

VAV Systems

Design, Performance and Commissioning Issues

Terminal Unit Maximum Flow - Where Does It Come From?



Instructor:

David Sellers

Senior Engineer

Facility Dynamics Engineering

March 7, 2018

ALTITUDE: SEA LEVEL
BAROMETRIC PRESSURE: 29.921 in. HG
ATMOSPHERIC PRESSURE: 14.696 psia

The Air Temperature at the Diffuser Serves the Load

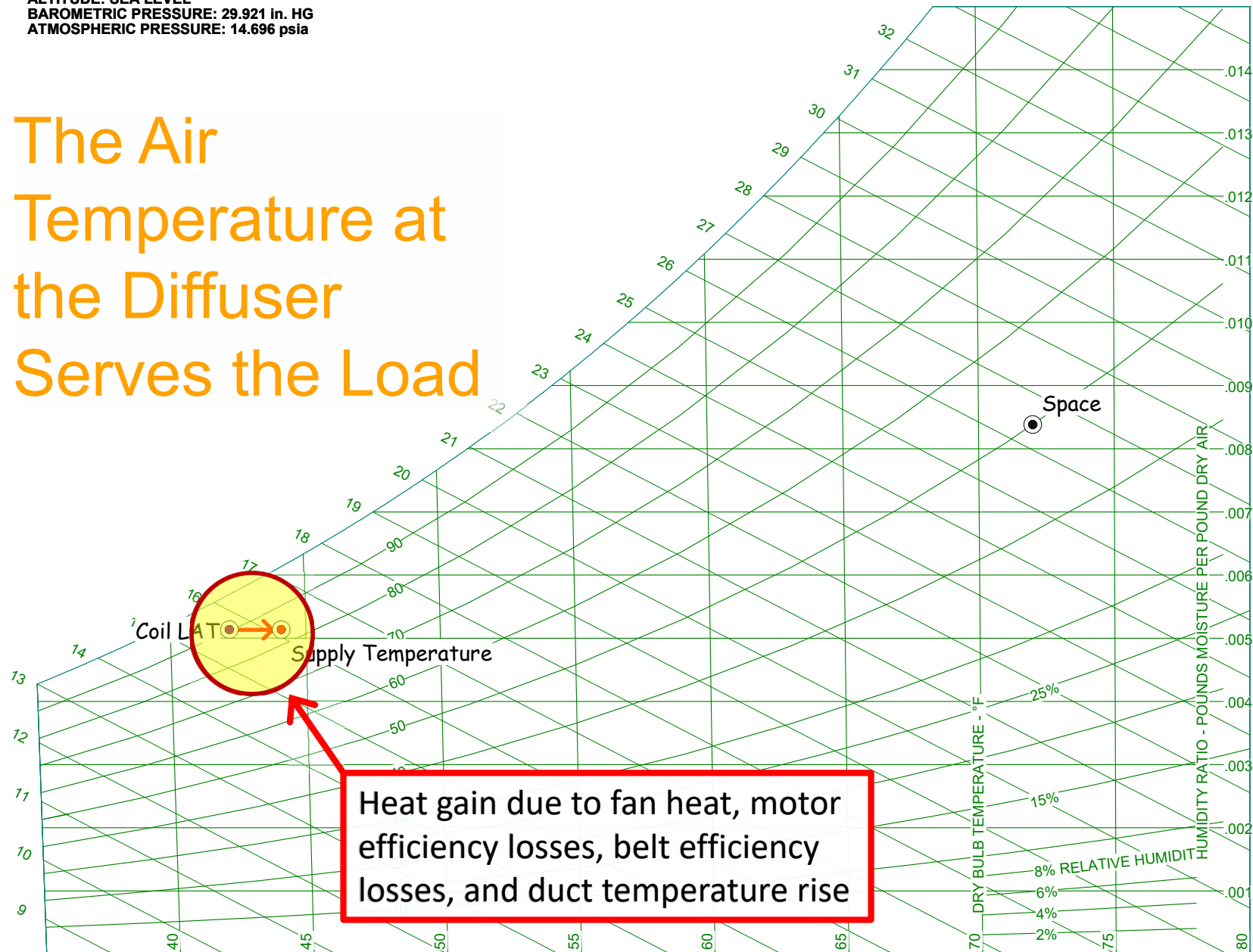


Chart by: AKTON PSYCHROMETRICS, www.aktonassoc.com

C:\Users\DSellers\Documents\FDE Tools\SketchUp\Marriott Ballroom AHU\Design Info Anahiem Marriott Basis\Ballroom Load Components - 1 Design - Supply Temp v1.aad

TERMINAL UNIT MAXIMUM FLOW - WHERE DOES IT COME FROM?

What is in This Module?

- What sets the maximum air flow in a VAV zone
- Why we have to do reheat

Setting the Maximum Flow Rate

$$Q_{\text{Btu per hour}} = 1.08 \times \text{Flow}_{\text{Cubic Feet per Minute}} \times (\text{Temperature}_{\text{In, } ^\circ\text{F}} - \text{Temperature}_{\text{Out, } ^\circ\text{F}})$$

Where:

$Q_{\text{Btu per hour}}$ = Sensible load from the load calculations

1.08 = Unit conversion constant for dry air at 70°F

$\text{Flow}_{\text{Cubic Feet per Minute}}$ = The flow rate required to offset the load

$(\text{Temperature}_{\text{In, } ^\circ\text{F}} - \text{Temperature}_{\text{Out, } ^\circ\text{F}})$ = Supply air to room temperature difference, °F

Setting the Maximum Flow Rate

$$\frac{Q_{\text{Btu per hour}}}{1.08 \times (\text{Temperature}_{\text{In}, ^\circ\text{F}} - \text{Temperature}_{\text{Out}, ^\circ\text{F}})} = \text{Flow}_{\text{Cubic Feet per Minute}}$$

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Setting the Maximum Flow Rate

$$\frac{736,670 \text{ Btu/hr}}{(1.08 \times (72^\circ\text{F} - 44.1^\circ\text{F}))} = \text{Flow}_{\text{Cubic Feet per Minute}} = 24,487 \text{ cfm}$$

Where:

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The Air Volume Thus Calculated:

- Will be exactly cold enough,
 - And exactly dry enough,
- to offset the zone latent and sensible gains*

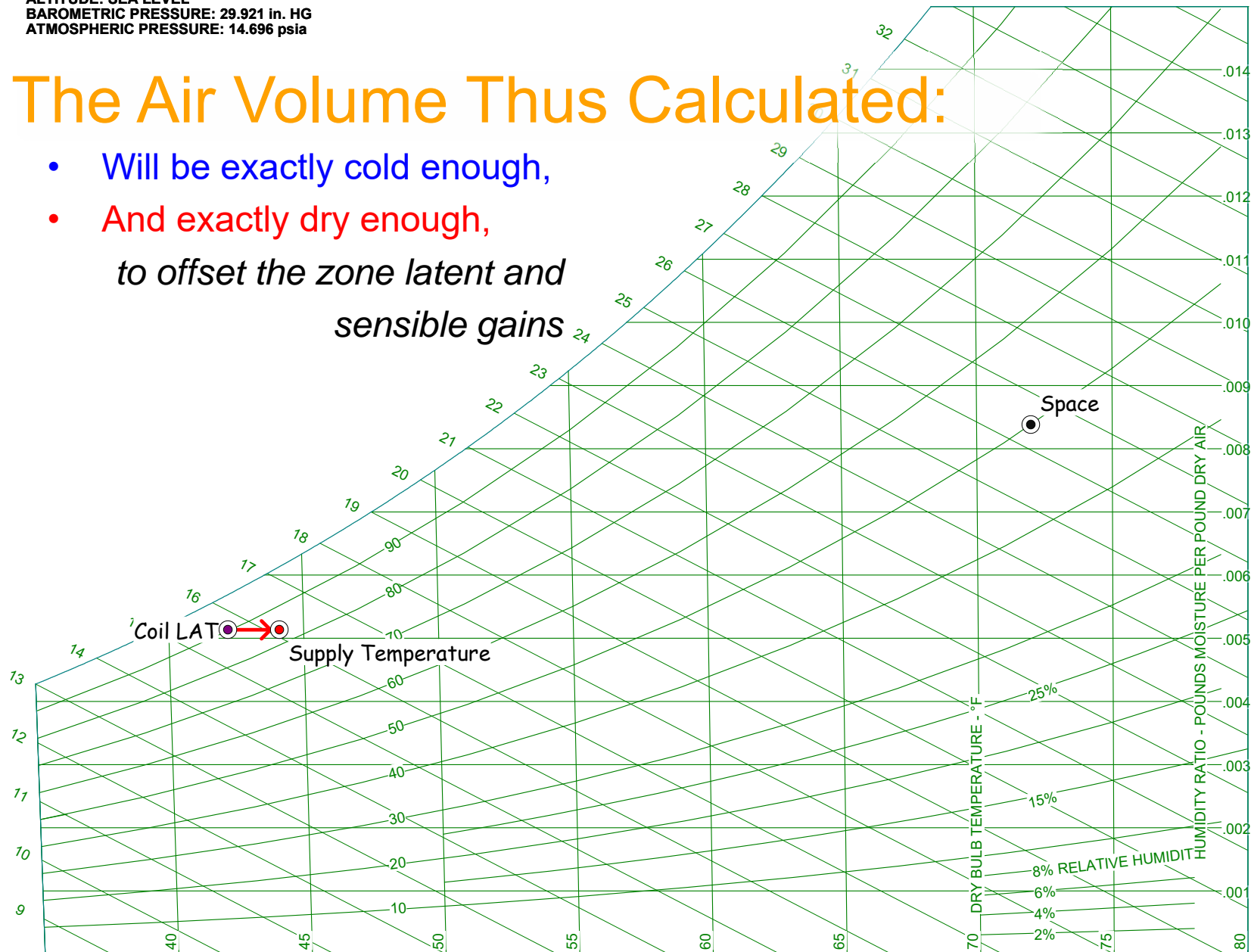


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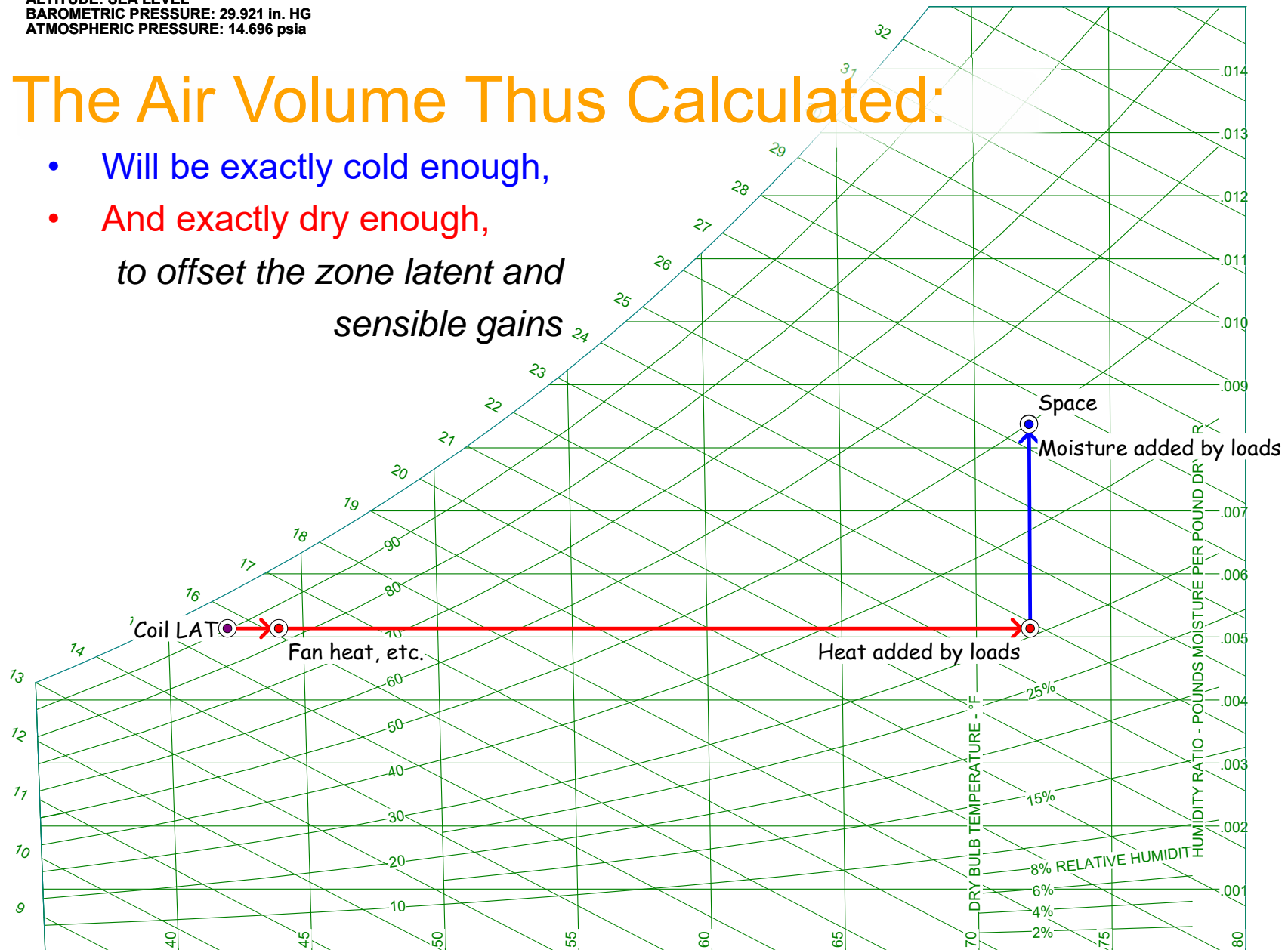


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TERMINAL UNIT MAXIMUM FLOW - WHERE DOES IT COME FROM?

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The Loads add Heat and Moisture to the Supply Air

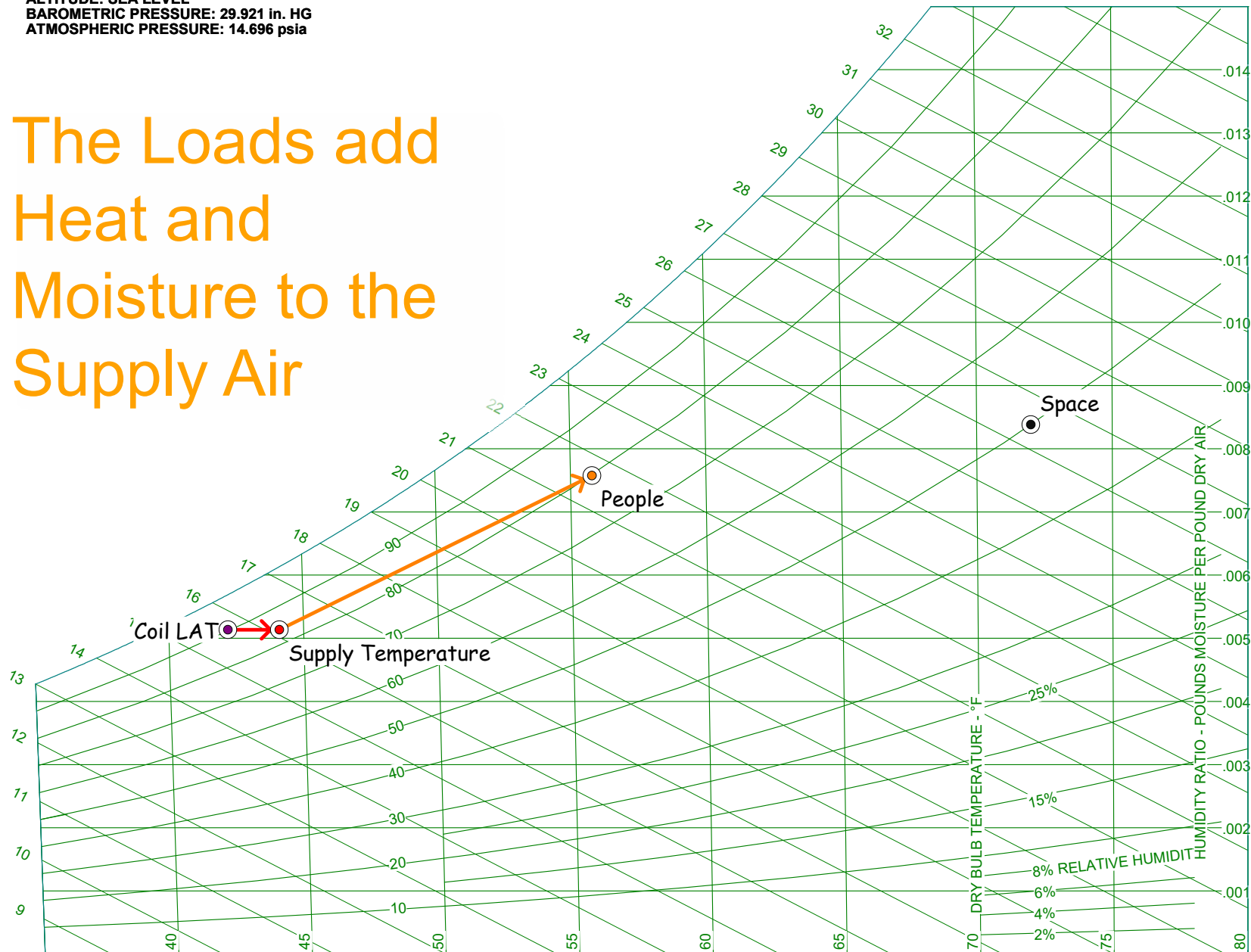


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TERMINAL UNIT MAXIMUM FLOW - WHERE DOES IT COME FROM?

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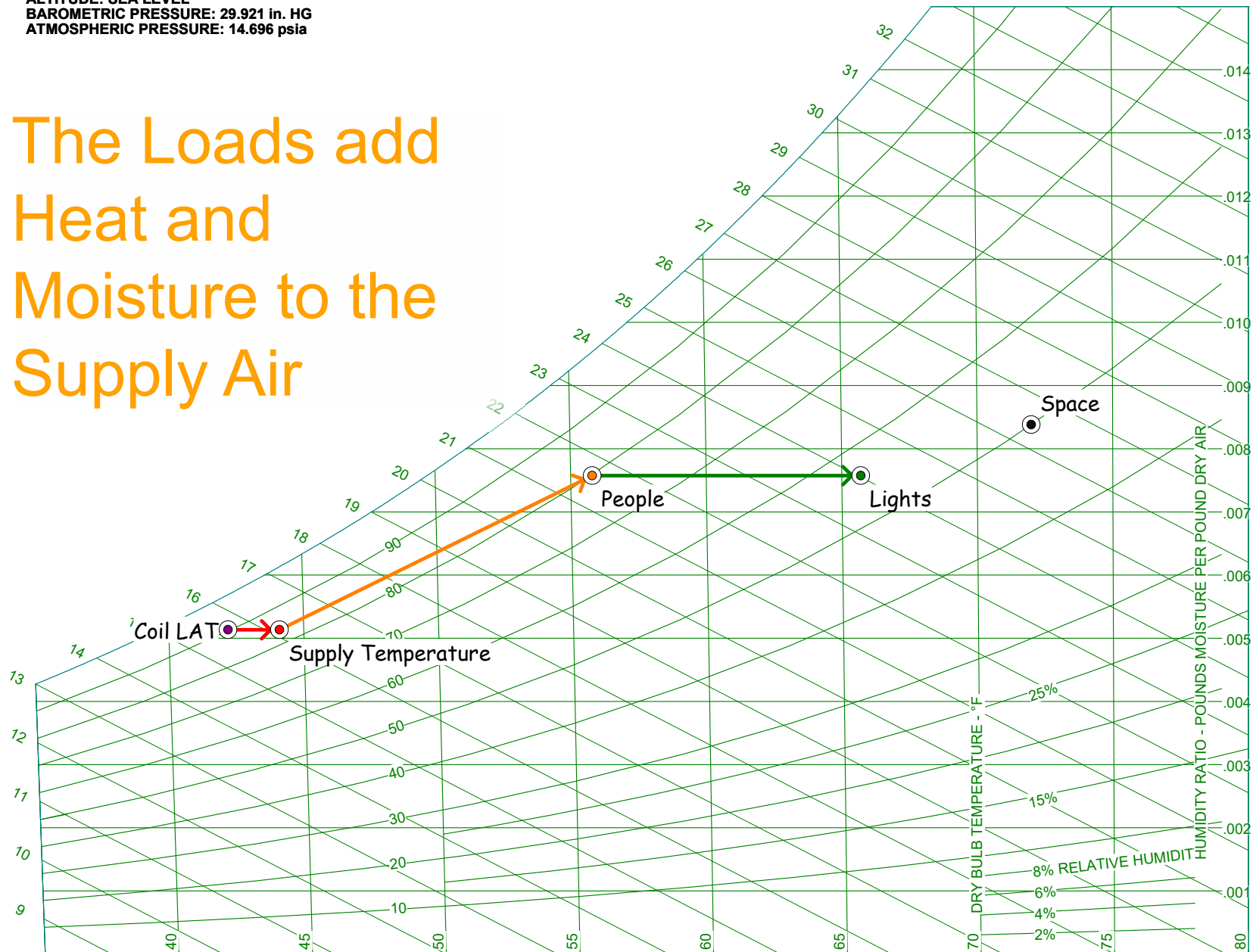


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TERMINAL UNIT MAXIMUM FLOW - WHERE DOES IT COME FROM?

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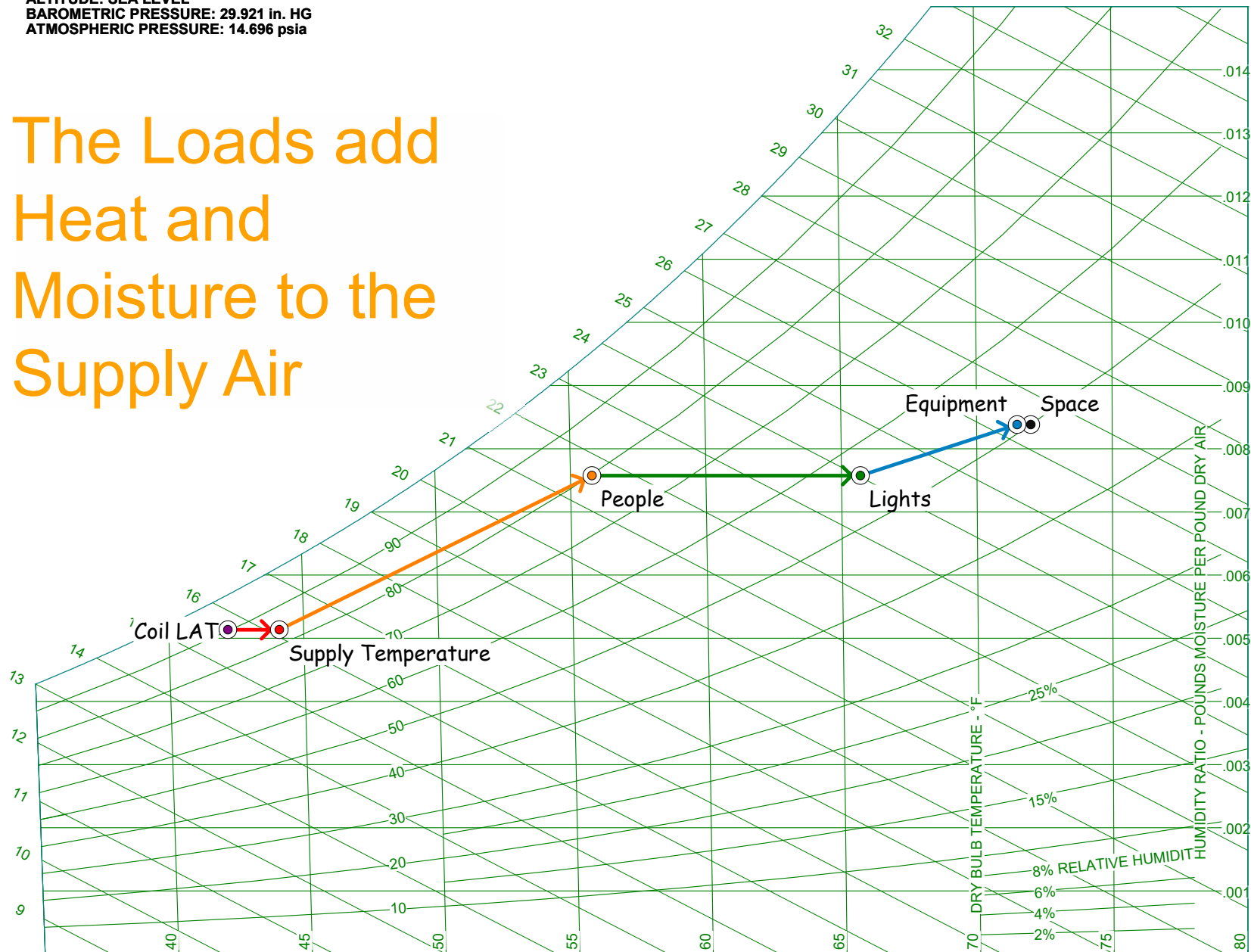


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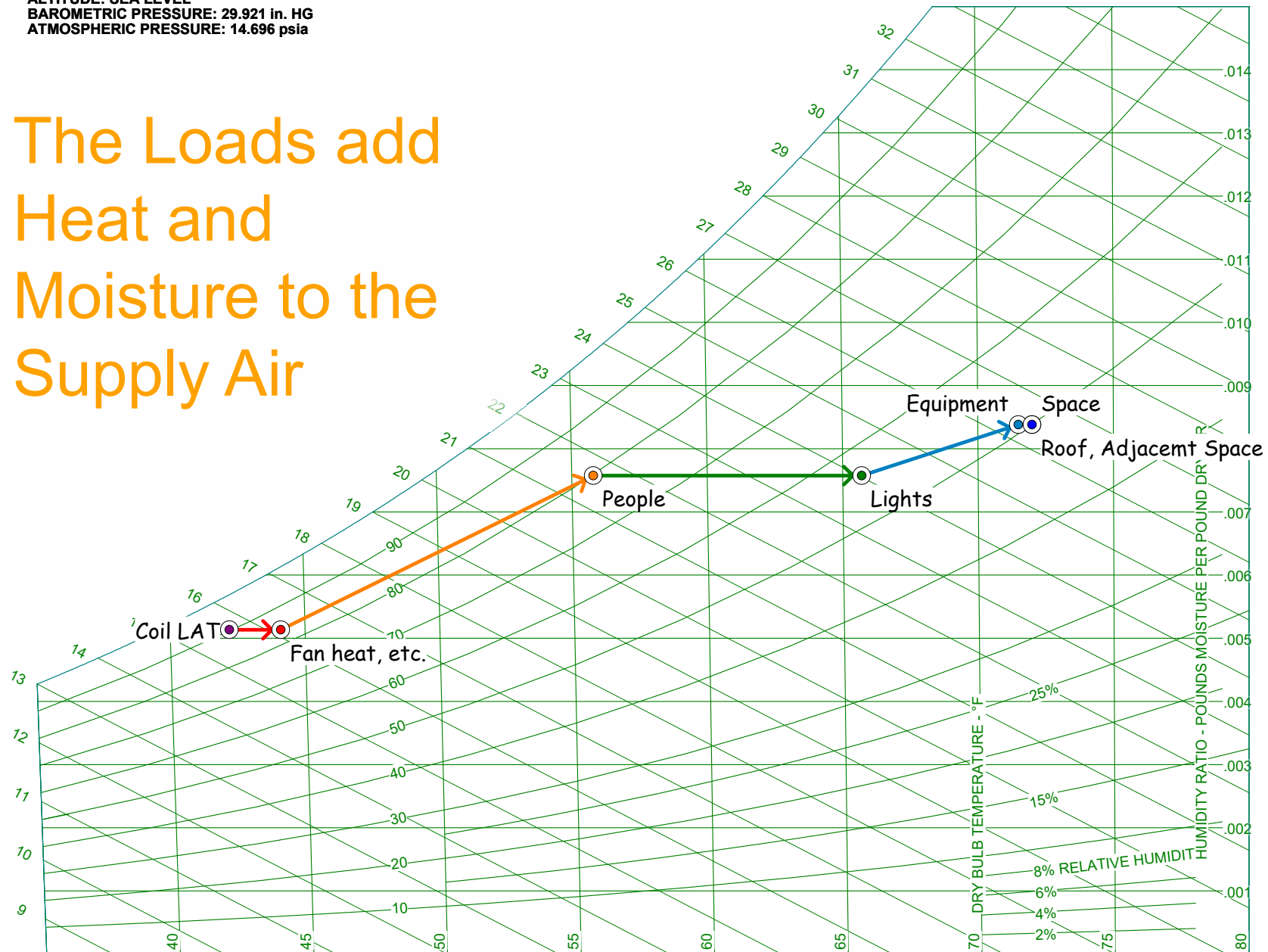


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TERMINAL UNIT MAXIMUM FLOW - WHERE DOES IT COME FROM?

Coil Leaving Conditions Below the Mid to Upper 40°F Range Can Be a Challenge

- Deep coils are required
 - Higher pressure drops
 - Harder to clean
 - Heavy condensation loads = potential carry-over of moisture

Coil Leaving Conditions Below the Mid to Upper 40°F Range Can Be a Challenge

- Colder refrigerant temperatures required
 - For direct expansion, refrigerant temperatures below 32° = frost on the coil
 - For chilled water, another heat transfer step is added
 - Air transfers heat to water
 - Water transfers heat to refrigerant
 - Refrigerant may be at or below freezing

Remember, there has to be a temperature difference for heat transfer to happen.

Approaches closer than 5°F between the heat sink and source typically become prohibitively expensive

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Higher Air Flow + Higher Coil Leaving Air Temperature May Work

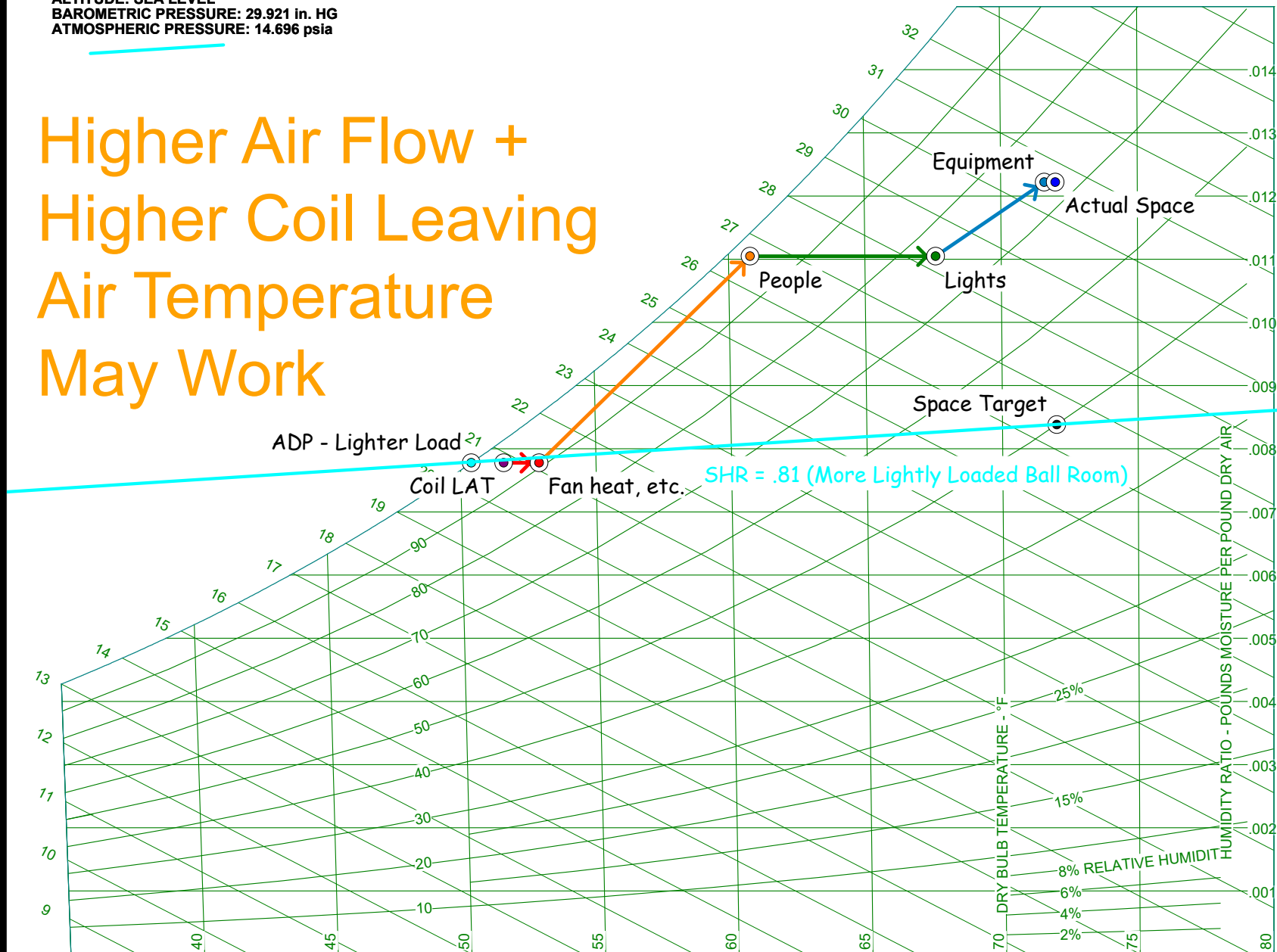


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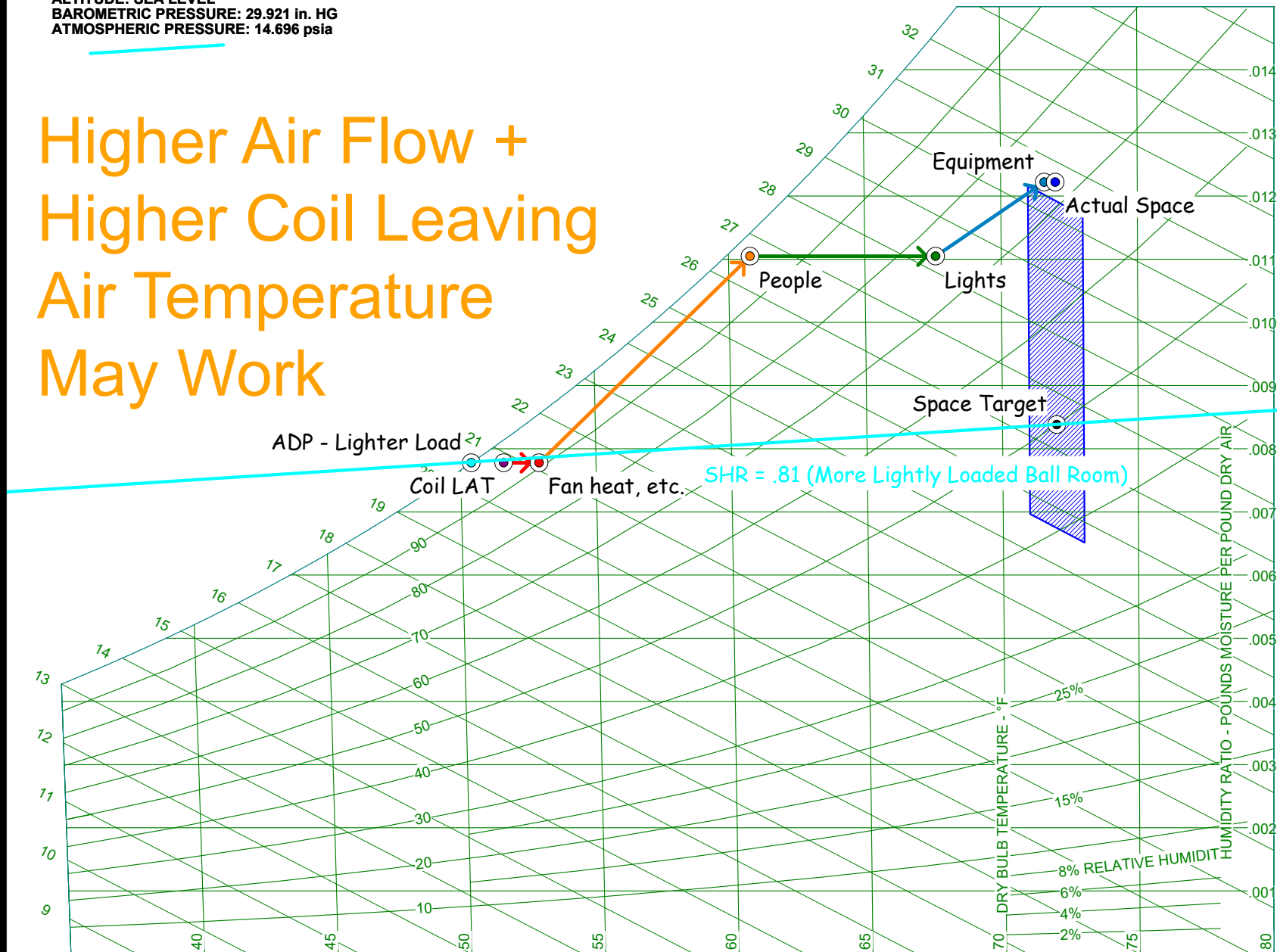


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Reductions in Sensible Load + No Reduction in Latent Load = Problem

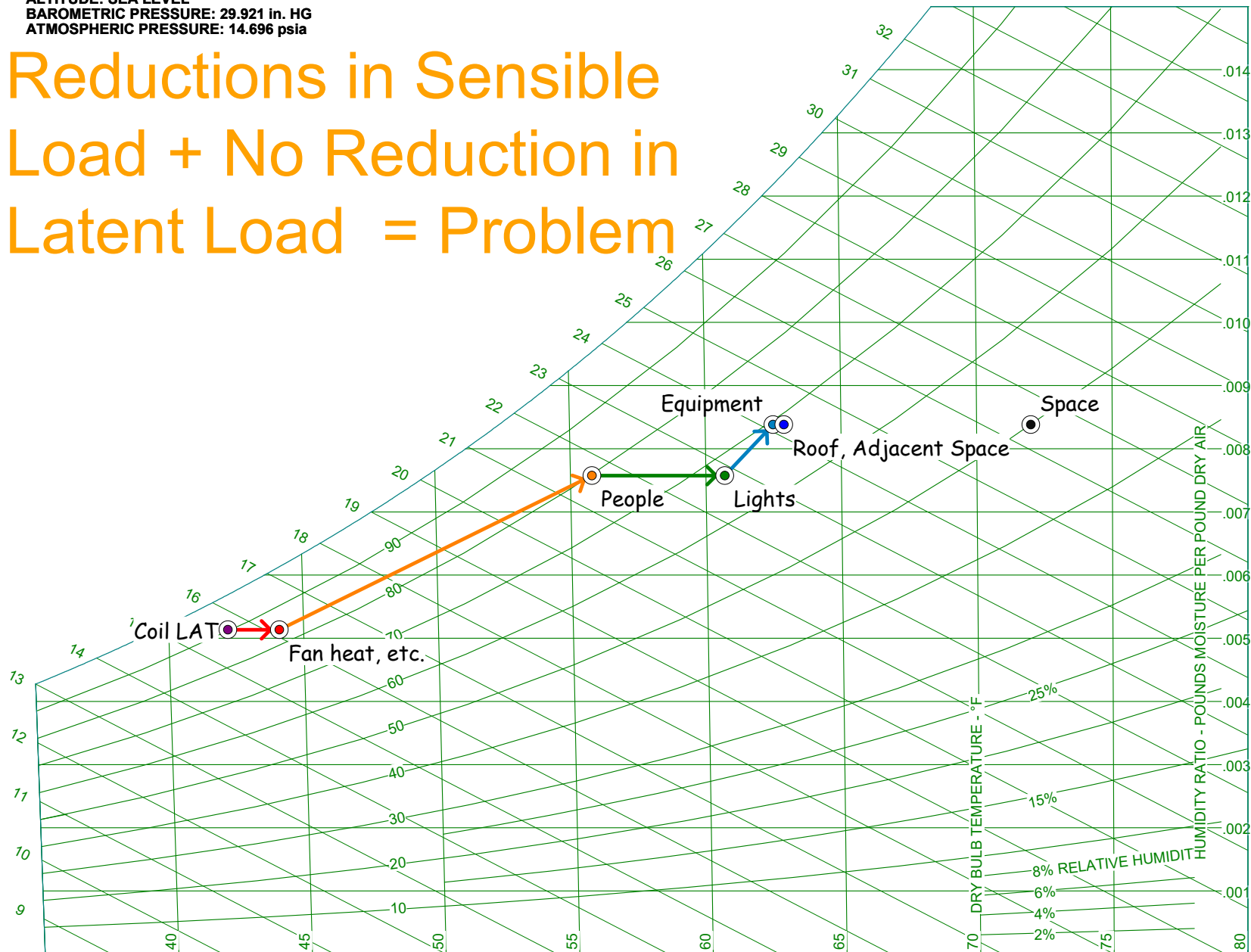


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Problem Solved!

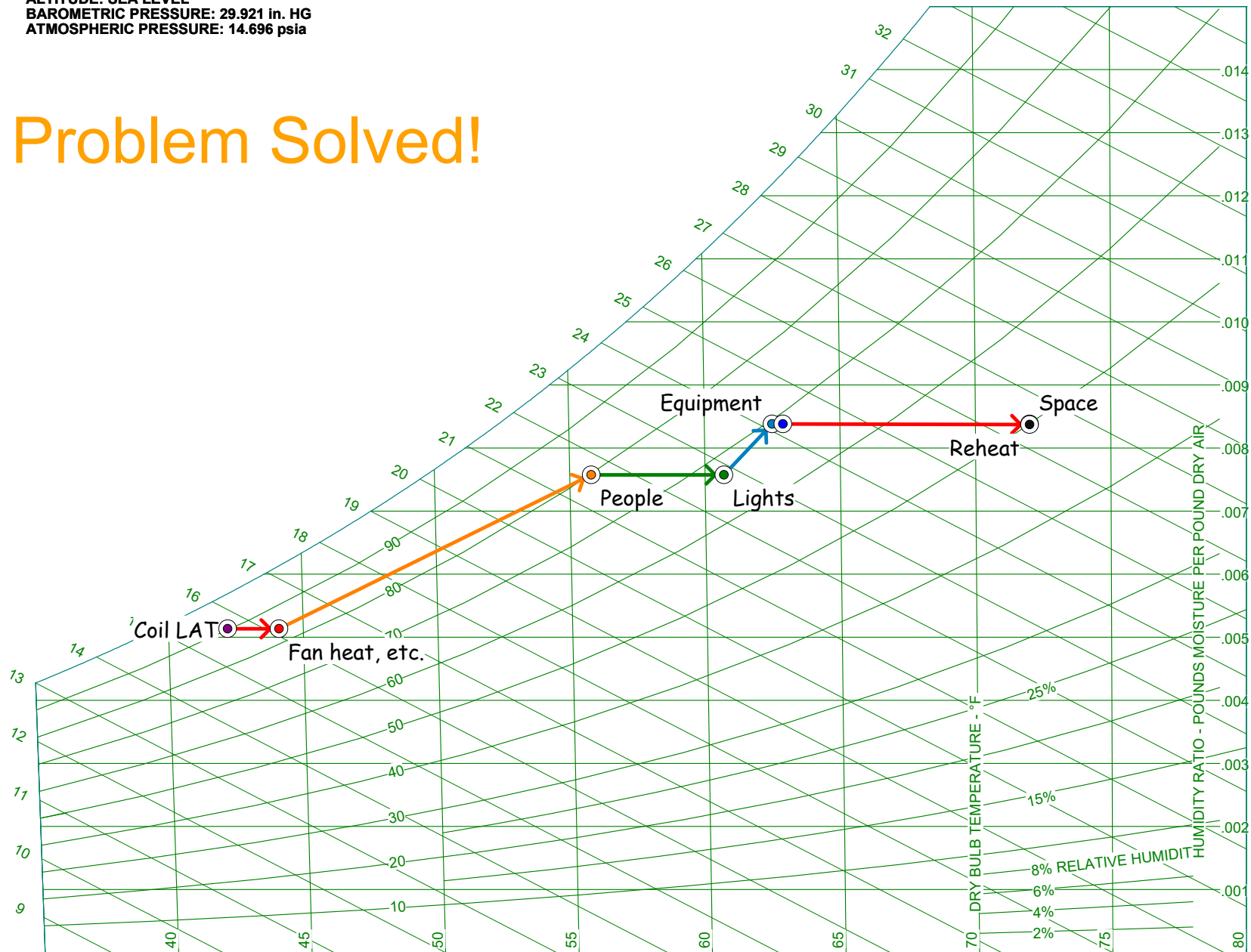


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Tailoring the Air Flow to the Sensible Load; a Sensible Solution

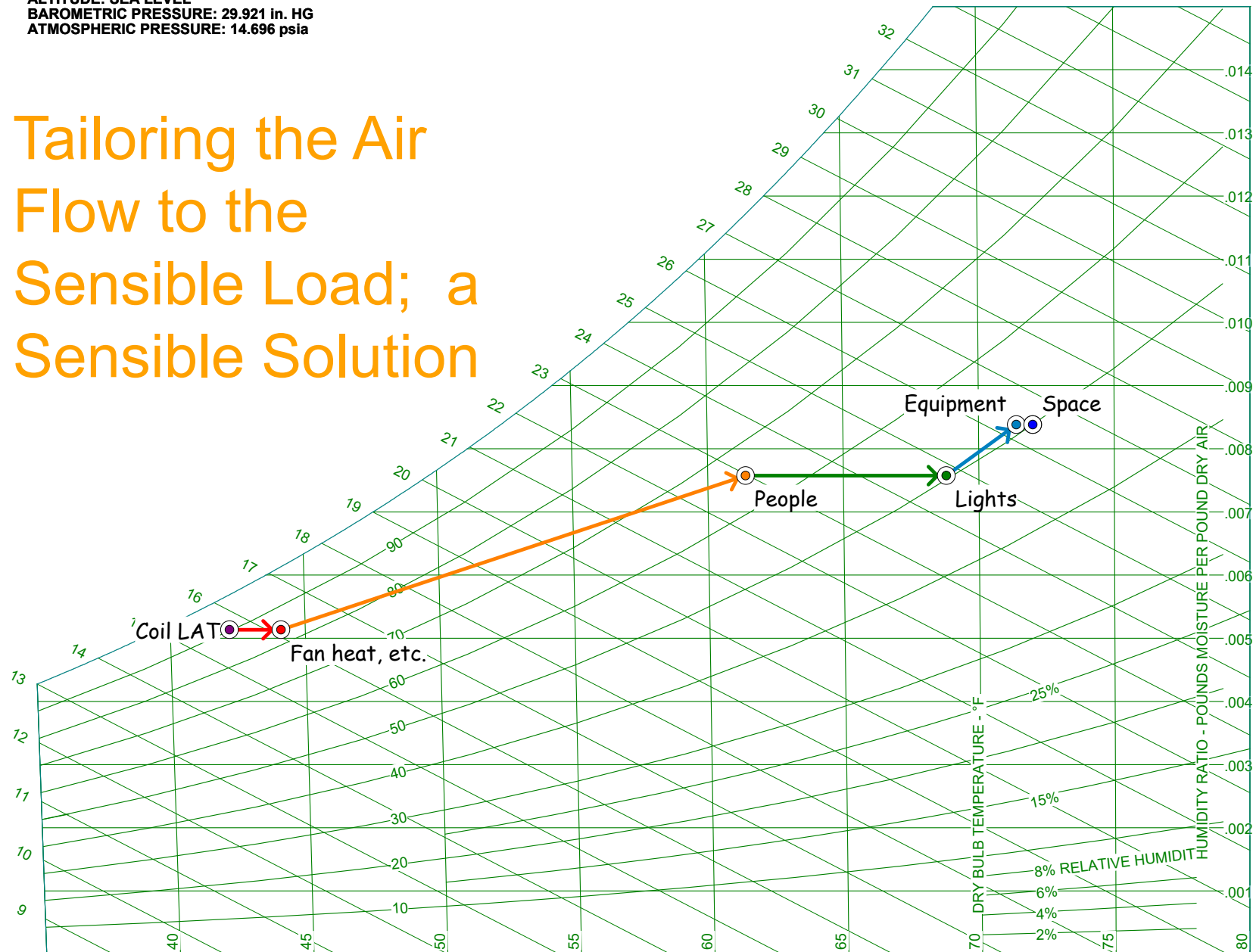


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A Lower Mass Flow Rate Means a Given Load will Cause a Large Change in the State of the Air

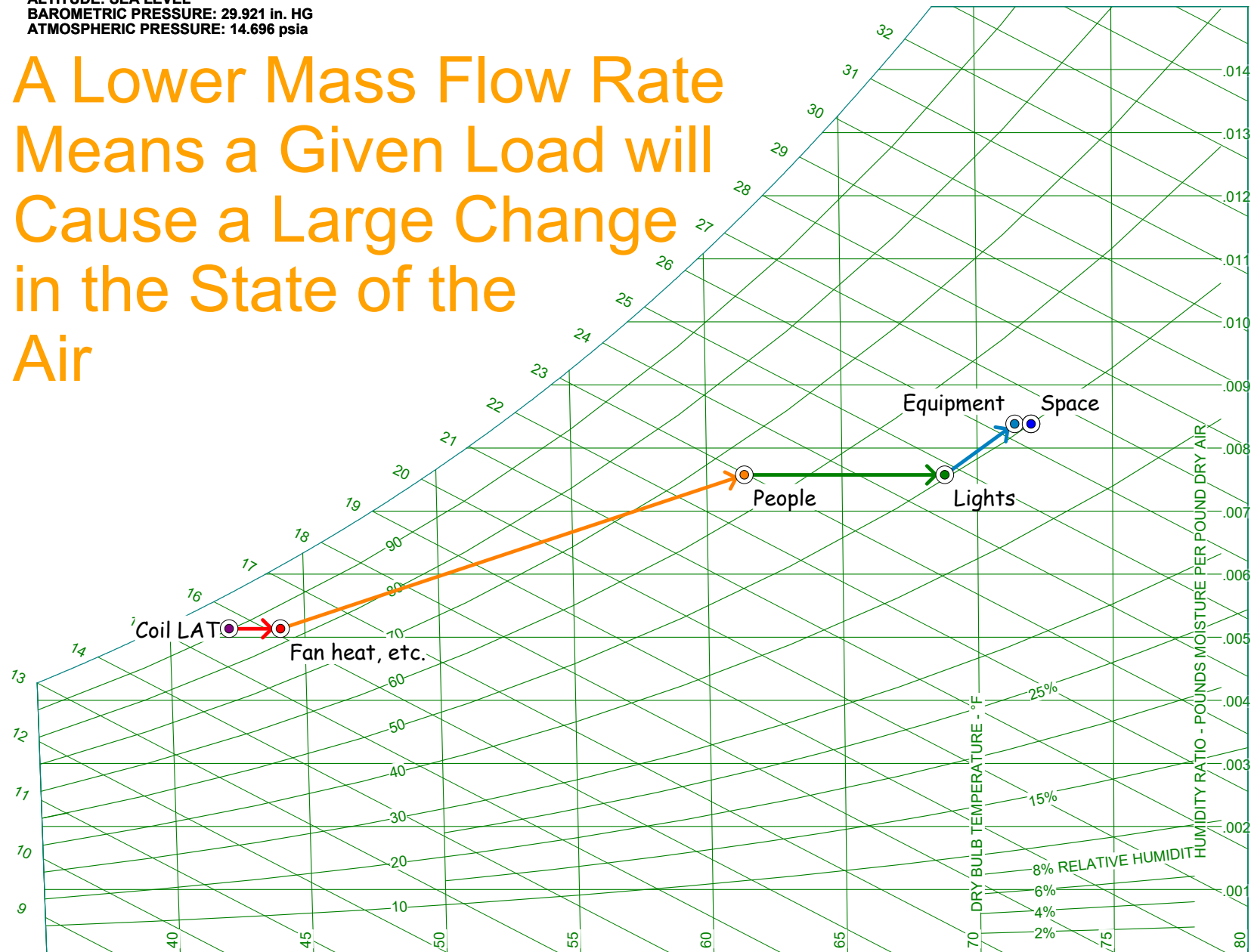


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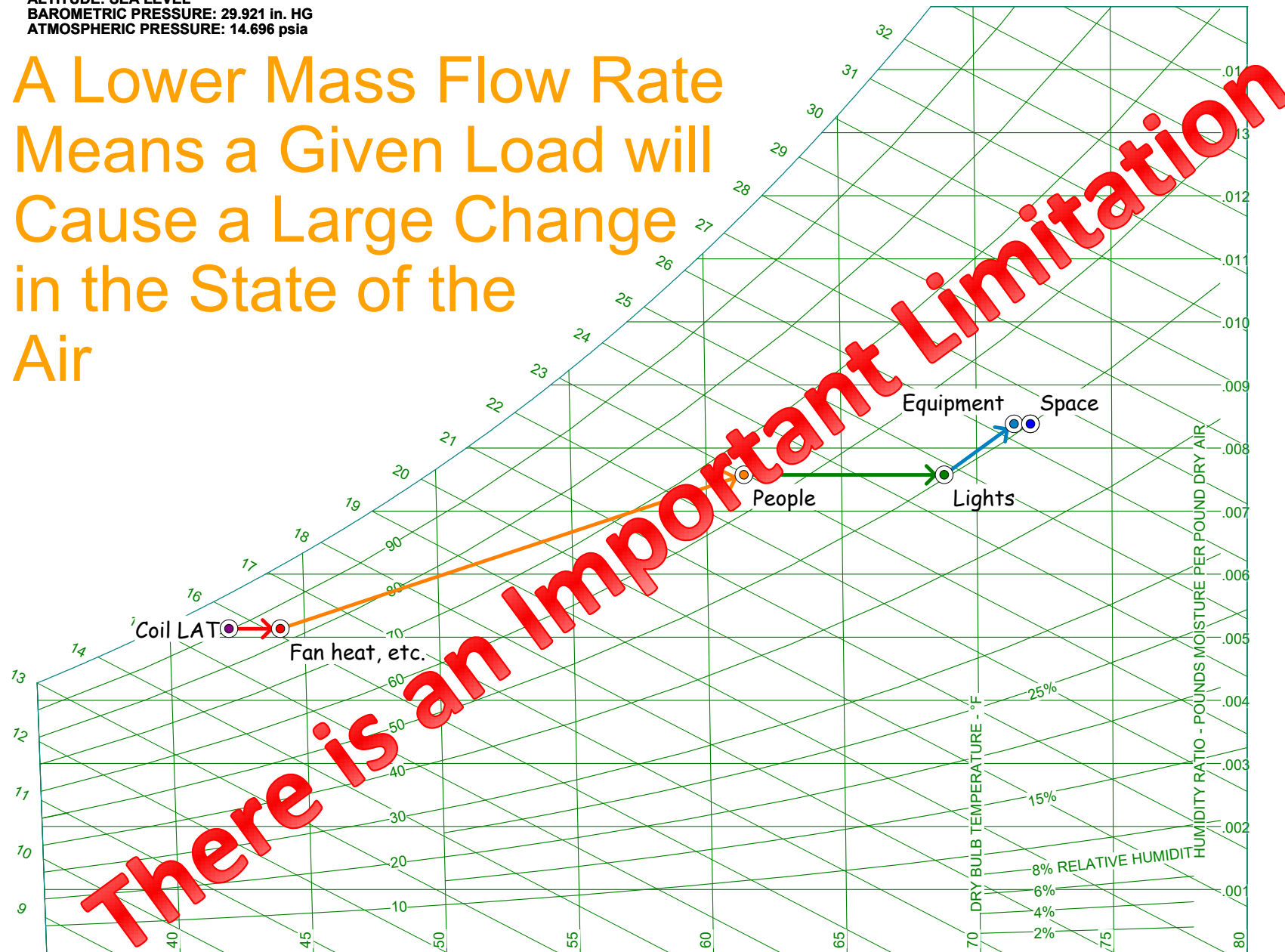


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