

# VAV Systems

Design, Performance and Commissioning Issues

What is a VAV System and Why Do We Use Them



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March 7, 2018

# Class Material Location

The slides and other supporting information for the class can be found at:

- <http://www.av8rdas.com/pacific-energy-center-classes1.html>
- They will be there until the next class, at which time they will be relocated down the page under the class title.

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- You still need to understand how it works and fix it if it doesn't work for you

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# Learning Objectives

**After completing this course you should be able to:**

1. Understand why we use VAV systems
2. Be familiar with the components in a typical VAV system
3. Understand how interactive VAV systems are with the loads and other systems and processes in the facility
4. Understand terminal units
5. Understand basic VAV control strategies

# Agenda

1. Introduction – Why VAV and what is it exactly?
2. Terminal unit basics
3. Flow and pressure measurement
4. Minimum and maximum flow settings
5. Fan flow and static control
6. Supply and return fan flow management
7. Interactive exercises

# What's In This Module?

- Why we do VAV
- Mathematical fundamentals of the VAV process
- Ripple effects of the VAV process
- What a typical VAV AHU looks like





# Why a Focus on VAV Systems?

*Because they are everywhere!*



# Why a Focus on VAV Systems?

## PIER VAV Research Results

- *For large commercial buildings best practices could save \$0.12 per square foot in fan energy*
- *For small commercial buildings (30,000 – 50,000 sq.ft.) best practices could save 10-15% in energy cost over current approaches*

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# The Fundamental Process at the Zone

## Sensible Heat Equation

$$Q = 1.08 \times cfm \times \Delta t$$

Where :

$Q$  = Load in btu/hr

1.08 = A units conversion constant

$cfm$  = Flow rate in cubic feet per minute

$\Delta t$  = Temperature difference in °F

Variable we are trying to deal with

- Load

*Most often gauged by the temperature of the zone relative to a set point (what we want)*

Variables we can (hope to) control

- Temperature

*Constraints;*

- Space conditions
- Coil discharge conditions

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*Constraints;*

- Peak load
- Ventilation requirement

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Variables we can (hope to) control

- Variable volume systems vary flow as load varies

# Varying the Flow as Load Varies is Powerful

# Why This Matters if You are Working with VAV Systems

Fan power is a function of flow and static pressure

$$bhp = \left( \frac{cfm \times static}{6,356 \times \eta_{fan_{static}}} \right)$$

Where:

*bhp* = Brake horse power into the fan drive shaft

*cfm* = Flow rate in cubic feet per minute

*static* = Fan static pressure

6,356 = A units conversion constant

$\eta_{fan_{static}}$  = Fan static efficiency; .40 = .60 for small fans,  
.68 - .78 for large fans

Divide by motor efficiency and multiply by .746 kW  
per horse power to get killoWatts



# Why This Matters if You are Working with VAV Systems

Static pressure is a function of flow

$$Pressure_{New} = Pressure_{Old} \times \left( \frac{Flow_{New}}{Flow_{Old}} \right)^2$$

Where:

$Pressure_{New}$  = The pressure you want to know in consistent units

$Pressure_{Old}$  = The pressure you know in consistent units

$Flow_{New}$  = The pressure you want to know in consistent units

$Flow_{Old}$  = The pressure you want to know in consistent units

# Why This Matters if You are Working with VAV Systems

Thus:

$$Bhp_{New} = Bhp_{Old} \times \left( \frac{Flow_{New}}{Flow_{Old}} \right)^3$$

Where:

$Bhp_{New}$  = The brake horse power you want to know  
in consistent units

$Bhp_{Old}$  = The brake horse power you know in consistent units

$Flow_{New}$  = A flow rate you have selected and want to predict  
the brake horse power requirement for

$Flow_{Old}$  = The flow rate associated with the known brake horse  
power

# Why This Matters if You are Working with VAV Systems

A Caveat: This applies to a fixed system and VAV systems are not fixed systems

