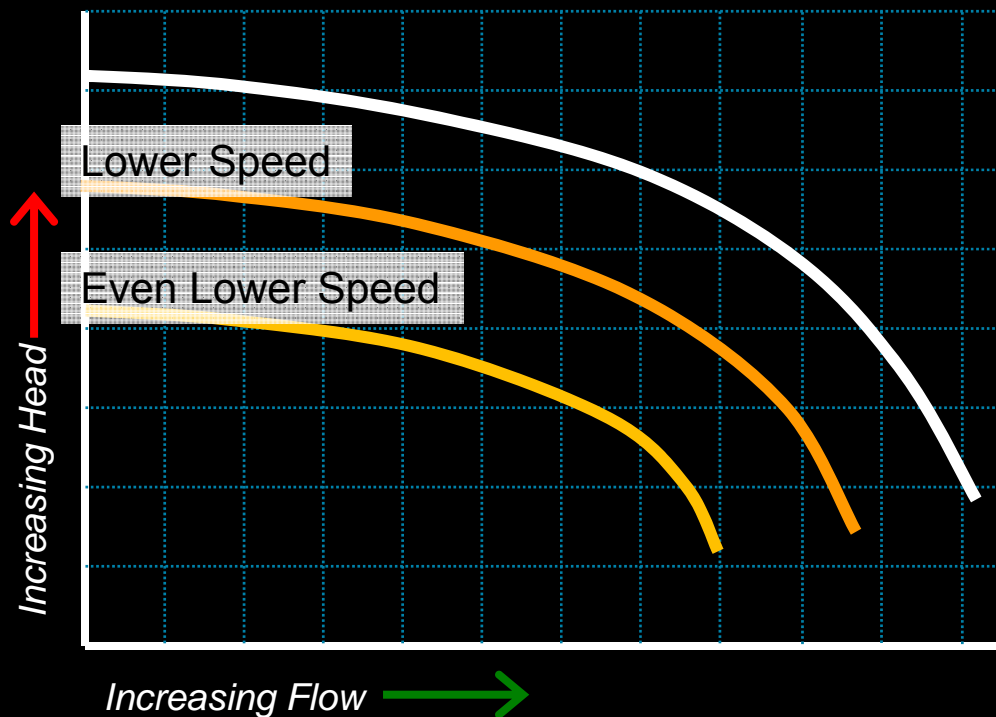
Incremental capacity steps

- Equipment requirement
- Project phasing

Dimensional constraints

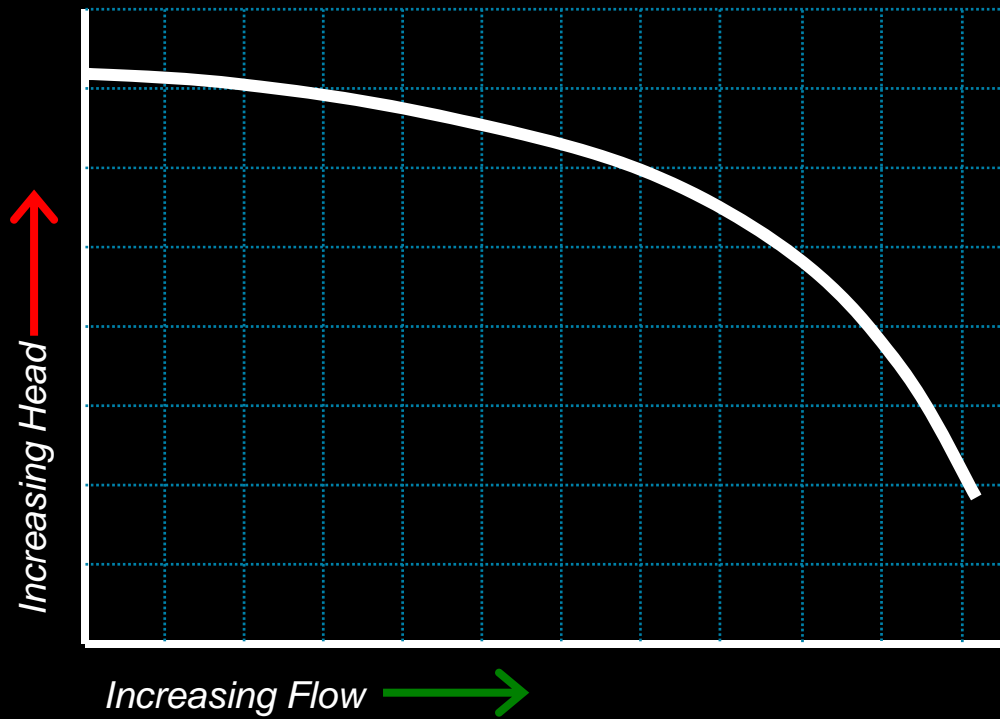
Future provisions

# Parallel Pumps



*Start with the curve for a single pump at a number of speeds*

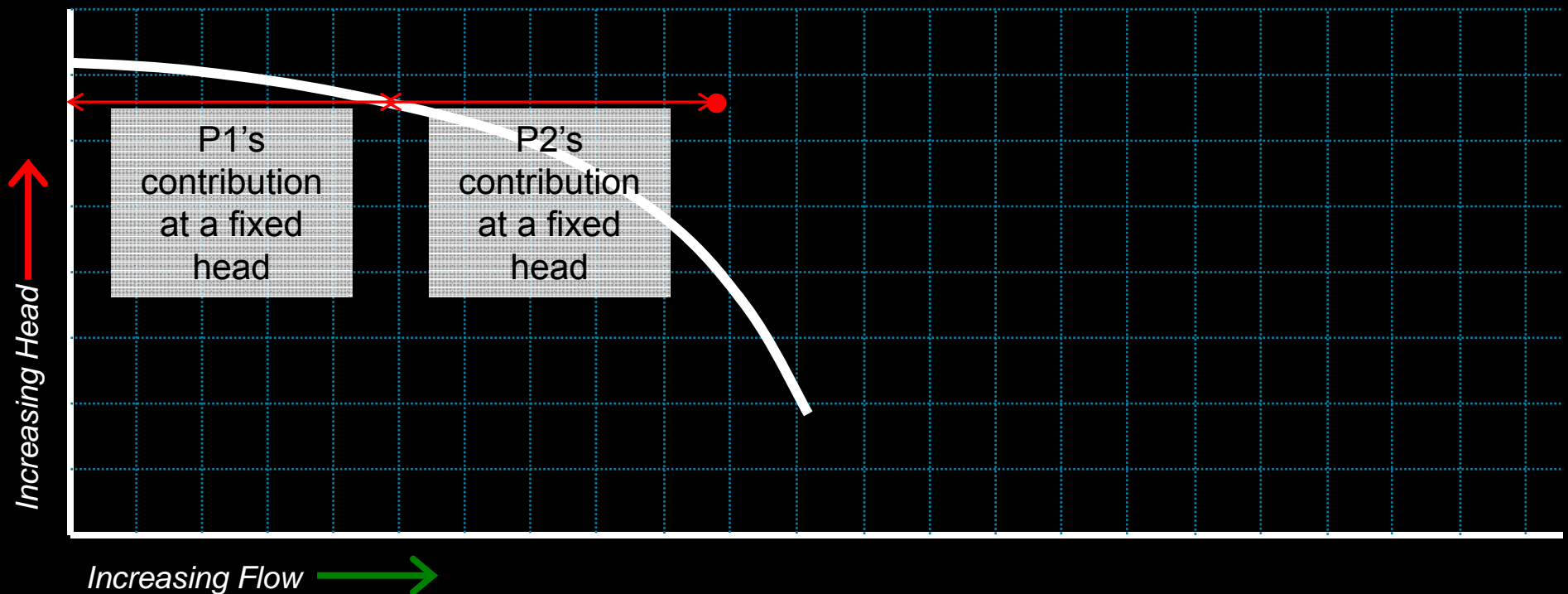
# Parallel Pumps



*Focusing on the full speed curve ...*

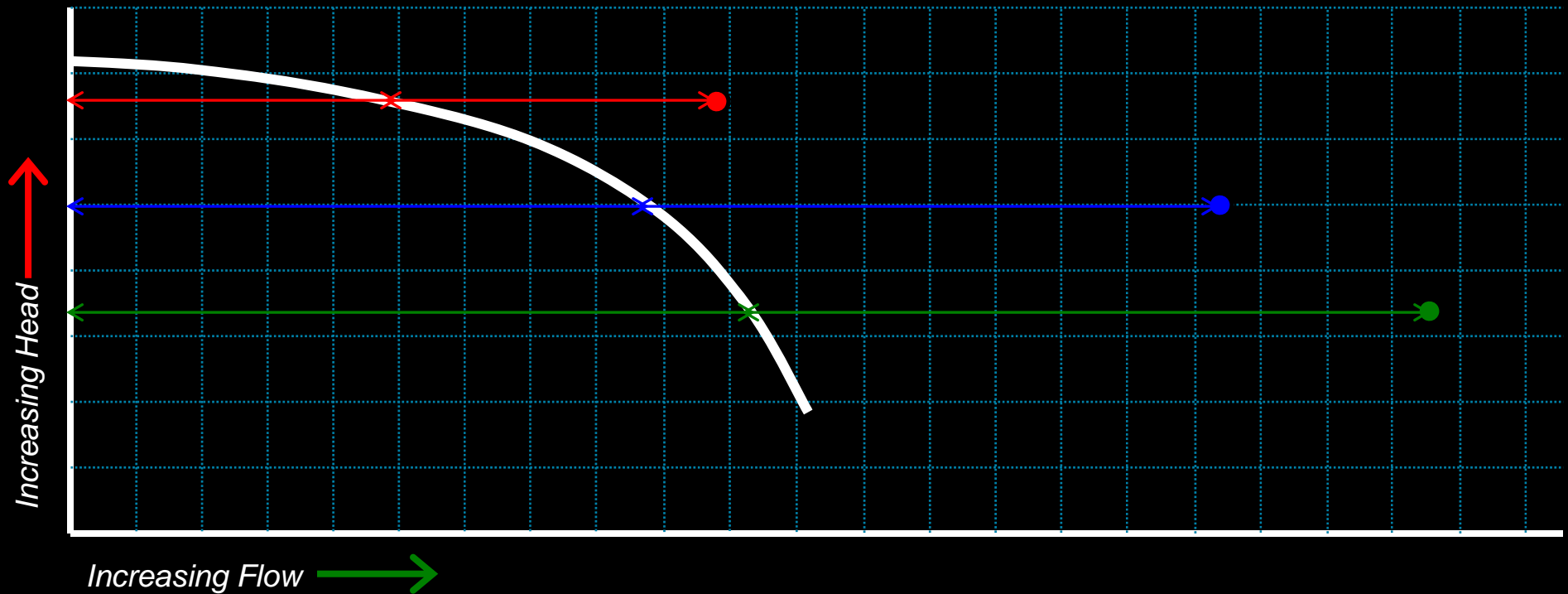


# Parallel Pumps



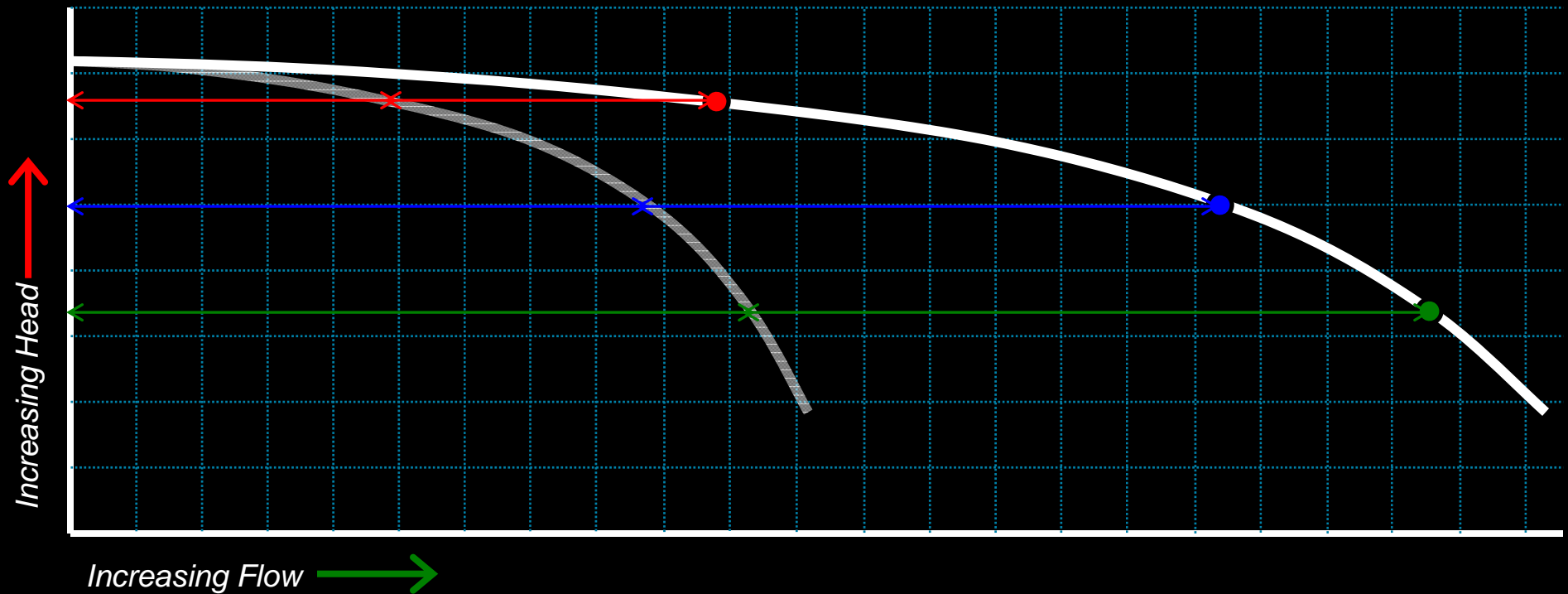
*Construct the curve for the same pump at the same speed operating in parallel by doubling the flow at different heads*

# Parallel Pumps



*Construct the curve for the same pump at the same speed operating in parallel by doubling the flow at different heads*

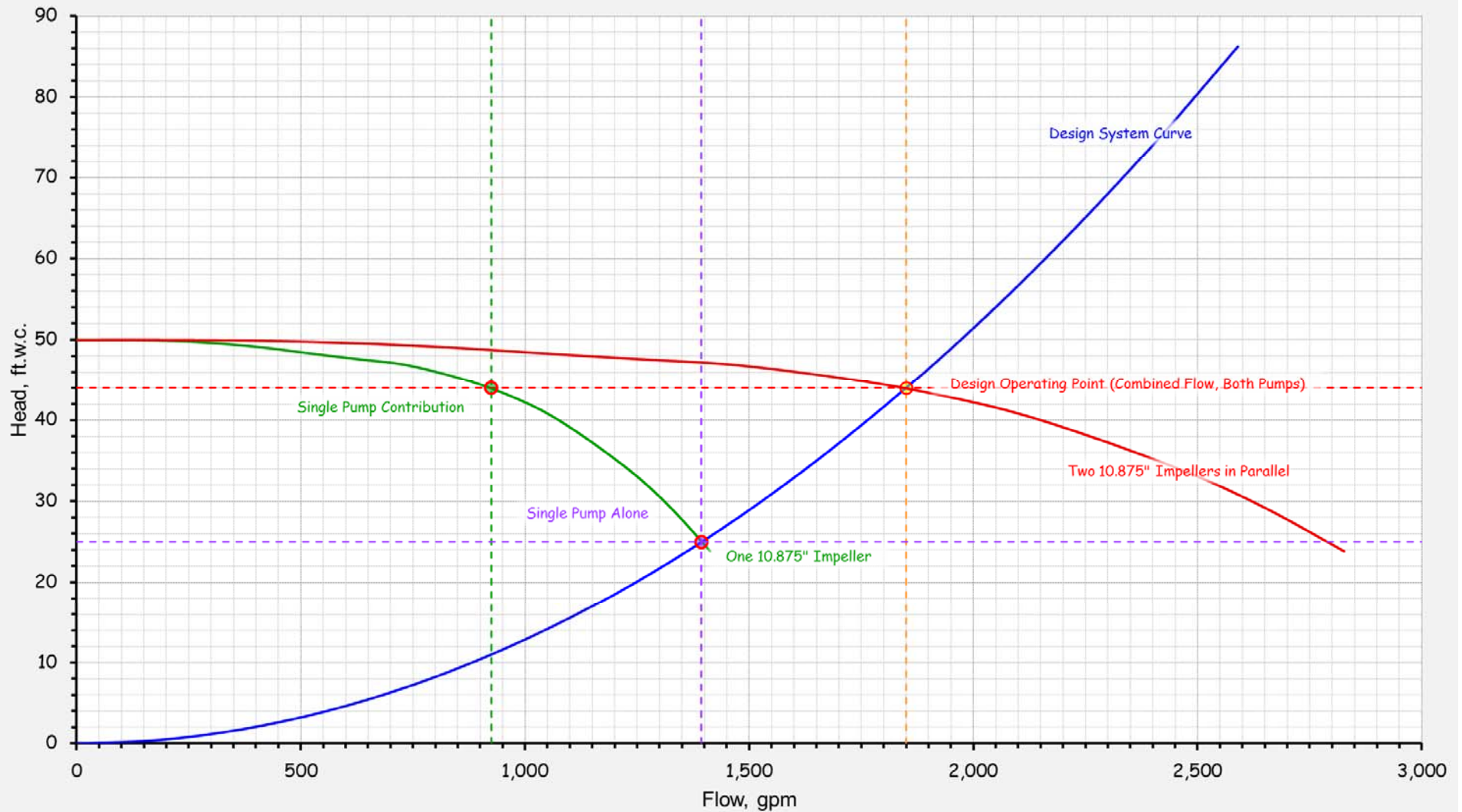
# Parallel Pumps



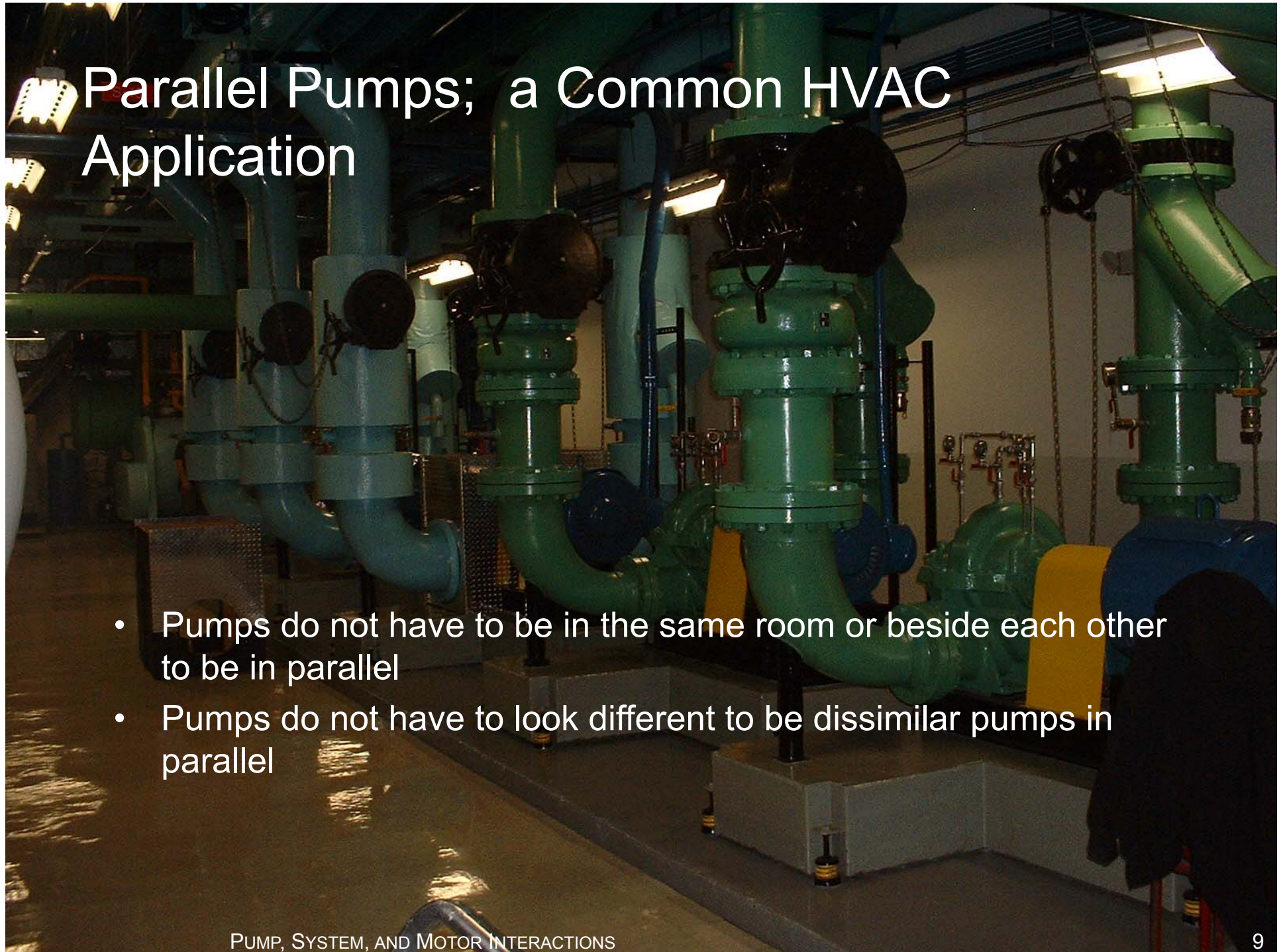
*Construct the curve for the same pump at the same speed operating in parallel by doubling the flow at different heads*

# Parallel Pumps – The Benefits of Digital

Bell and Gossett 1510 6G  
1150 rpm







# Parallel Pumps; a Common HVAC Application

- Pumps do not have to be in the same room or beside each other to be in parallel
- Pumps do not have to look different to be dissimilar pumps in parallel

# Applying Centrifugal Loads to Induction Motors

Centrifugal loads like pumps and fans extract power as a function of torque and speed

- Requirements at the design operating point are a function of the pump or fan design
- Requirements at other points can be projected based on the affinity laws

Induction motor horse power is a function of torque and speed

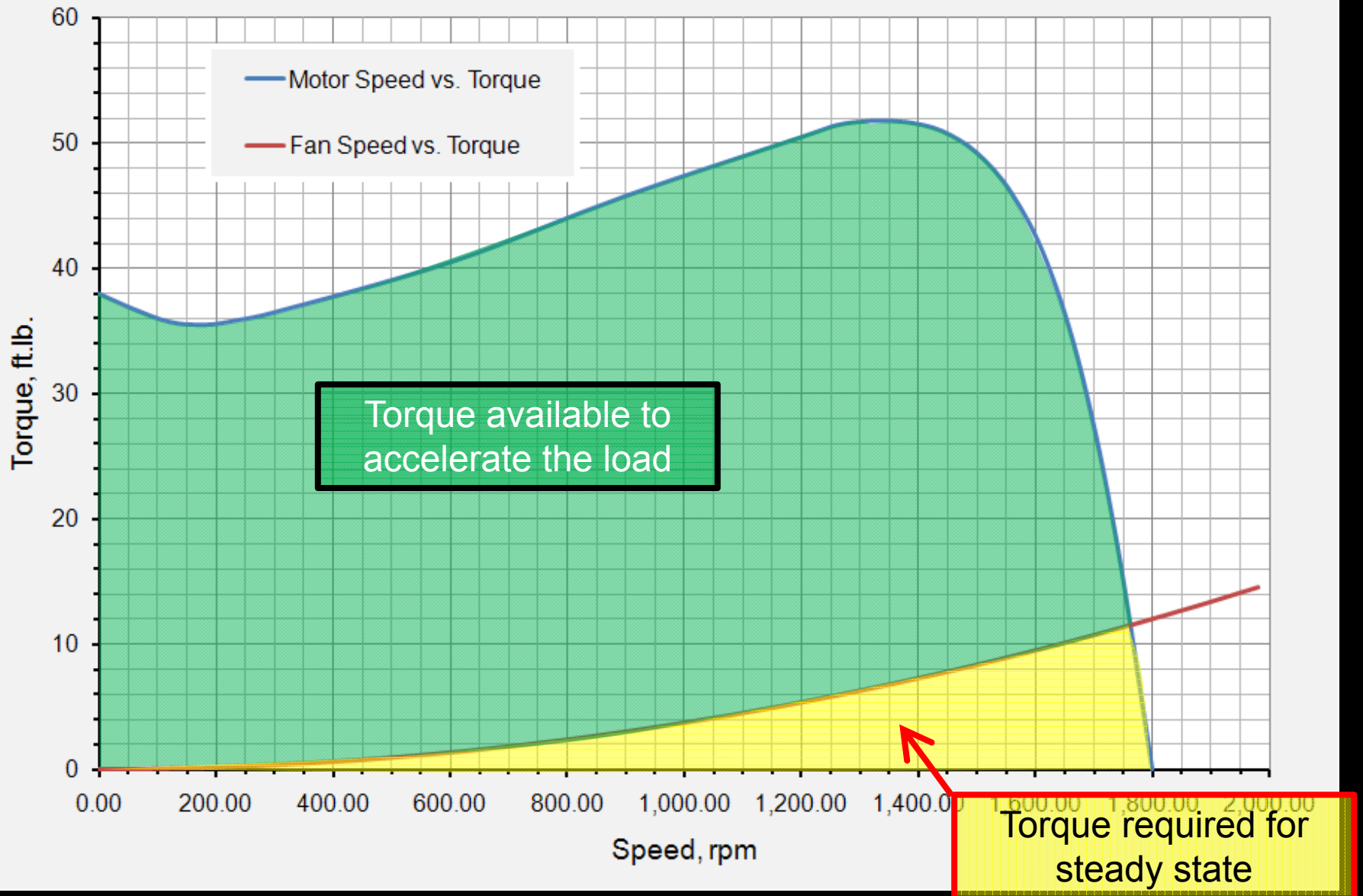
- Relationship is a function of motor design
- Relationship depicted on the torque-speed curve

$$torque_{Speed ?} = torque_{FullSpeed} \times \left( \frac{rpm_{Speed ?}}{rpm_{FullSpeed}} \right)^2$$

## U.S. Motors B421 with Twin City BAE-222

5 hp, 460 vac, 60 hz

DWDI 7,500 cfm at 2.5 in.w.c

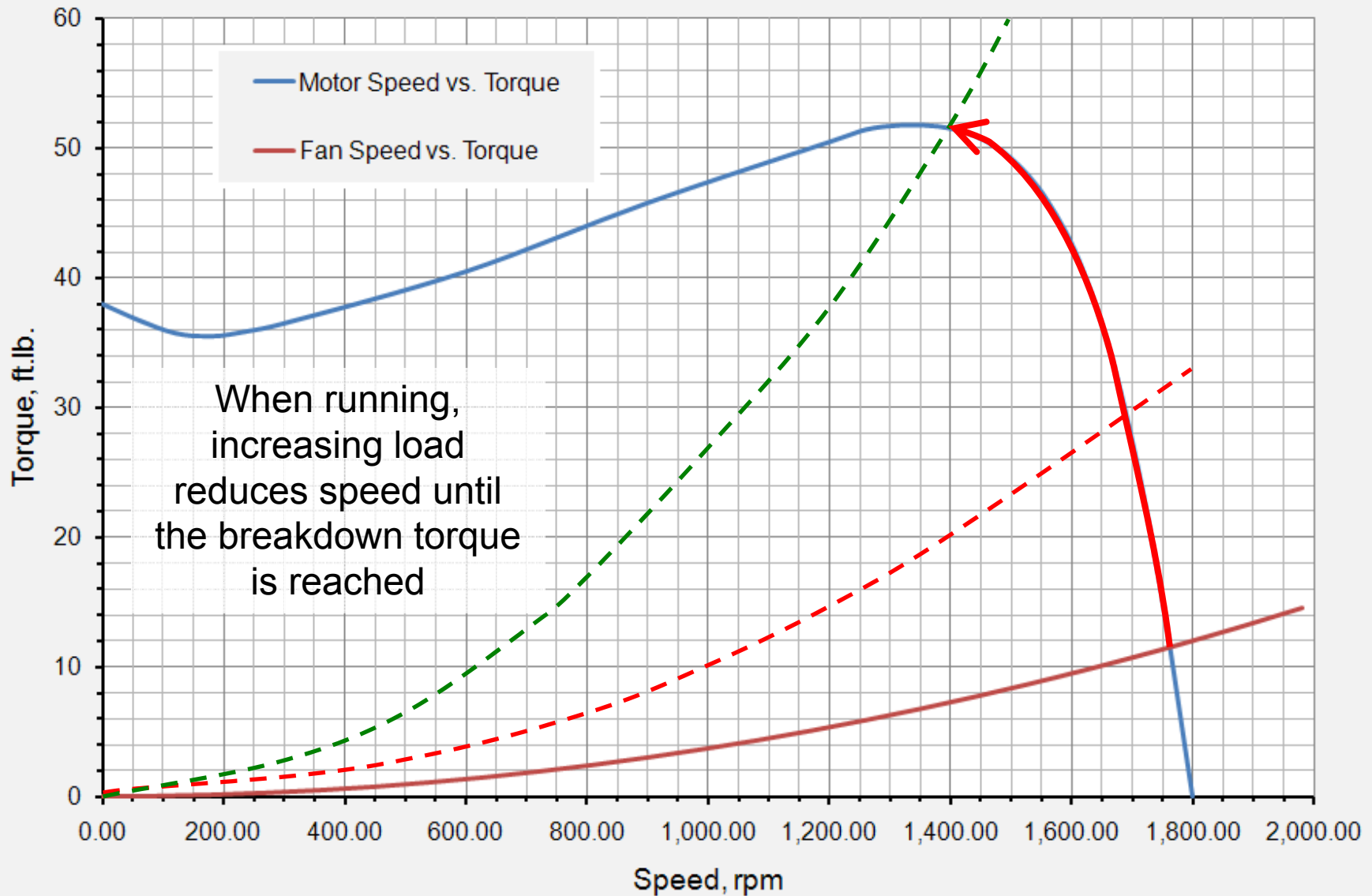




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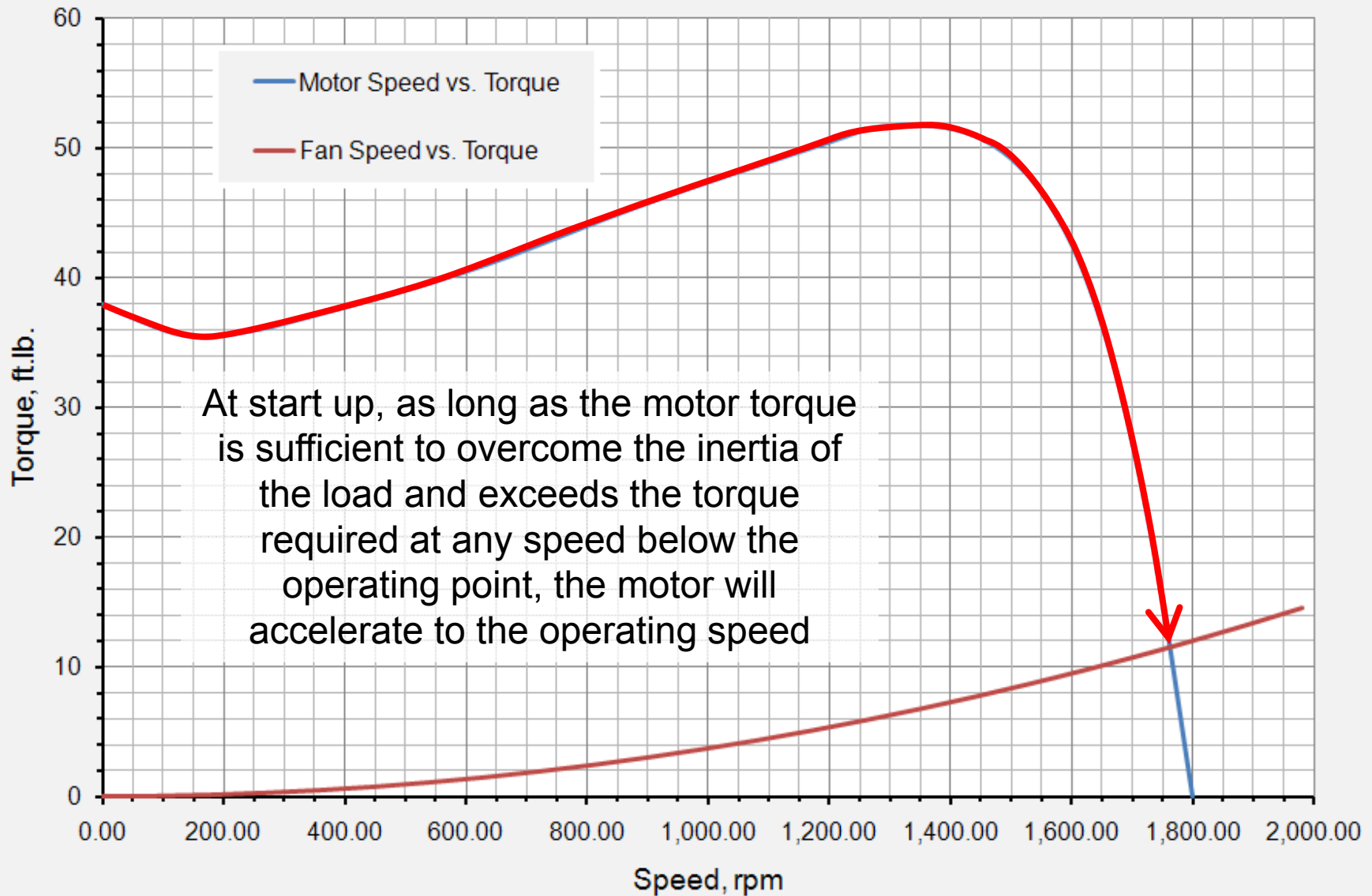
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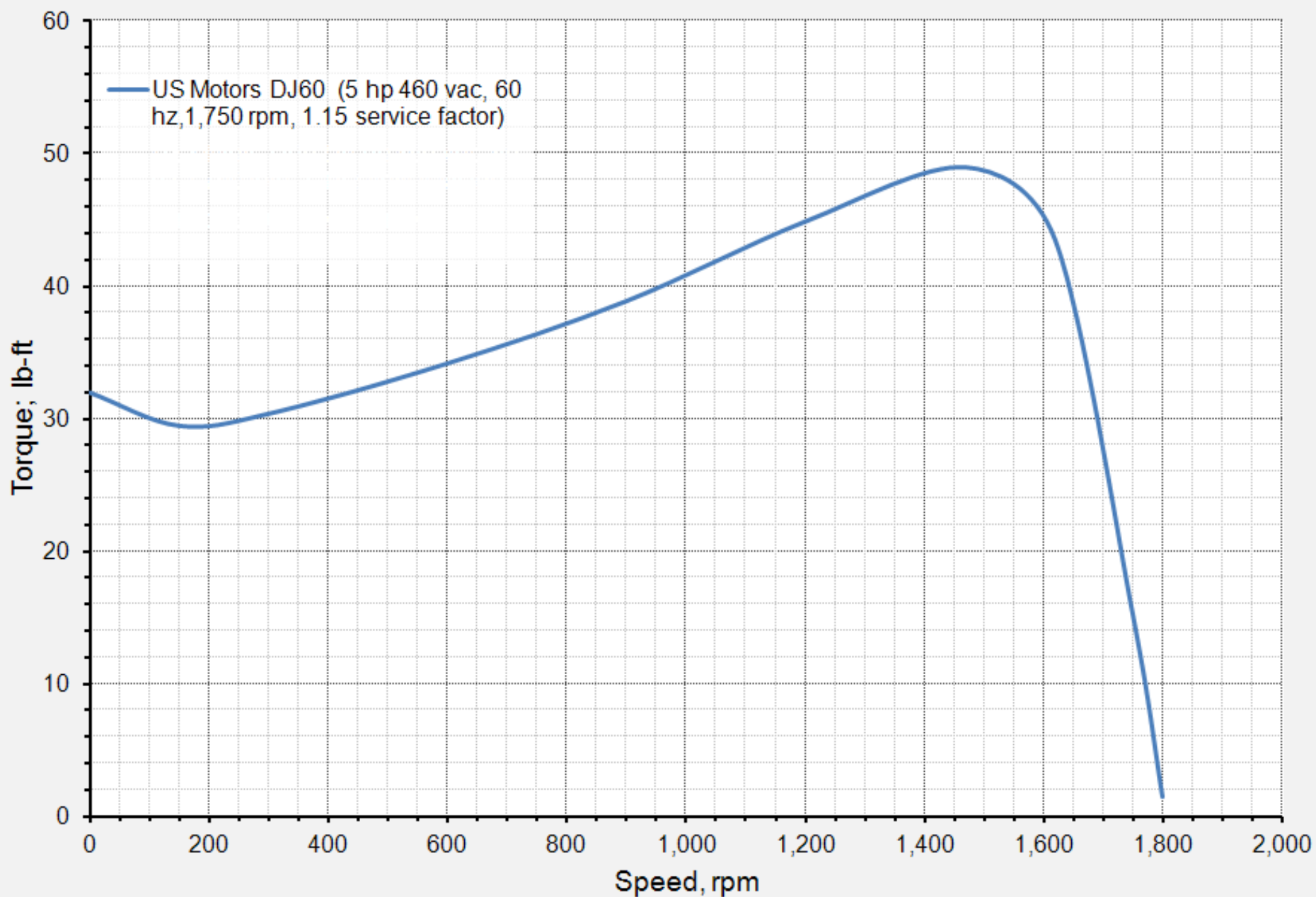
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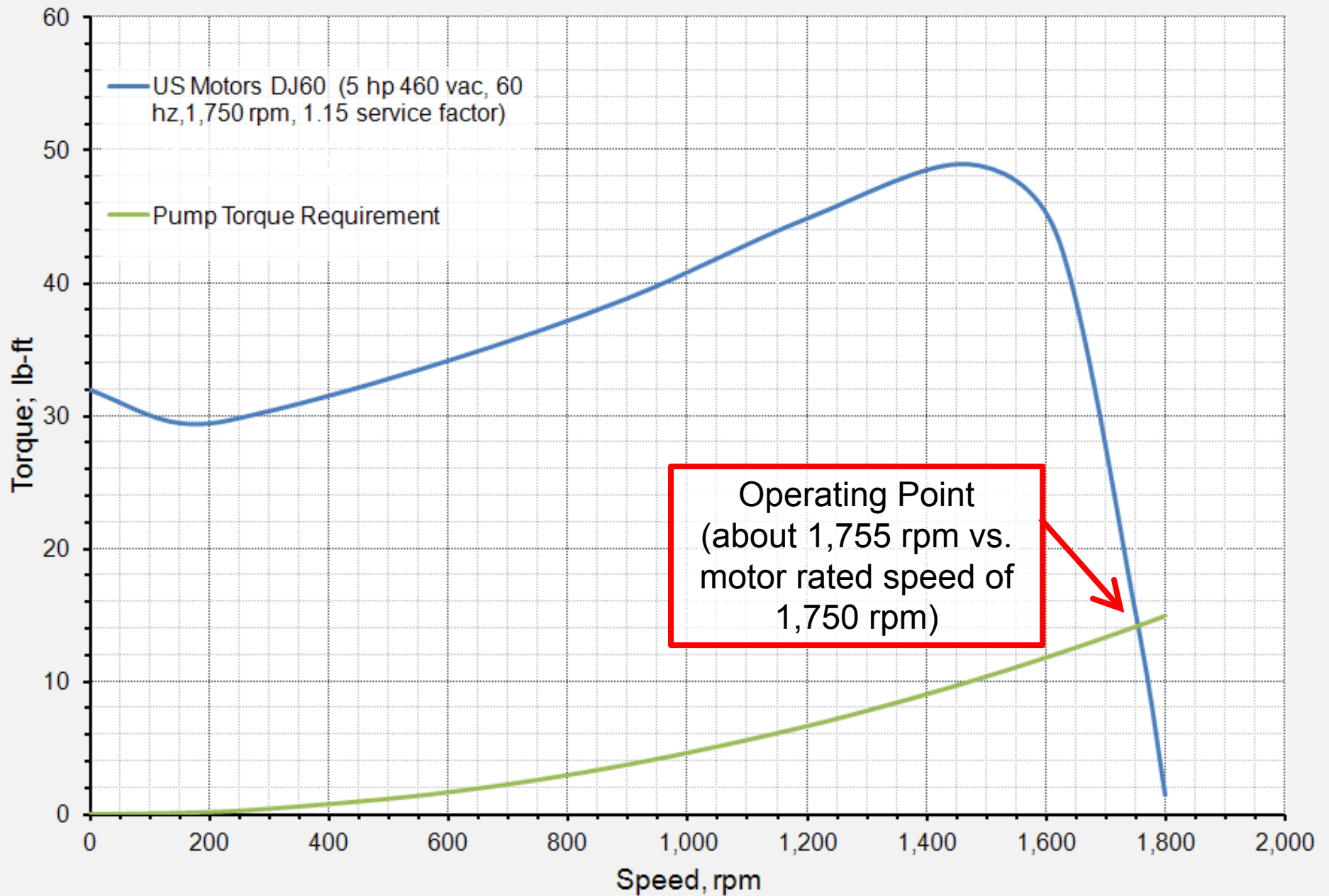


## Speed vs. Torque for the PEC Chilled Glycol Pump Motor

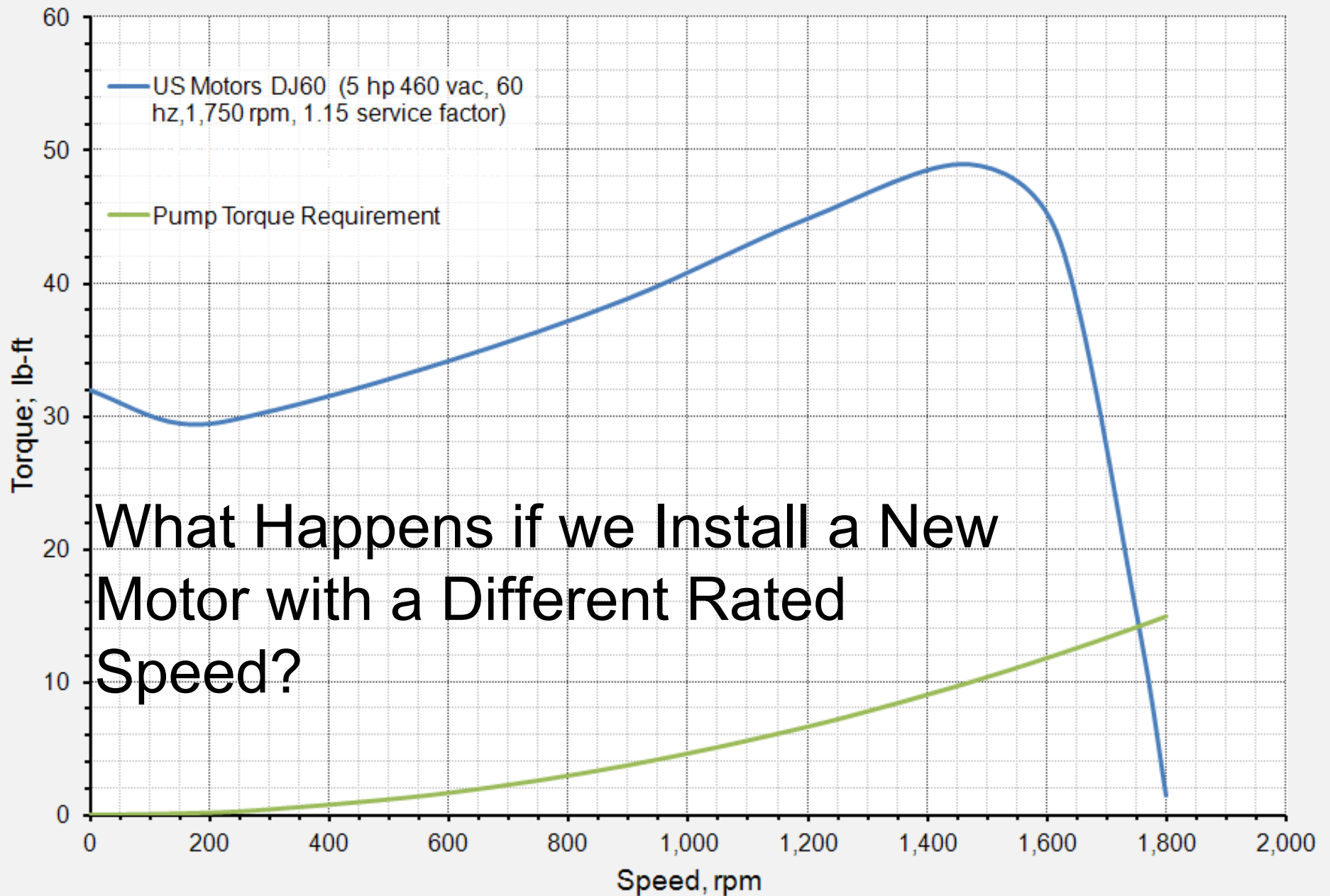




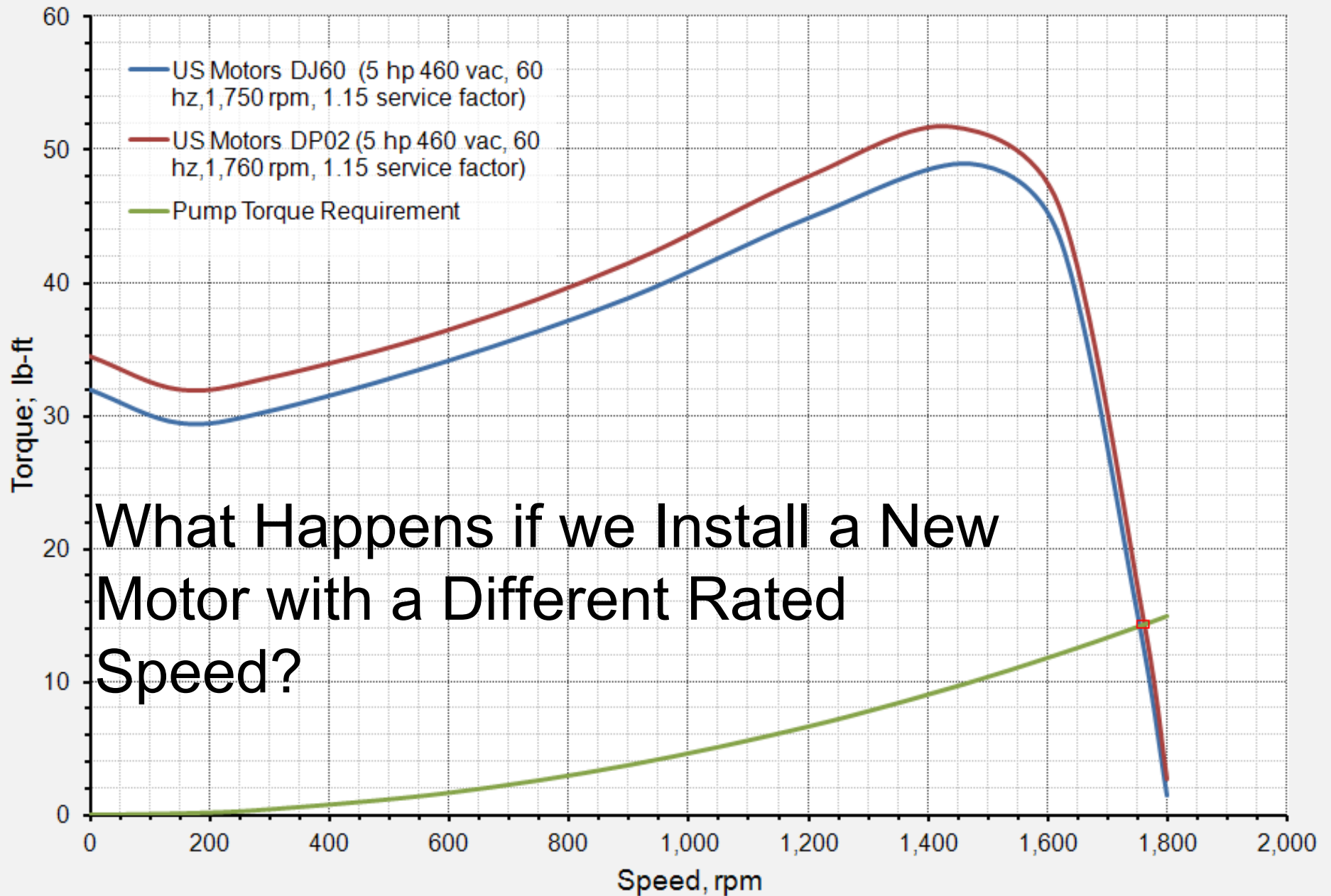
## Speed vs. Torque for the PEC Chilled Glycol Pump and Motor



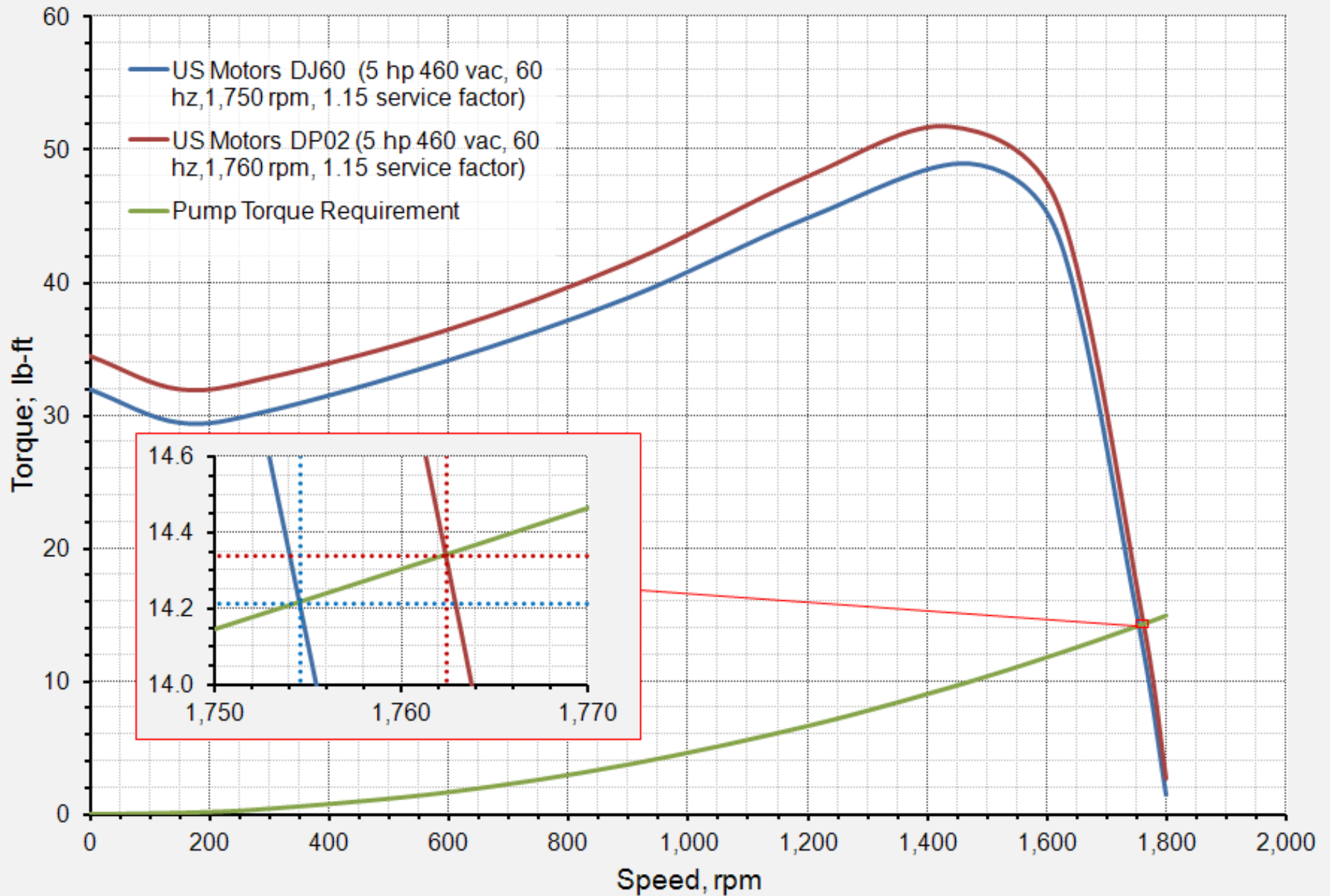
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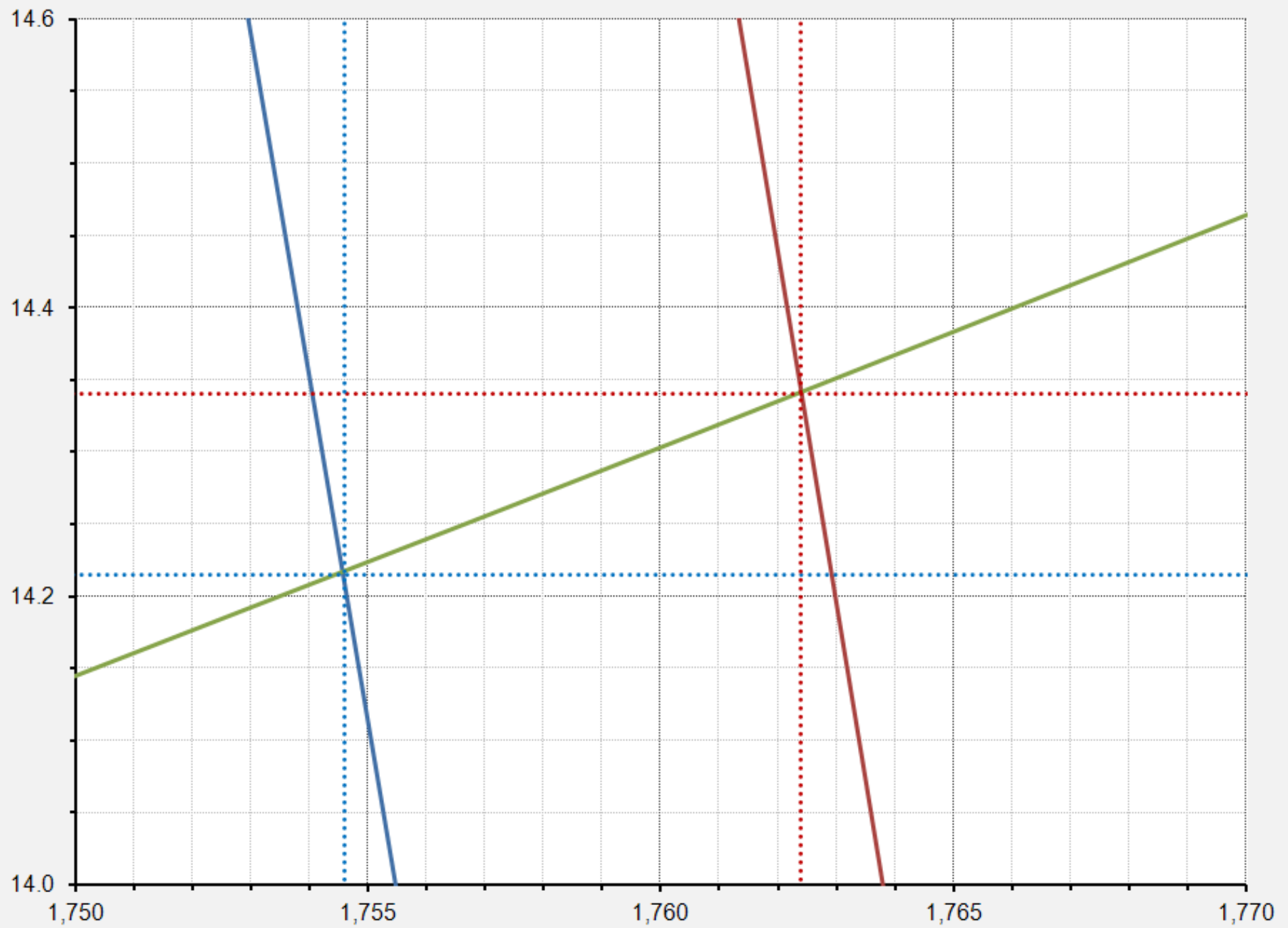


## Speed vs. Torque for the PEC Chilled Glycol Pump and Two Motors



## Speed vs. Torque for the PEC Chilled Glycol Pump and Two Motors





# Energy Cost Associated with the Shift in Operating Point

Pump full speed bhp	4.75	Based on original CHW pump selection
Pump full load motor speed	1,755	rpm (from the DJ60 curve at design hp)
Pump design torque	14.22	lb-ft <sub>Force</sub>
Speed at new motor balance point	1,762	rpm (read from chart)
Torque at new motor balance point	14.34	lb-ft (read from chart)
bhp at new operating point	4.81	hp
Increase from original bhp	0.06	hp
	0.05	kW
Increased kWh	405	kWh per year (round the clock operation)
Increased \$	\$61	per year (\$0.15/kWh electrical cost)



# Energy Cost Associated with a Motor Efficiency Improvement

Original bhp required	4.75	hp
Efficiency improvement	1.5%	
hp savings	0.07	hp
kW savings	0.05	kW
Anticipated Savings kWh	477	kWh per year (round the clock operation)
Anticipated Savings \$	\$72	per year (\$0.15/kWh electrical cost)

# The Bottom Line if you Don't Pay Attention to Rated Speed

What you thought you would save

- 477 kWh per year
- \$72 per year (\$0.15/kWh electrical cost)
- 7.2 year simple payback

What you actually save

- 72 kWh per year
- \$11 per year (\$0.15/kWh electrical cost)
- 46.1 year simple payback