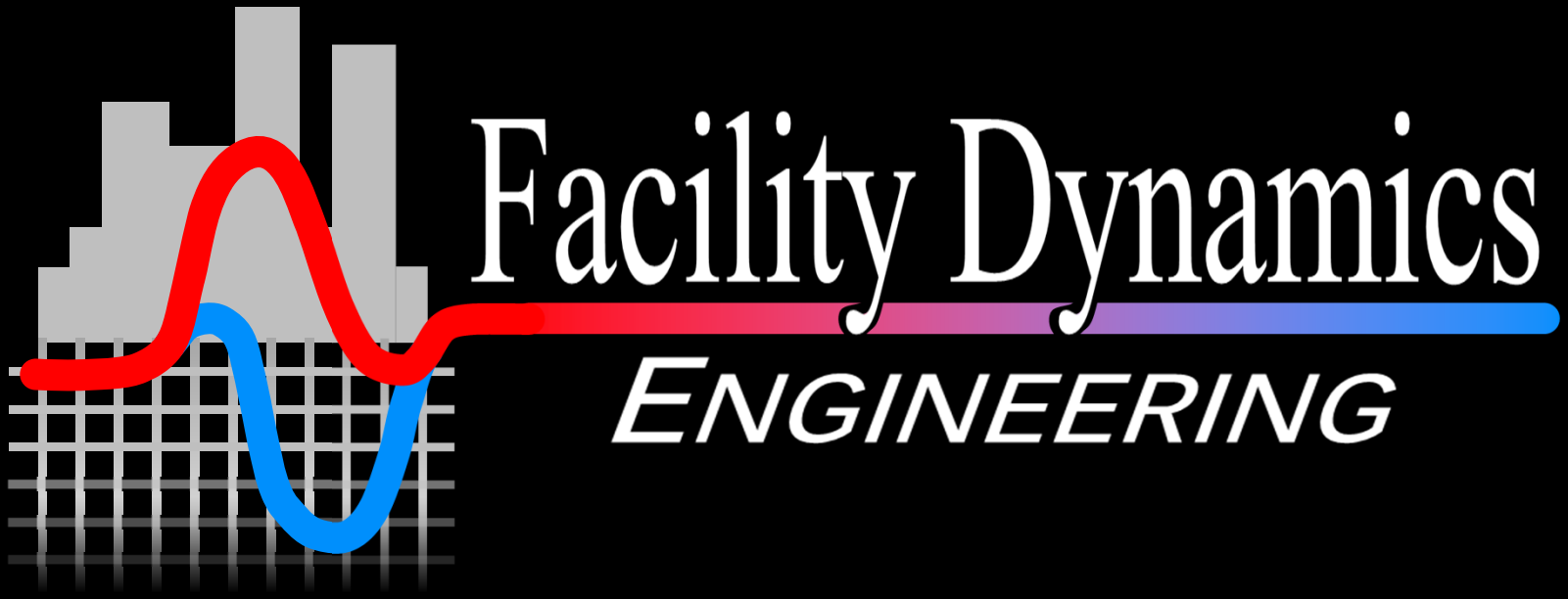


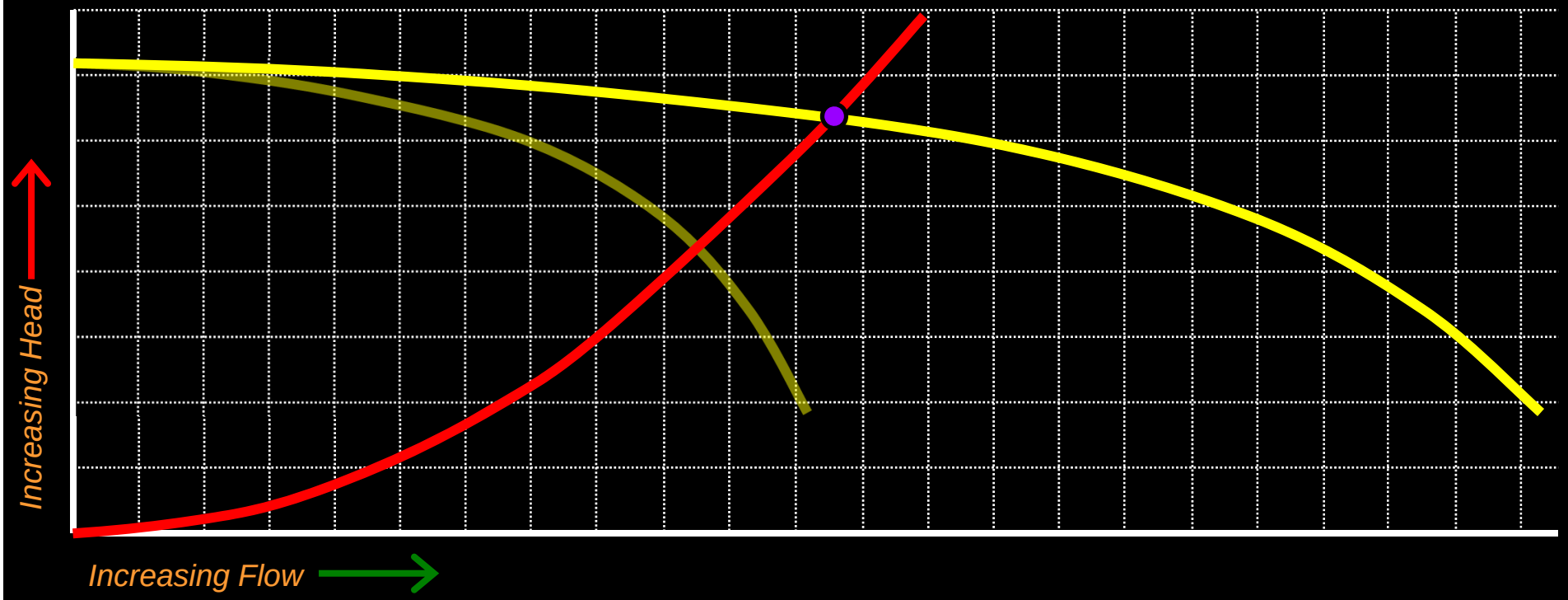
Affinity Laws

Presented By:

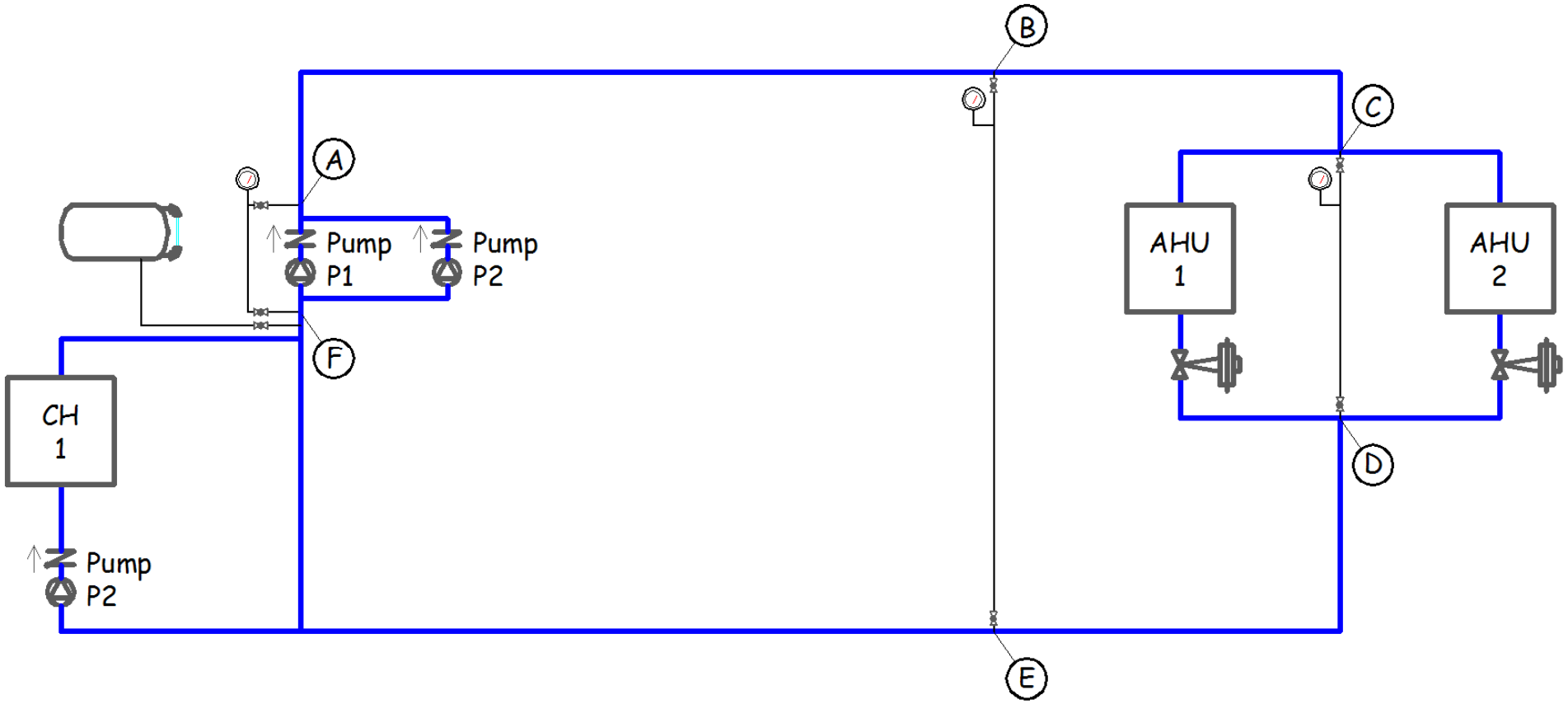
David Sellers; Facility Dynamics Engineering

Senior Engineer

Parallel Pumps

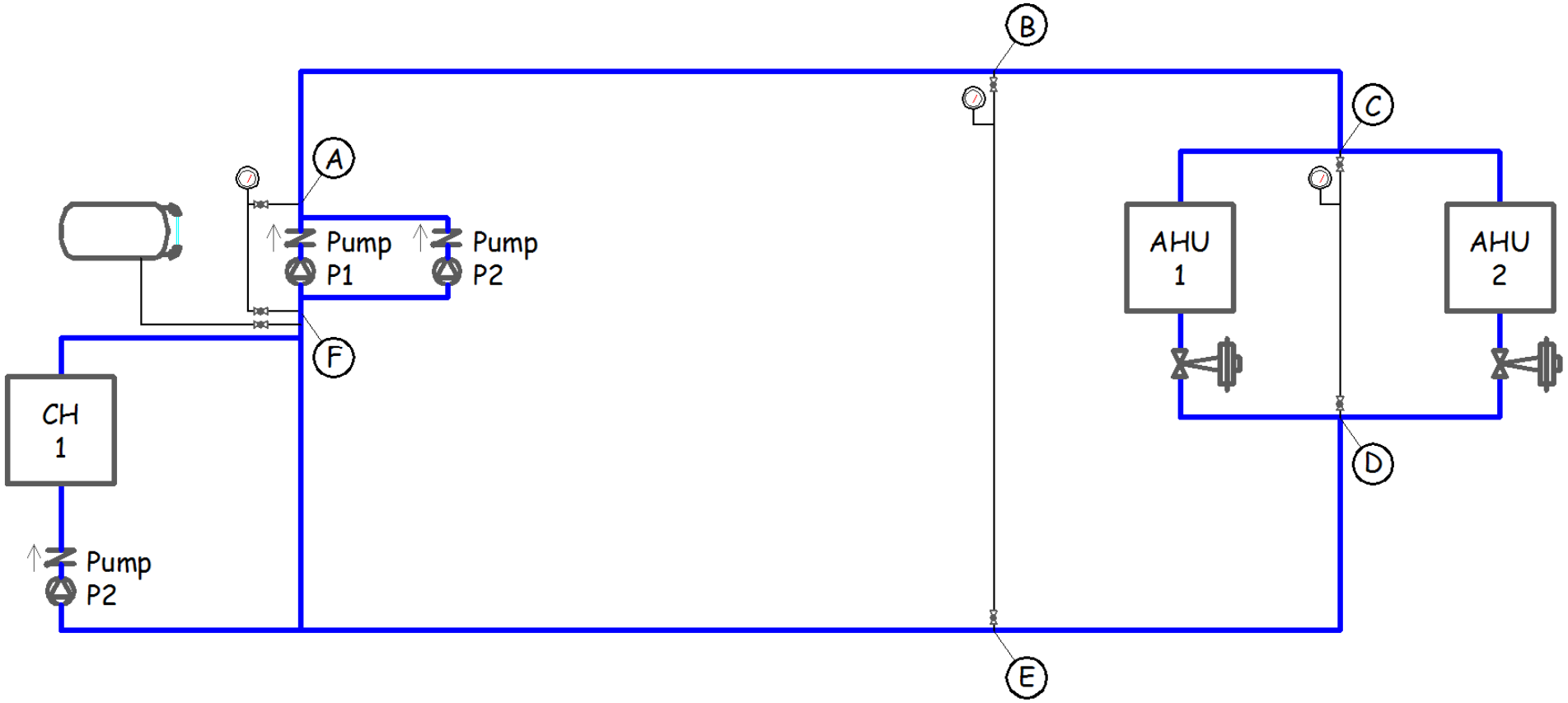


If both pumps are operating at the same speed and one fails, the operating point shifts down the system curve to the point where the single pump curve crosses it; typically more than 50% of the original flow but at less head



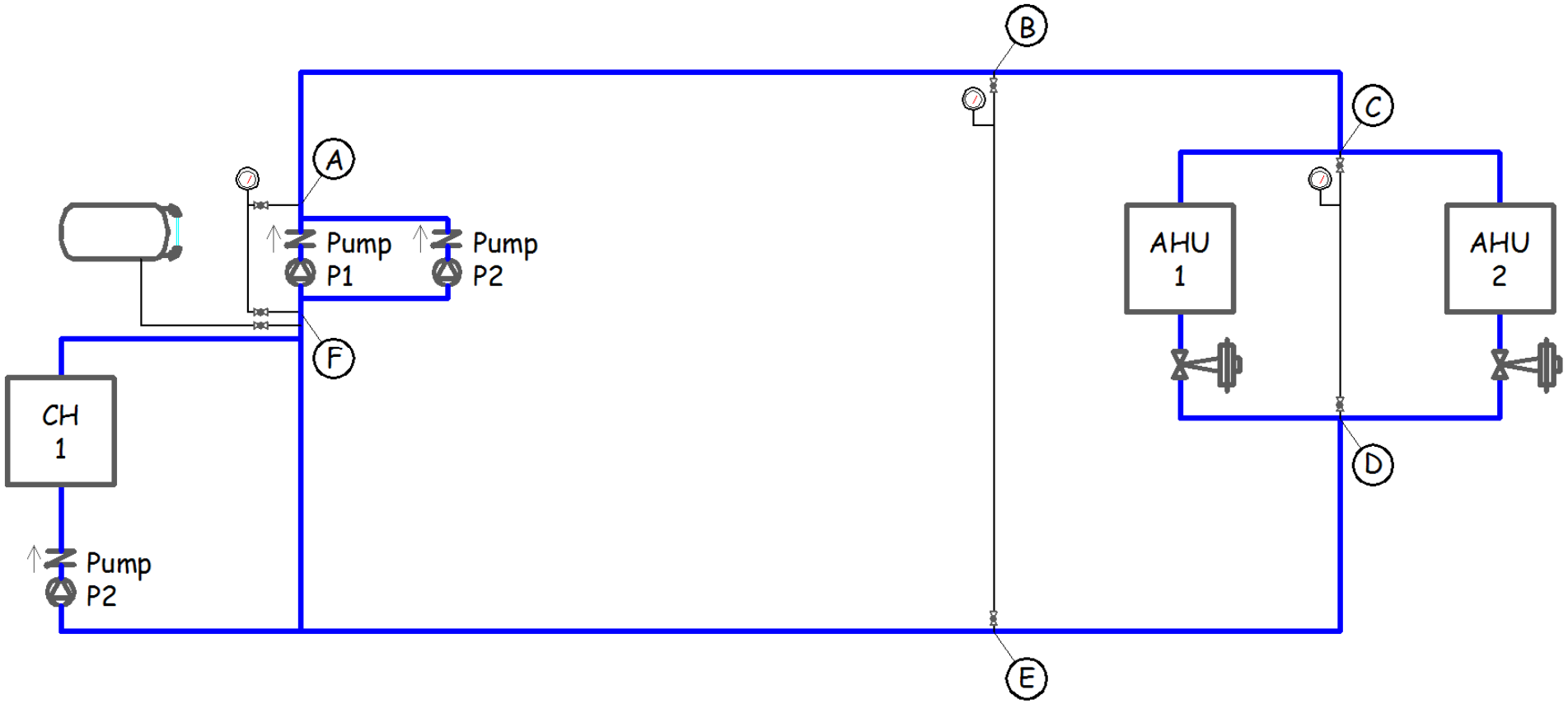
- Assume the pumps are 100% redundant

  What should the logic be, specifically, to operate the pumps on a lead/lag basis?



- Assume the pumps are 100% redundant

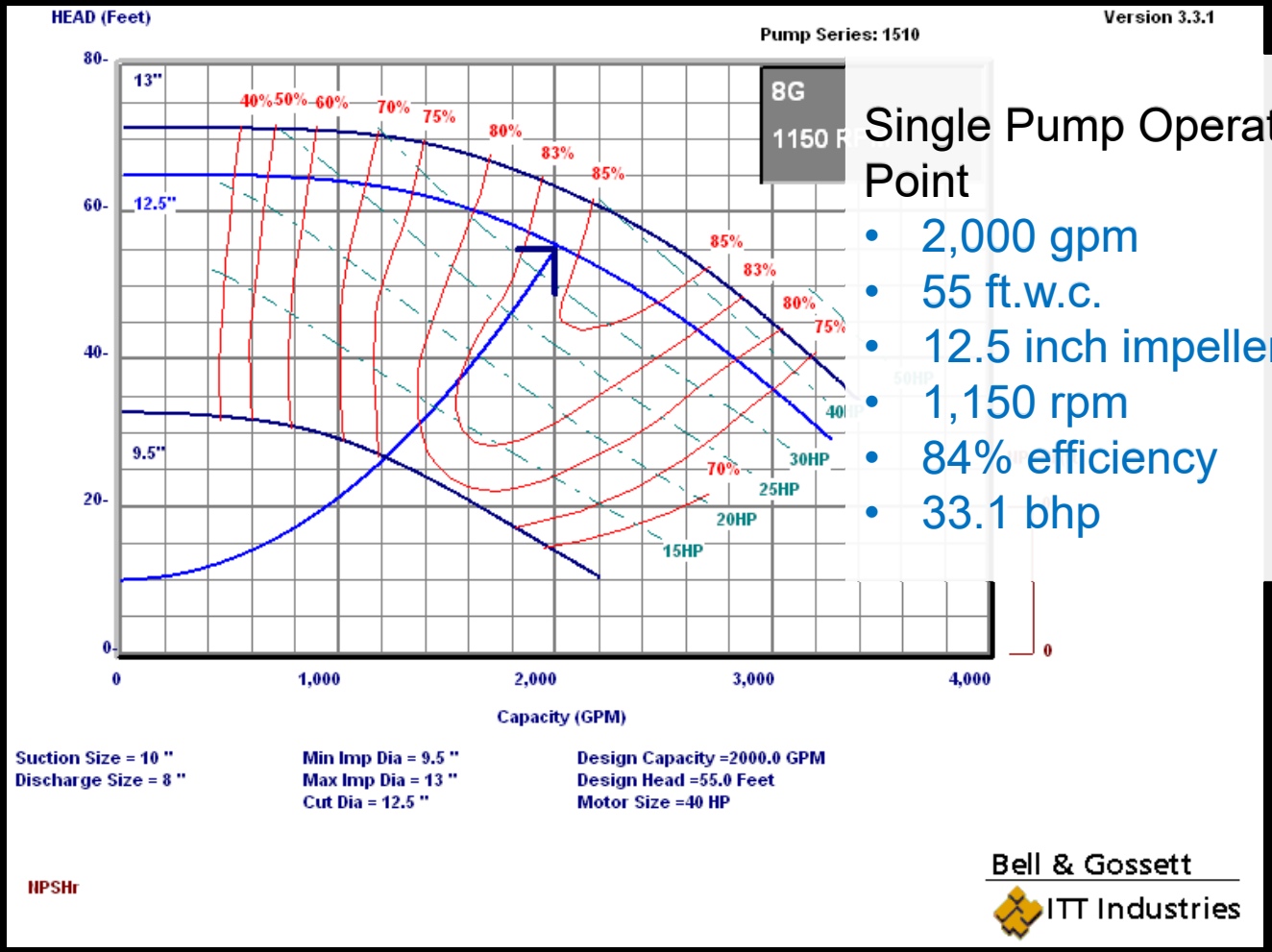
  Is there any problem with running one pump and starting the other pump if a failure is detected?



- Assume the pumps are 100% redundant

  Will running two pumps at reduced speed save energy relative to running one pump at full speed?

One Pump Operating Point

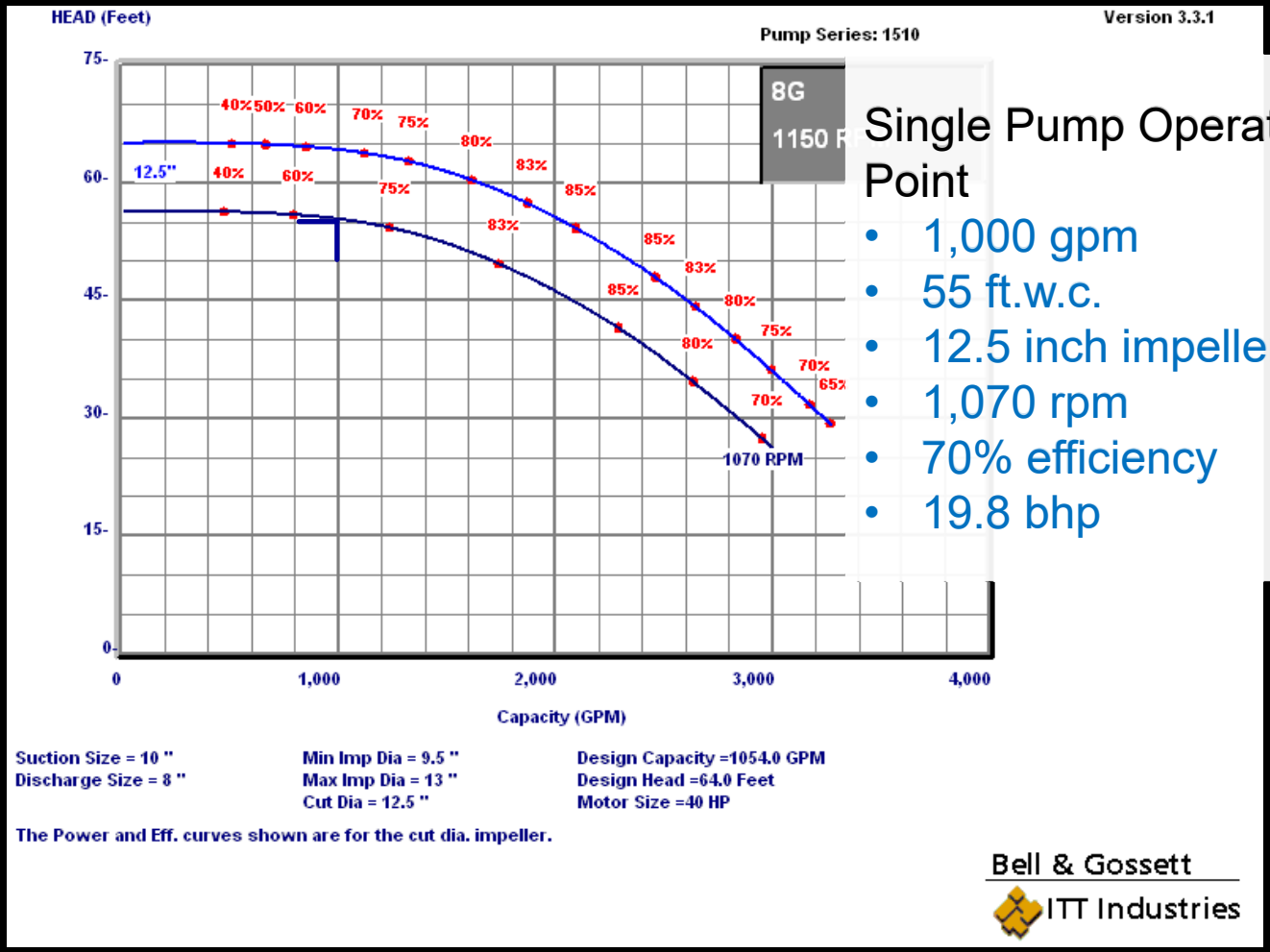


Single Pump Operating Point

- 2,000 gpm
- 55 ft.w.c.
- 12.5 inch impeller
- 1,150 rpm
- 84% efficiency
- 33.1 bhp

Two Pumps Doing What One Could Do

Each Pump Moves Half the Flow at the Design Head

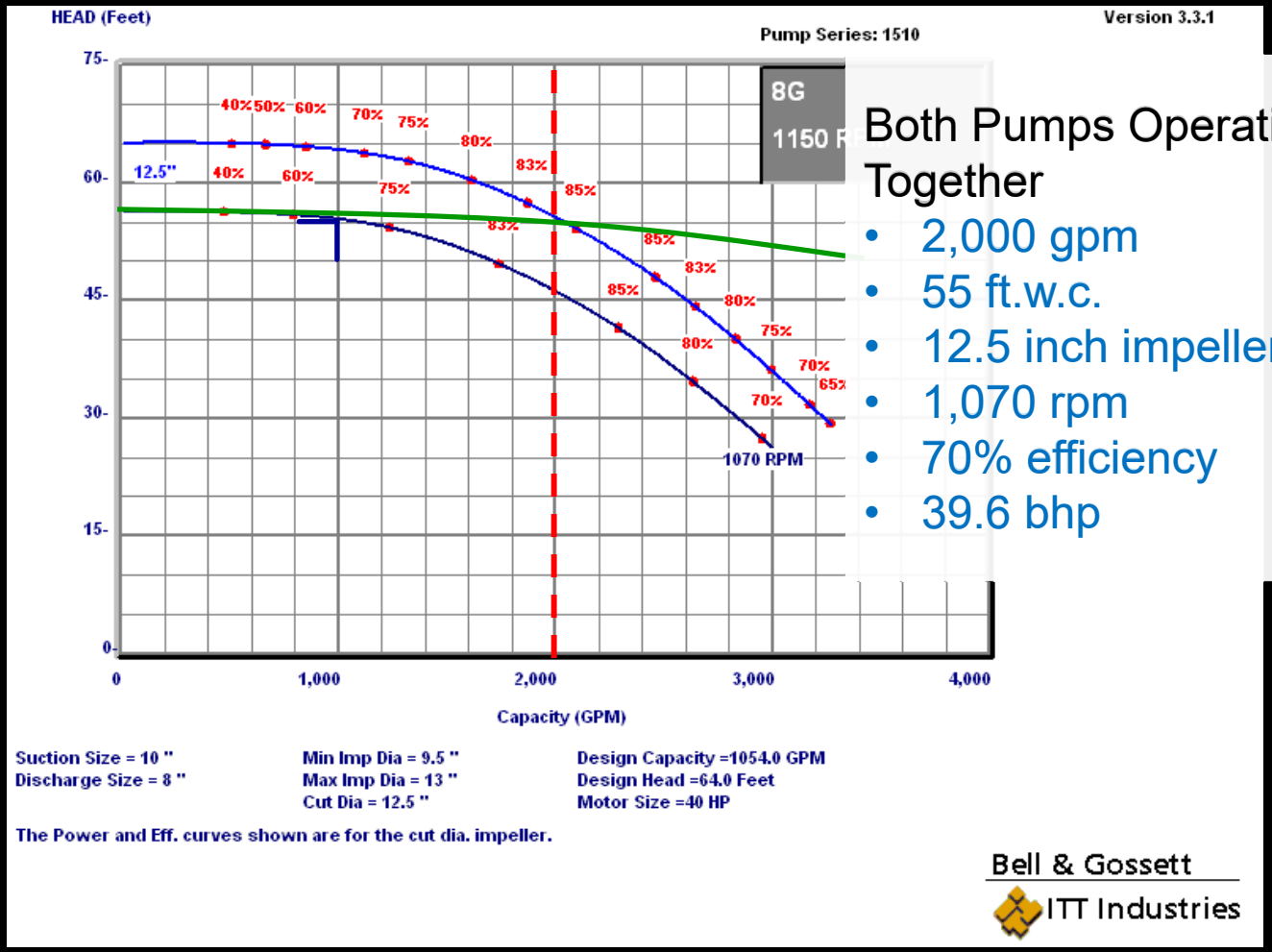


Single Pump Operating Point

- 1,000 gpm
- 55 ft.w.c.
- 12.5 inch impeller
- 1,070 rpm
- 70% efficiency
- 19.8 bhp

Two Pumps Doing What One Could Do

Each Pump Moves Half the Flow at the Design Head

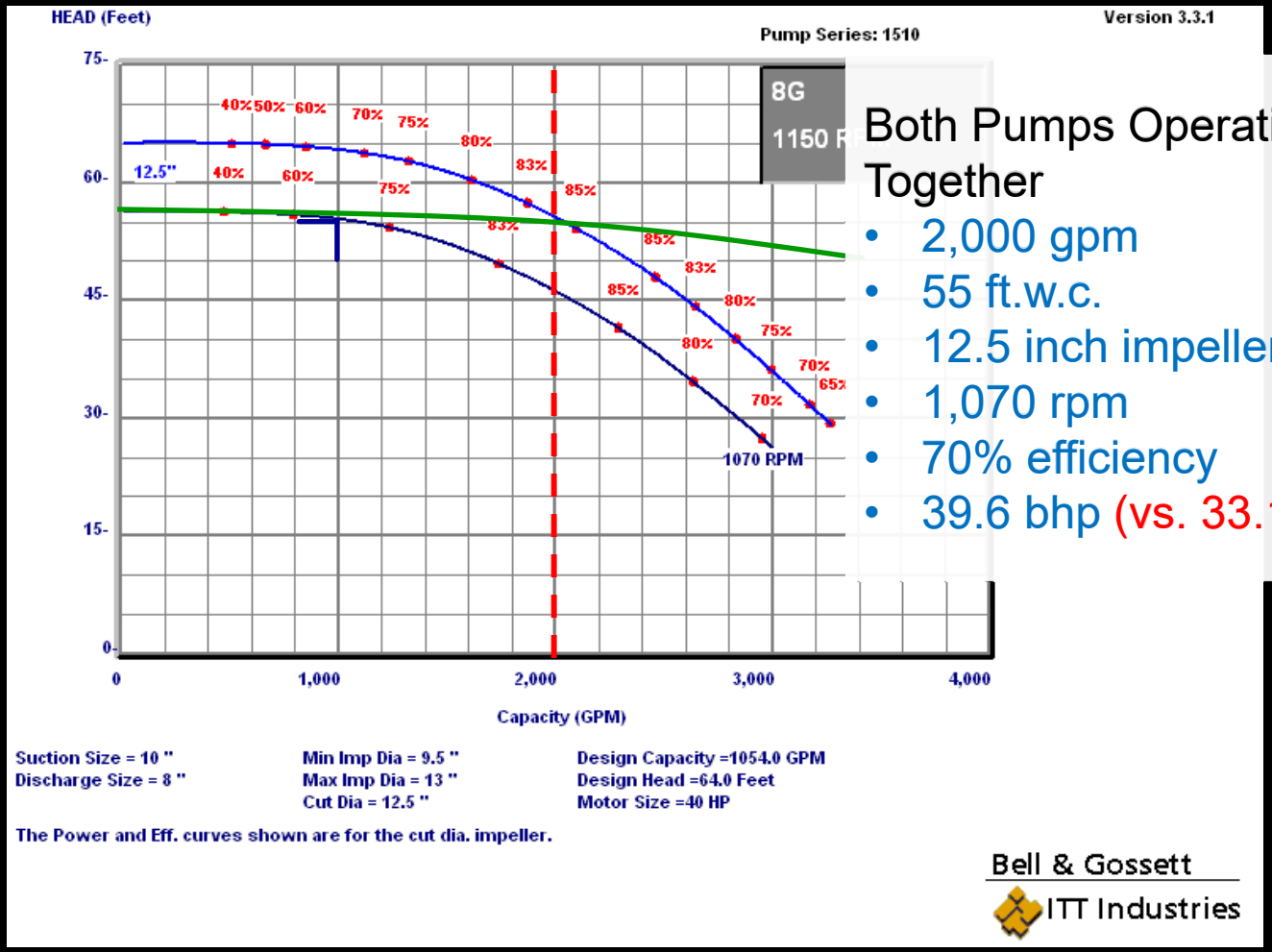


Both Pumps Operating Together

- 2,000 gpm
- 55 ft.w.c.
- 12.5 inch impeller
- 1,070 rpm
- 70% efficiency
- 39.6 bhp

Two Pumps Doing What One Could Do

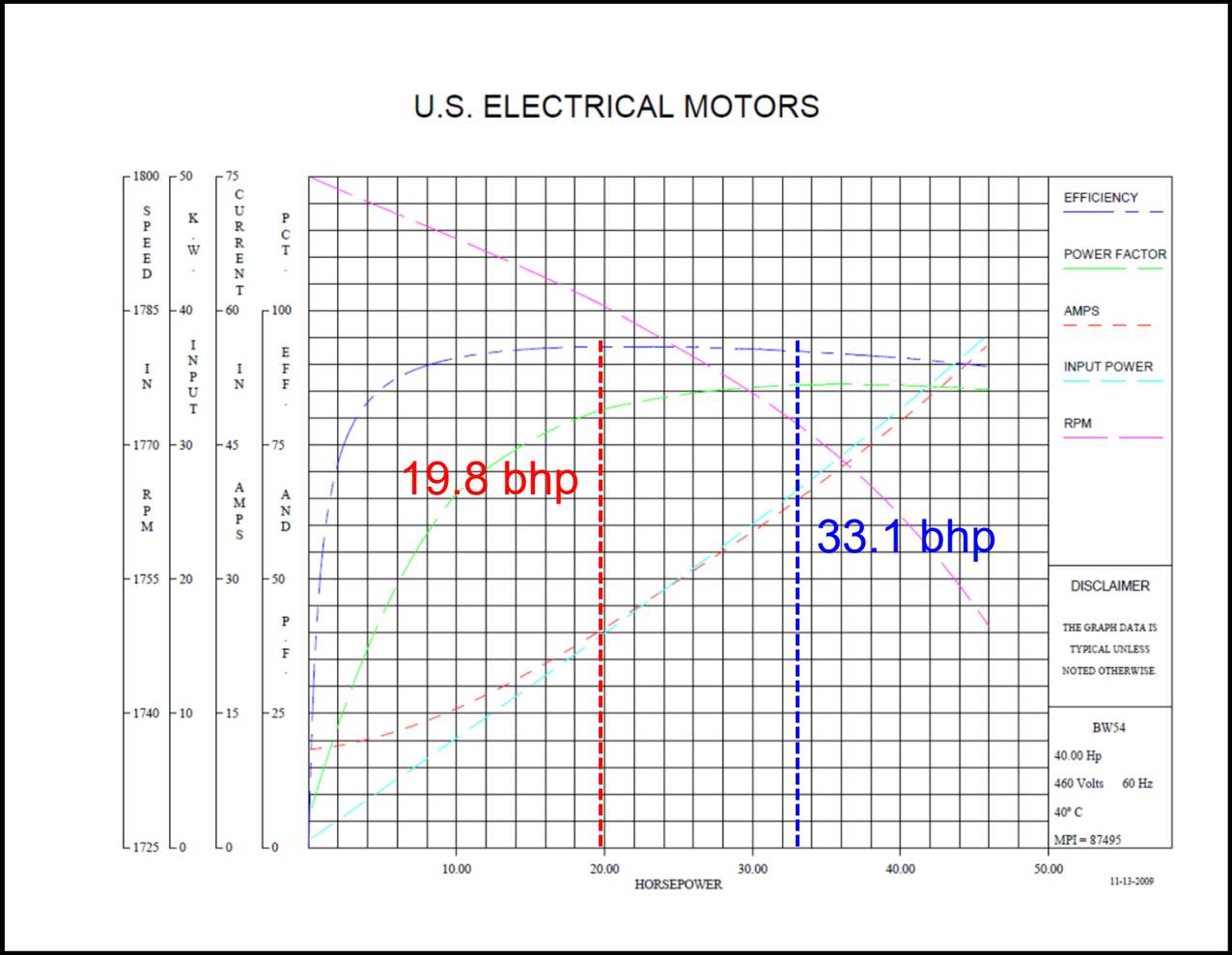
Each Pump Moves Half the Flow at the Design Head



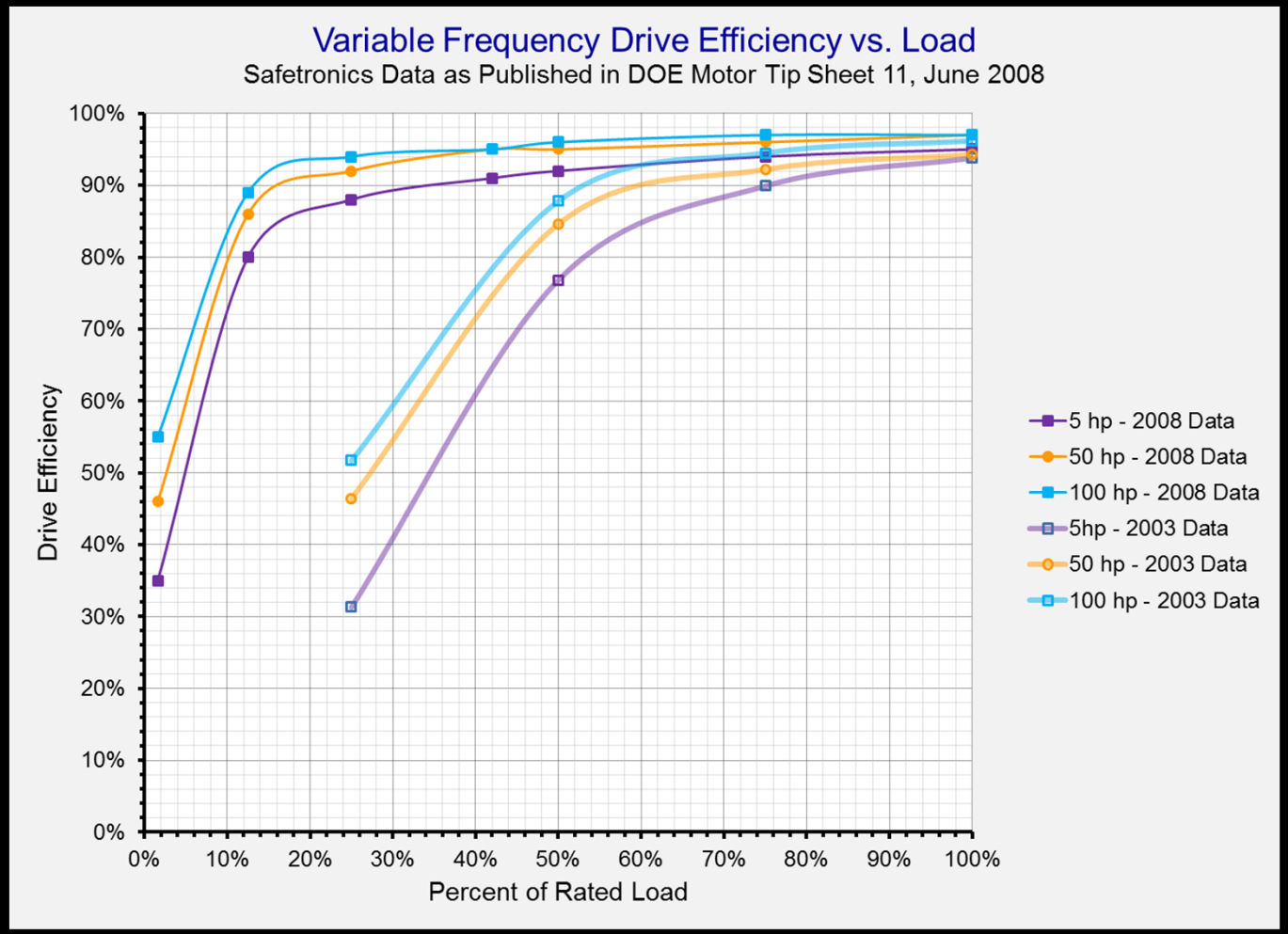
Both Pumps Operating Together

- 2,000 gpm
- 55 ft.w.c.
- 12.5 inch impeller
- 1,070 rpm
- 70% efficiency
- 39.6 bhp (vs. 33.1)

The Motor Efficiency Changed

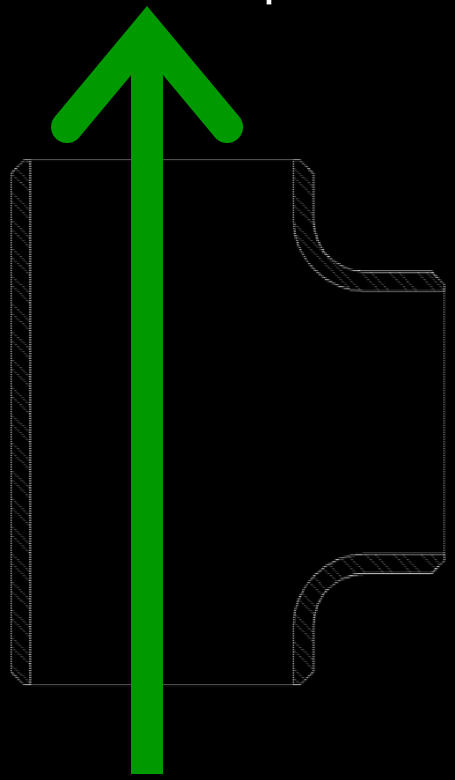


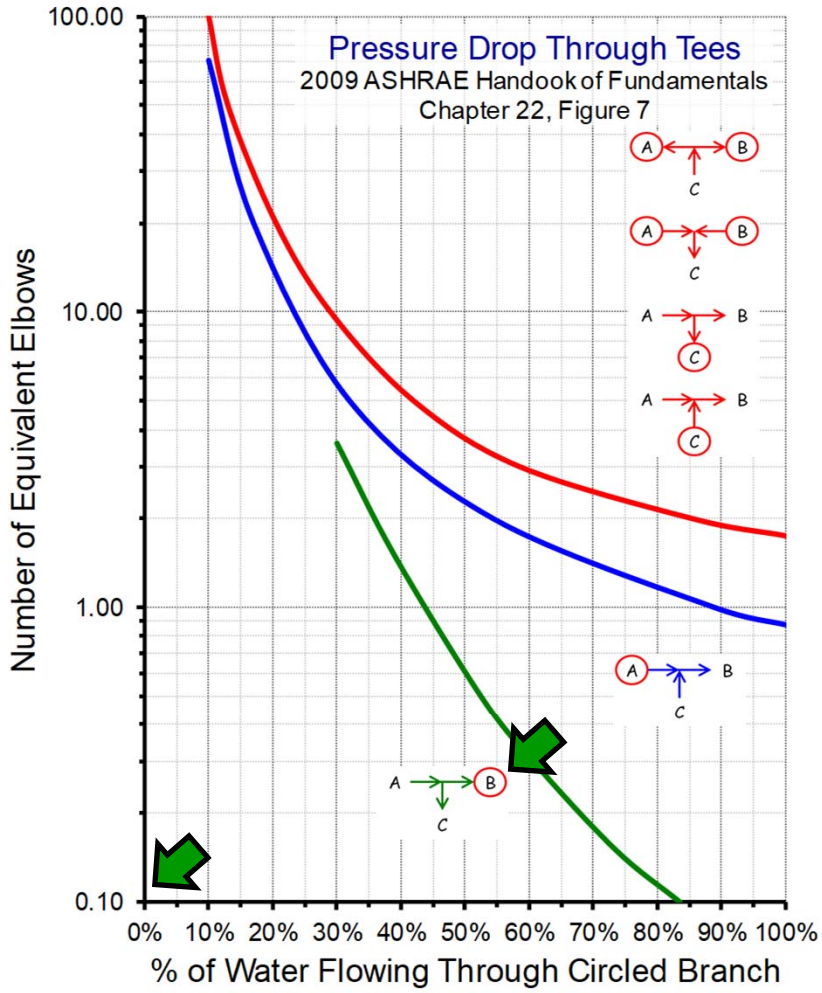
The Drive Efficiency Changed



The Fluid Mechanics Changed

One Pump



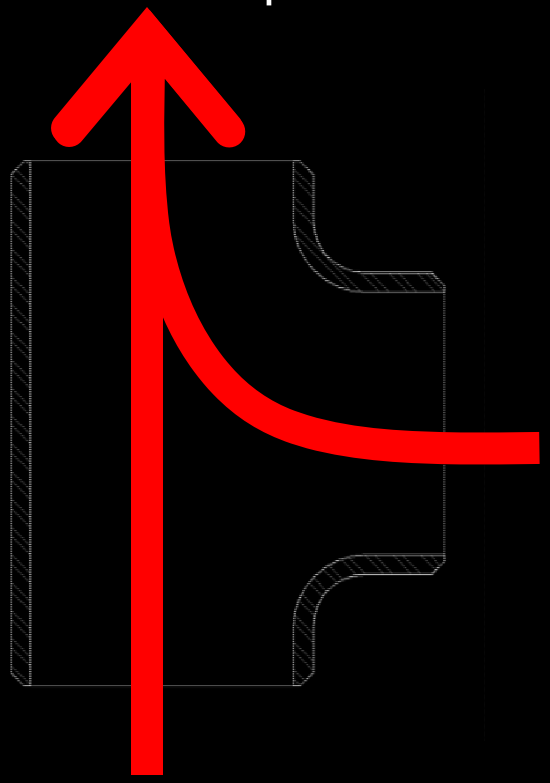


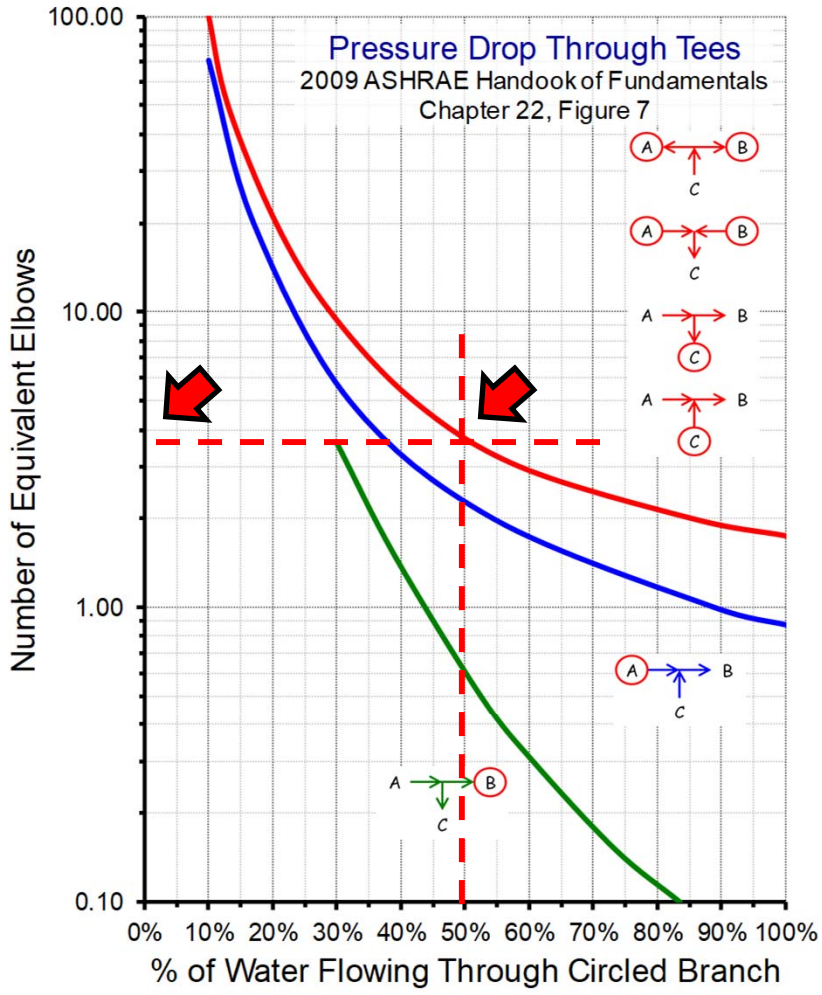
Notes:

1. Chart is based on straight tees (i.e., branches A, B, and C are the same size).
2. Pressure loss in desired circuit is obtained by selecting the proper curve according to illustrations, determining the flow at the circled branch, and multiplying the pressure loss for the same size elbow at the flow rate in the circled branch by the equivalent elbows indicated.
3. When the size of an outlet is reduced, the equivalent elbows shown in the chart do not apply. Therefore, the maximum loss for any circuit for any flow will not exceed 2 elbow equivalents at the maximum flow occurring in any branch of the tee.
4. Top curve is average of 4 curves, one for each circuit shown.
5. Data from Giesecke and Badgett 1931, 1932.

The Fluid Mechanics Changed

One Pump





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The Bottom Line

You're Still Doing the Same Amount of Work

But probably not as efficiently

$$kW = \left(\frac{Flow \times Head}{3{,}960 \times \eta_{Pump} \times \eta_{Motor} \times \eta_{Drive}} \right) \times \frac{.746\ kw}{hp}$$

Where :

kW = Power into the motor and its drive system

$Flow$ = Flow rate in gallons per minute

$Head$ = Pump head in feet water column

$3{,}960$ = A units conversion constant

η_{Pump} = Pump efficiency

η_{Motor} = Motor efficiency

η_{Drive} = Drive efficiency

$\frac{.746\ kw}{hp}$ = kW to hp conversion constant

What About Running Two Tower Fans at Half Speed vs. One at Full Speed?

Same Question for These Towers?

Same Question for These Towers?