



# 16<sup>th</sup> NATIONAL CONFERENCE ON BUILDING COMMISSIONING

## ***Design Phase Commissioning and the Rest of Life***

*The Impact of Design Phase Issues on  
Construction and On-going Operation and  
Maintenance*

Presented By:  
David Sellers, Senior Engineer  
Facility Dynamics Engineering



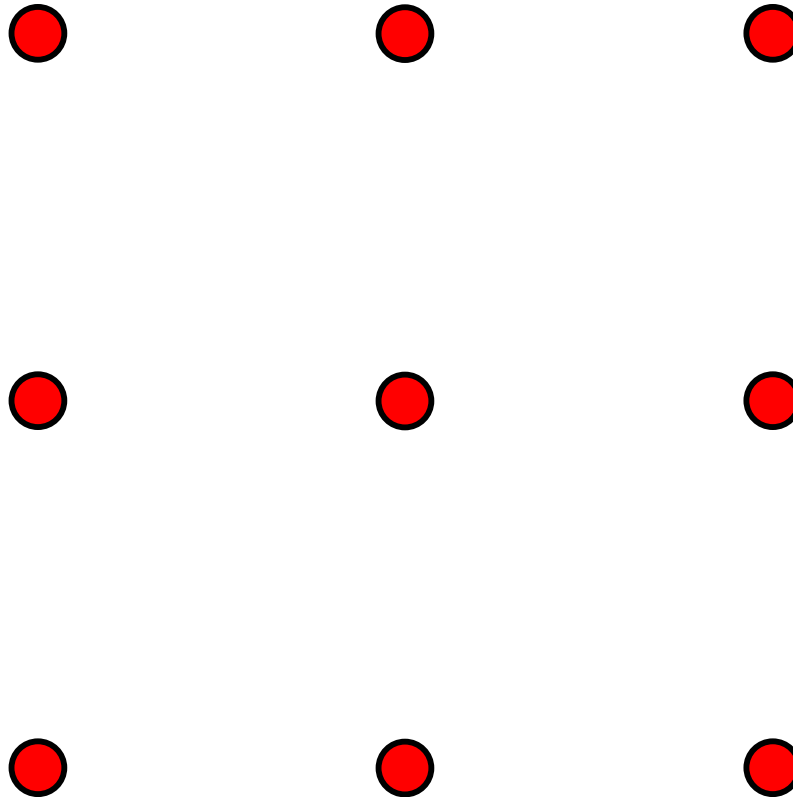
# Design Phase Commissioning; Implications Beyond Design Phase

- If somebody doesn't pay attention to the details the first time around, then Mother Nature will bring them to your attention later
- Lessons learned make your future projects better, no matter how you learned them

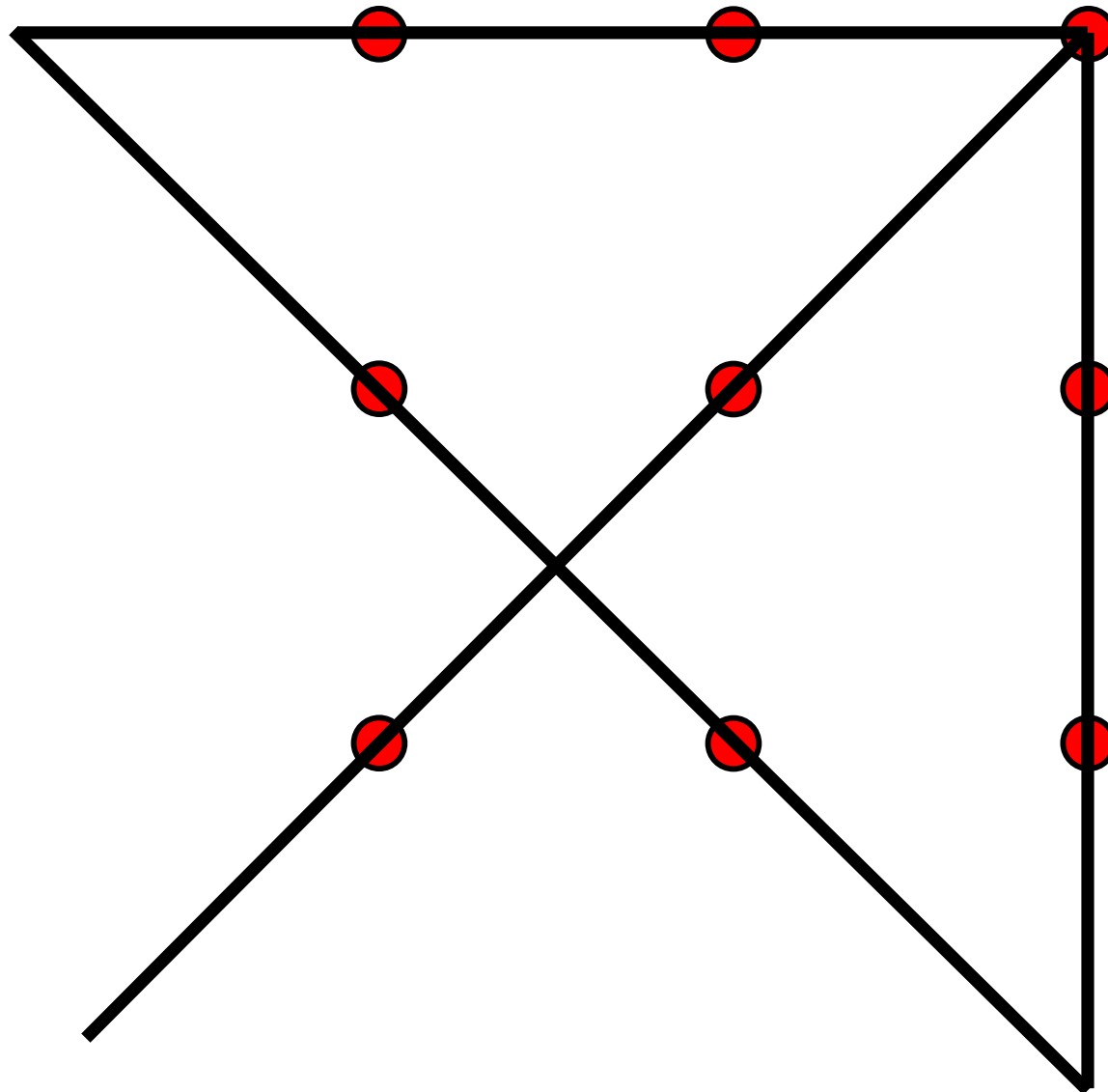


# Innovation and Design Phase Commissioning Go Hand-in-Hand

*Connect all of the dots  
with 4 straight lines  
with out lifting your  
pencil and with out  
retracing a line*



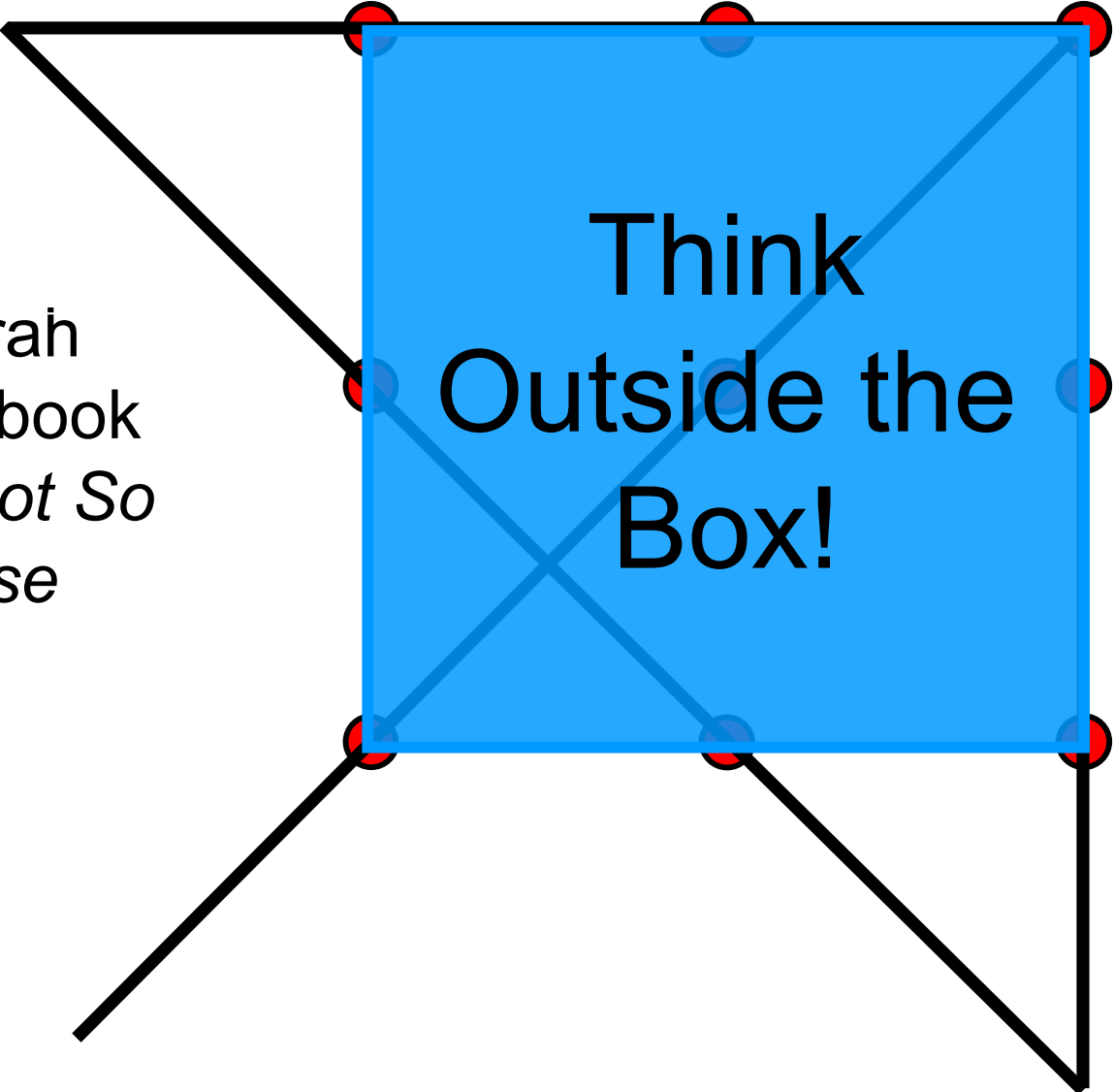
# Innovation and Design Phase Commissioning Go Hand in Hand





# Innovation and Design Phase Commissioning Go Hand in Hand

From Sarah  
Susanka's book  
titled *The Not So  
Big House*



Think  
Outside the  
Box!

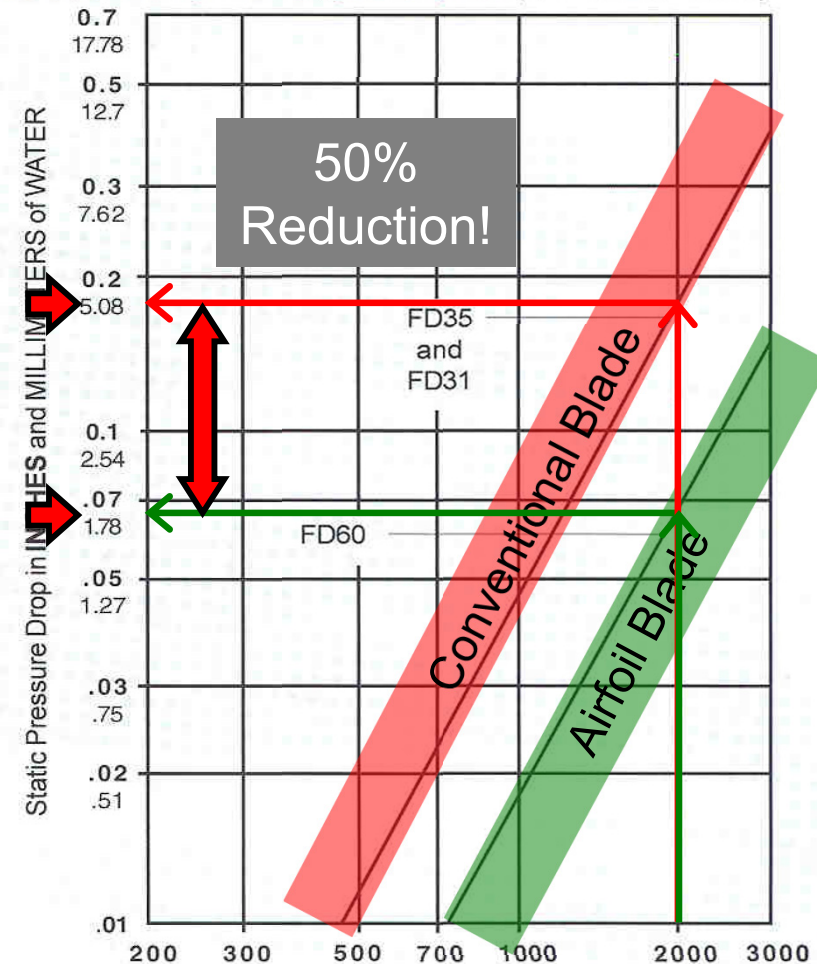
# Design is the Time to Capture Savings

- Design time well spent leads to:
  - Project capital expenditures that are optimized
  - Operating costs that are optimized
  - Equipment life cycles that are maximized

# Minimizing Pressure Drop to Capture Savings

- Upgrade smoke dampers to airfoil blade design
- Savings potential = 50%+ reduction in pressure drop for every hour of operation

Pressure Drop - Damper Open (24" x 24" size)



# Savings via a blade design change

Horse power =  $\frac{(\text{Flow in cfm}) \times (\text{Fan static pressure in in.w.c.})}{(\text{Conversion constant} \times \text{Fan efficiency} \times \text{Motor efficiency})}$

Flow rate = 46,687 cfm

Static pressure eliminated = 0.23 in.w.c. (Ruskin data for an FD60 at 3,000 fpm)

Assumed fan efficiency = 80%

Assumed motor efficiency = 85%

Fan horse power = 2.43 hp.

Kw = 1.81

Operating hours per year = 2,600

Annual kWh savings potential = 4,714 kWh per year

Assumed electrical cost = \$0.0750 \$/kWh

Annual savings potential for AHU9 = \$354 per year

# The Simple Payback at Design

Damper area = 15.82 sq.ft.

Conventional blade cost per square foot = \$88

Airfoil blade cost per square foot = \$104

Cost difference for this damper = \$257

Simple payback = 0.73 years

- The payback decays rapidly if the change is not made until after the damper is purchased or installed!

# Thinking Outside the Box to Capture Savings

- Coordinating code and life safety requirements eliminates dampers
  - Operating cost savings
  - First cost savings

NFPA required AHU smoke isolation damper (typical all systems originally)

Combination Fire and Smoke Damper at Shaft Wall (typical of all systems)

# Operating Cost Savings

Horse power =  $\frac{(\text{Flow in cfm}) \times (\text{Fan static pressure in in.w.c.})}{(\text{Conversion constant} \times \text{Fan efficiency} \times \text{Motor efficiency})}$

Flow rate = 46,687 cfm

Static pressure eliminated = 0.18 in.w.c. (Ruskin data for an FD60 at 3,000 fpm)

Assumed fan efficiency = 80%

Assumed motor efficiency = 85%

Fan horse power = 1.89 hp.

Kw = 1.41

Operating hours per year = 2,600

Annual kWh savings potential = 3,666 kWh per year

Assumed electrical cost = \$0.0750 \$/kWh

Annual savings potential for AHU9 = \$275 per year

For 16 air handling units = \$4,400 per year



# First Cost Savings

Supply damper area =	15.82	sq.ft.
Damper first cost =	\$104	\$/sq.ft.
	<u>One Unit</u>	<u>All Units</u>
Supply damper savings	\$1,642	\$26,266
Return damper savings	<u>\$3,283</u>	<u>\$52,532</u>
Total savings	\$4,925	<u>\$78,798</u>

- This does not include the installation costs and wiring, so the actual savings could easily be twice this much!

# Design is the Time to Capture Intent

- *Design Intent* is integral to the commissioning process
  - Defines functional testing requirements
  - Defines acceptance requirements
  - Defines on-going operating parameters

# Thinking Outside the Box in Seattle

The Owners intent to provide a high quality internal environment leads the commissioning provider to ask a question during a design review meeting

*Do we really need to humidify in Seattle?*



*Hmmmm, lets think about that for a minute ....*

# Looking Humidification in the Bigger Picture

- Lost
  - Absolute assurance that indoor humidity will not be below 30% for 100 hours per year
- Gained
  - \$100,000 savings in first cost
  - \$6,000 savings in annual operating cost

# Design Details and Excellence Go Hand in Hand

*The magic behind every outstanding performance is always found in the smallest of details*

Gary Ryan Blair

<http://ezinearticles.com/?Pay-Attention-to-Details&id=245279>

# What Kind of Details Matter in HVAC?

- Application – Is there a match?

2000 ASHRAE Systems and Equipment Handbook

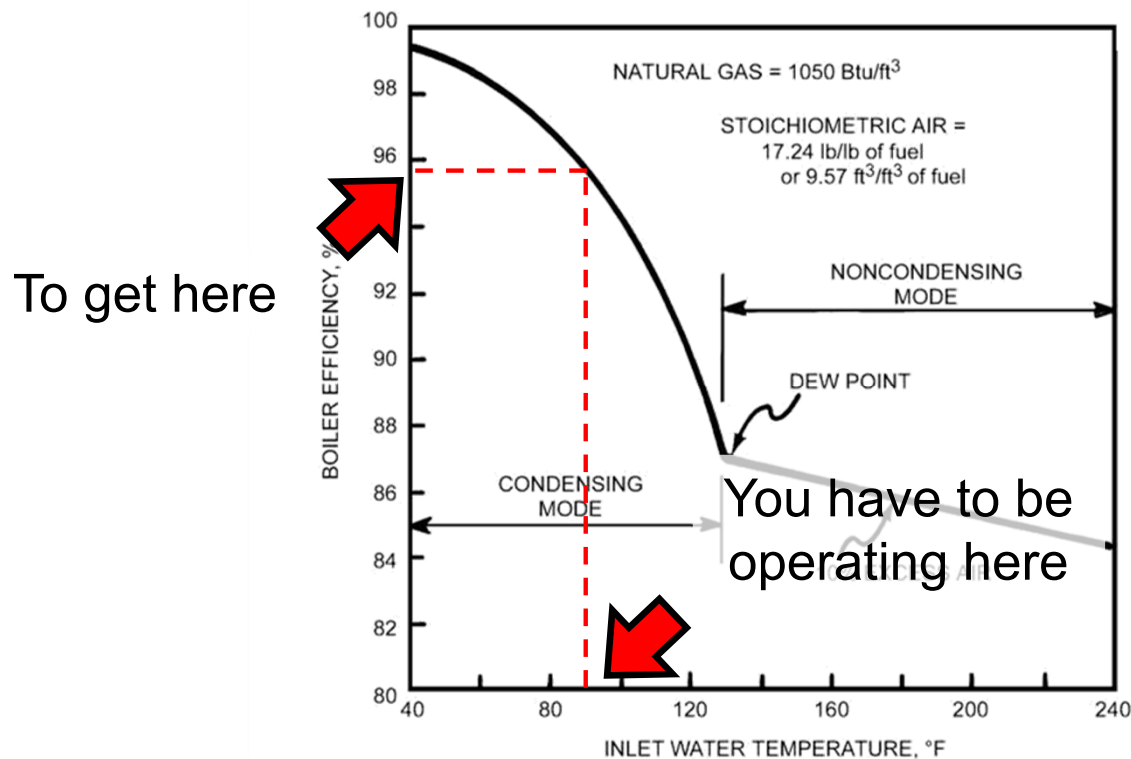
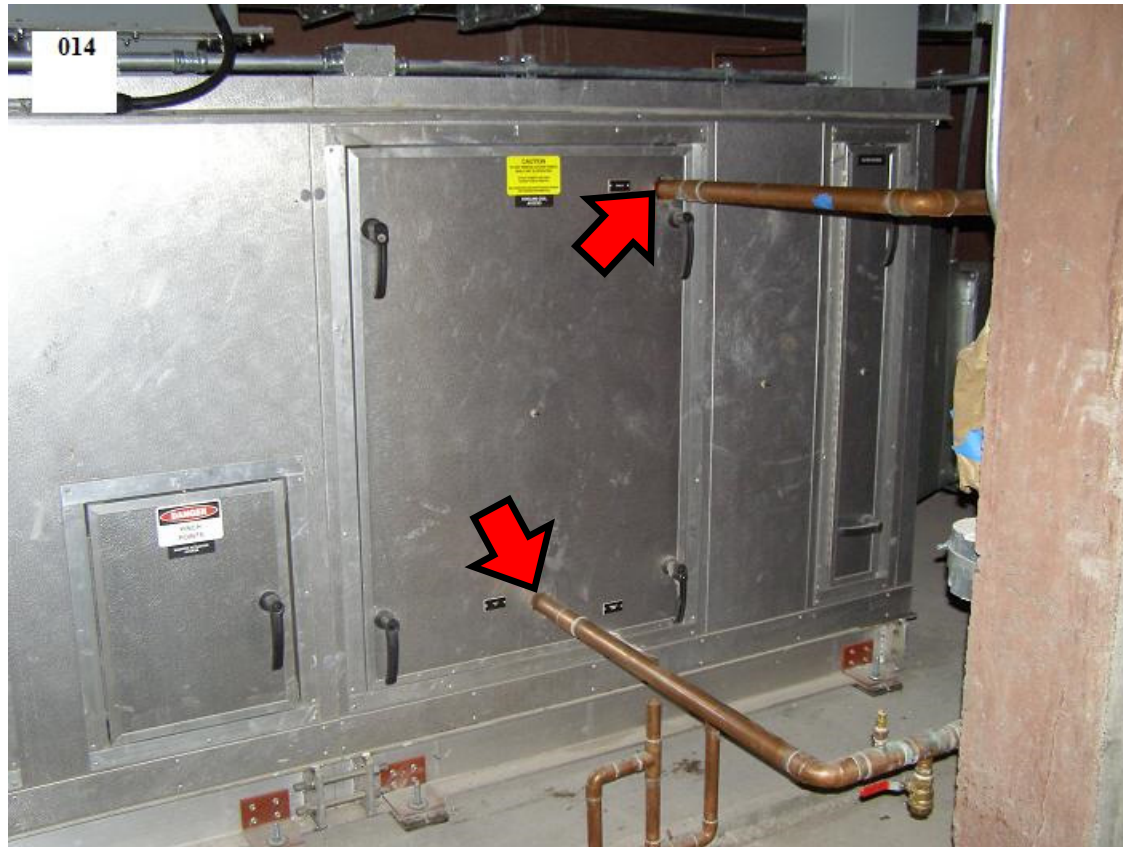


Fig. 5 Effect of Inlet Water Temperature on Efficiency of Condensing Boilers

# What Kind of Details Matter in HVAC?

- Application – Can it work as intended?

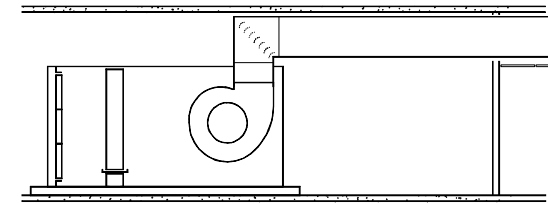
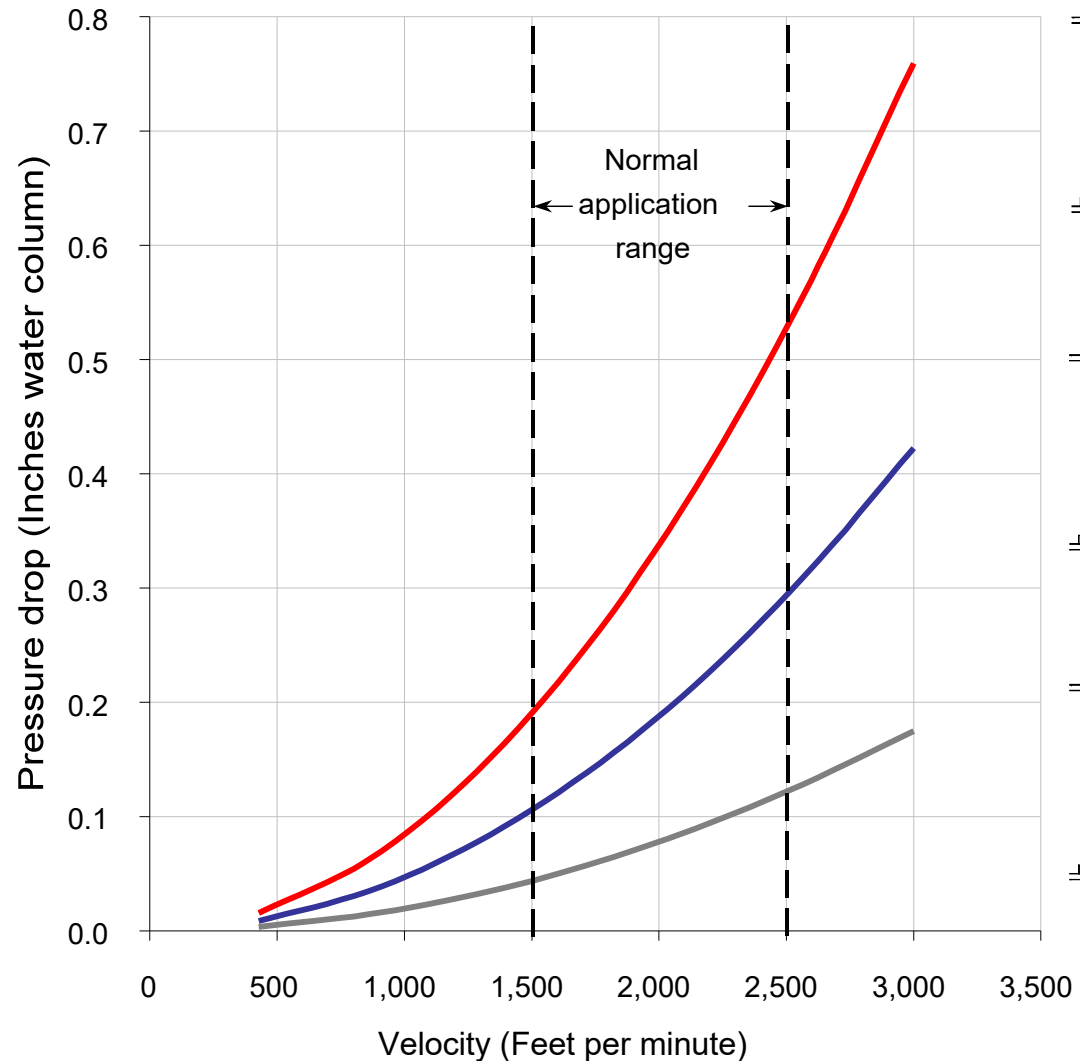


# What Kind of Details Matter in HVAC?

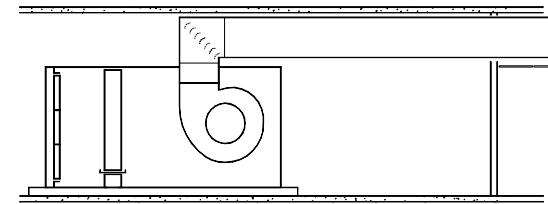
- Configuration
  - Will it fit?
  - Is it shaped right?
  - Is it appropriately positioned relative to the system?



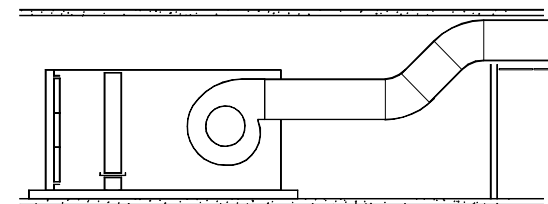
# Configuration and Fan Applications



Top discharge, reversed turn

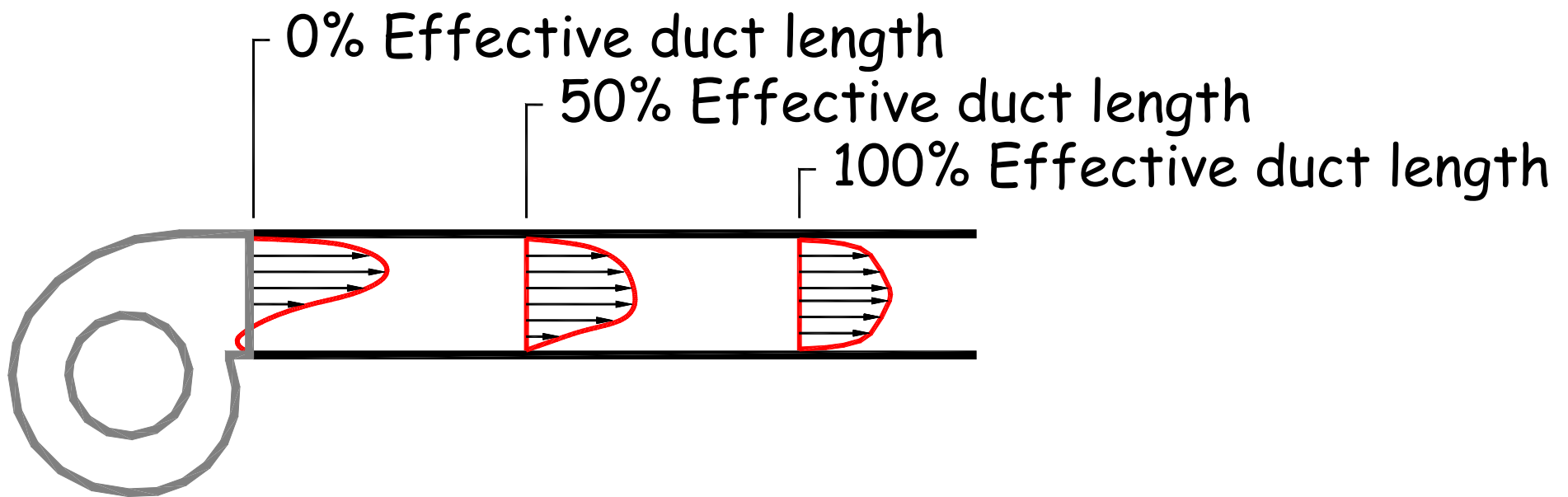


Top discharge, forward turn



Front discharge with offset after  
100% effective duct length

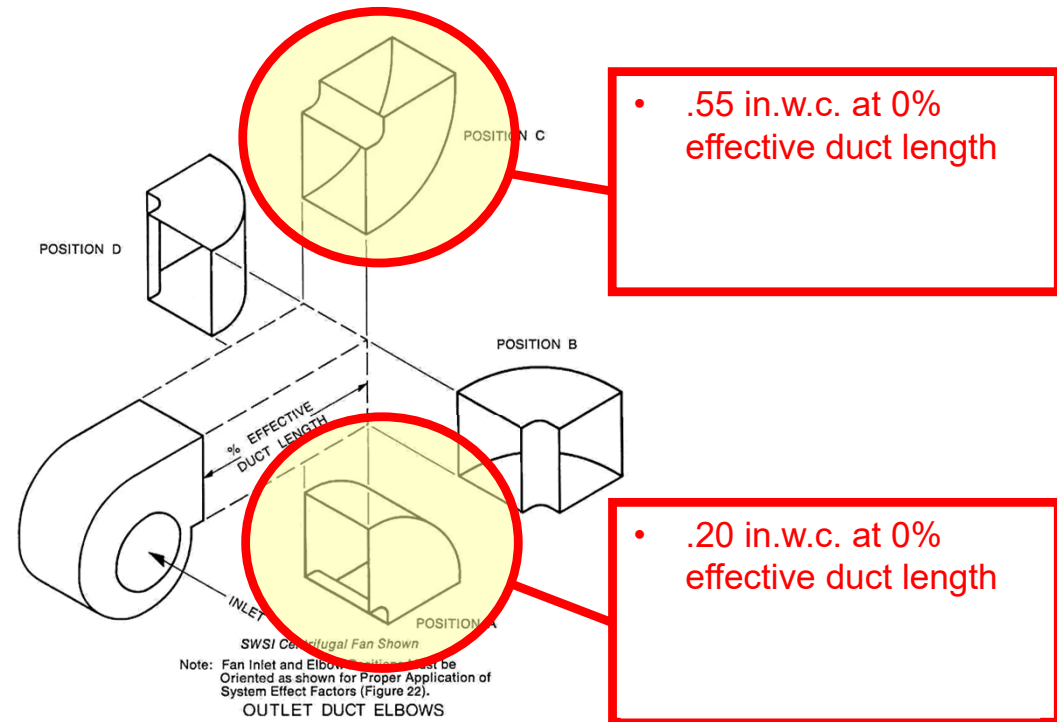
# Fan Tests are Based on Ideal Conditions



# System Effect

*The effect of the system connections on the fan's performance.*

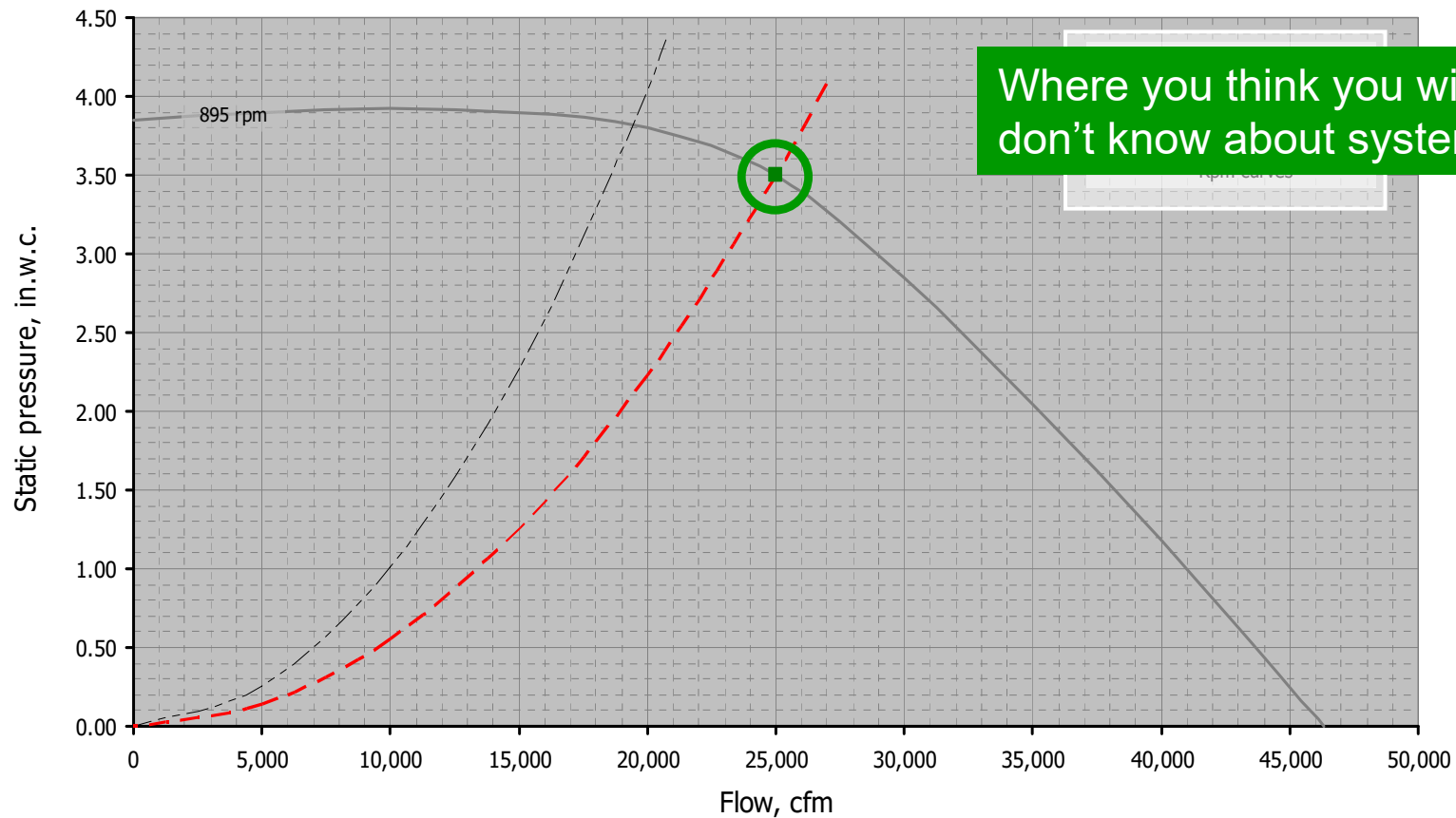
- Accounted for by a system effect factor
- Velocity dependent
- Connection configuration dependent
  - Relative to discharge velocity profile
  - Relative to distance from fan



*System effect assessed at 2,500 fpm and an outlet area to blast area ratio of 70%*

# Performance Implications

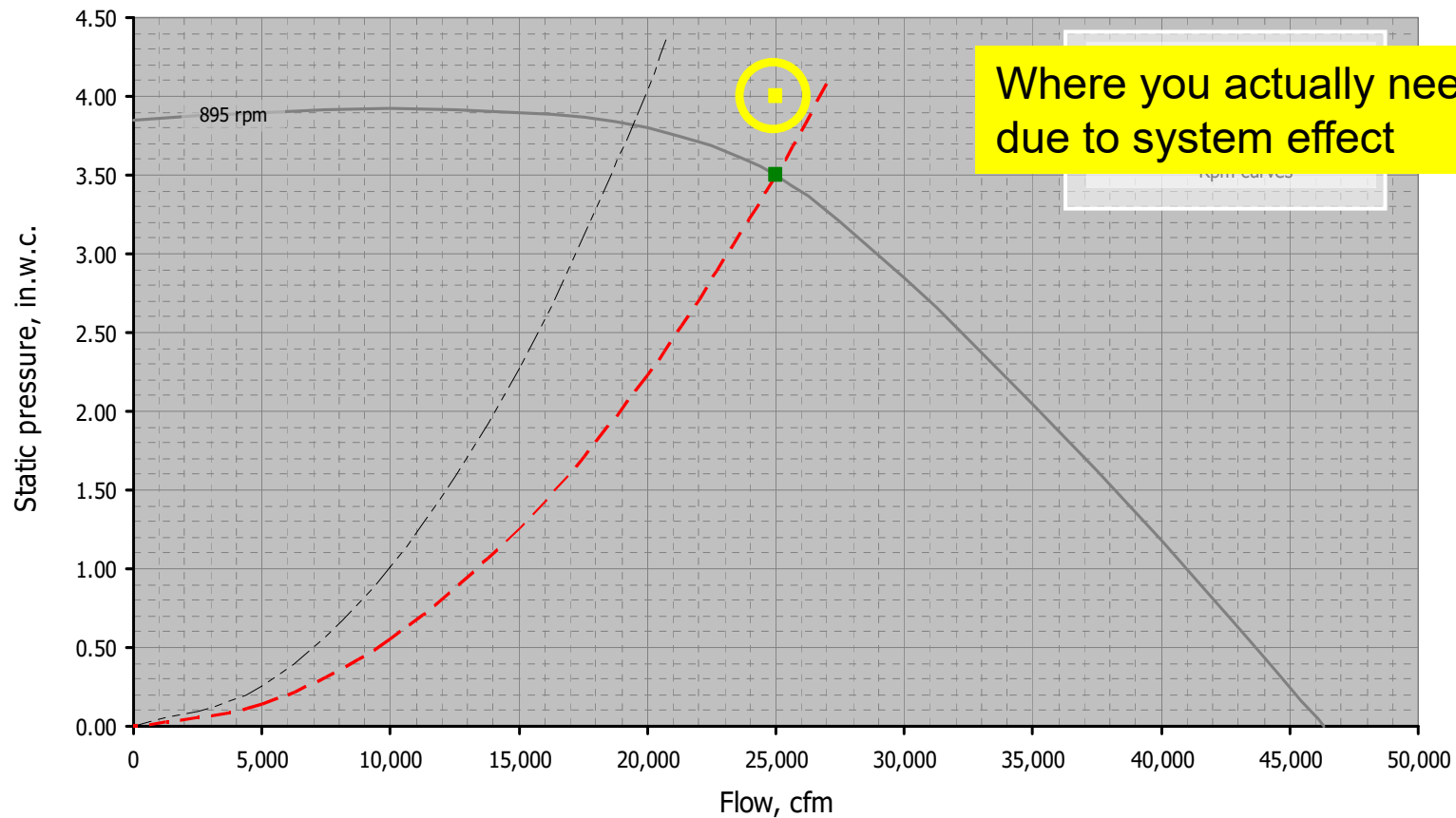
Supply Fan - Greenheck 36-AFDW-41



Where you think you will be if you don't know about system effect

# Performance Implications

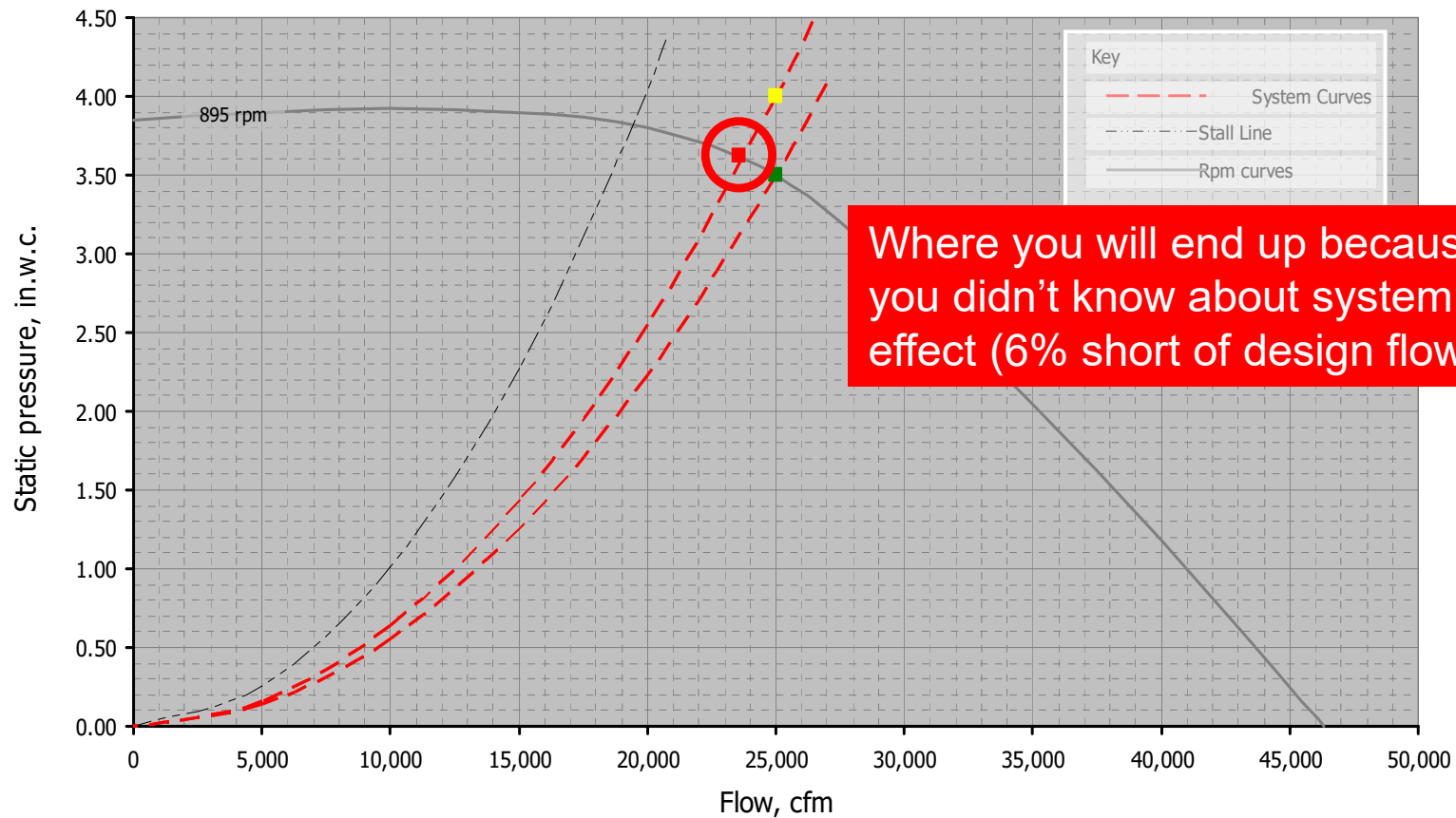
Supply Fan - Greenheck 36-AFDW-41



Where you actually need to be due to system effect

# Performance Implications

Supply Fan - Greenheck 36-AFDW-41



# Fan Energy is Directly Related to Flow and Fan Static Pressure

- Flow rate – 25,000 cfm
- Unnecessary static pressure burden – 0.25 in.w.c.  
– 72%  
used – 1.4 bhp

# This Problem is Often Solved by Throwing Energy at It

- Fan brake horsepower requirement is typically less than the incremental motor horsepower supplied
- Motor service factor provides some margin for error
  - For our example:
    - Brake horsepower at design is approximately 18 bhp
    - Brake horsepower required if system effect is accommodated is approximately 21 bhp
    - Horsepower available from a 20 hp motor with a service factor of 1.15 is 23 hp



# This Problem is Often Solved by Throwing Energy at It

- For our example:
  - Speed the fan up and everyone wins!

# This Problem is Often Solved by Throwing Energy at It

- For our example:
  - Speed the fan up and everyone wins!

*(Except the planet)*

*We don't inherit the world  
from our ancestors;  
We borrow it from our children*

Unknown



Image Science and Analysis Laboratory, NASA-Johnson Space Center. "The Gateway to Astronaut Photography of Earth."

# This Problem is Often Solved by Throwing Energy at It

- For our example:
  - Speed the fan up and everyone wins!

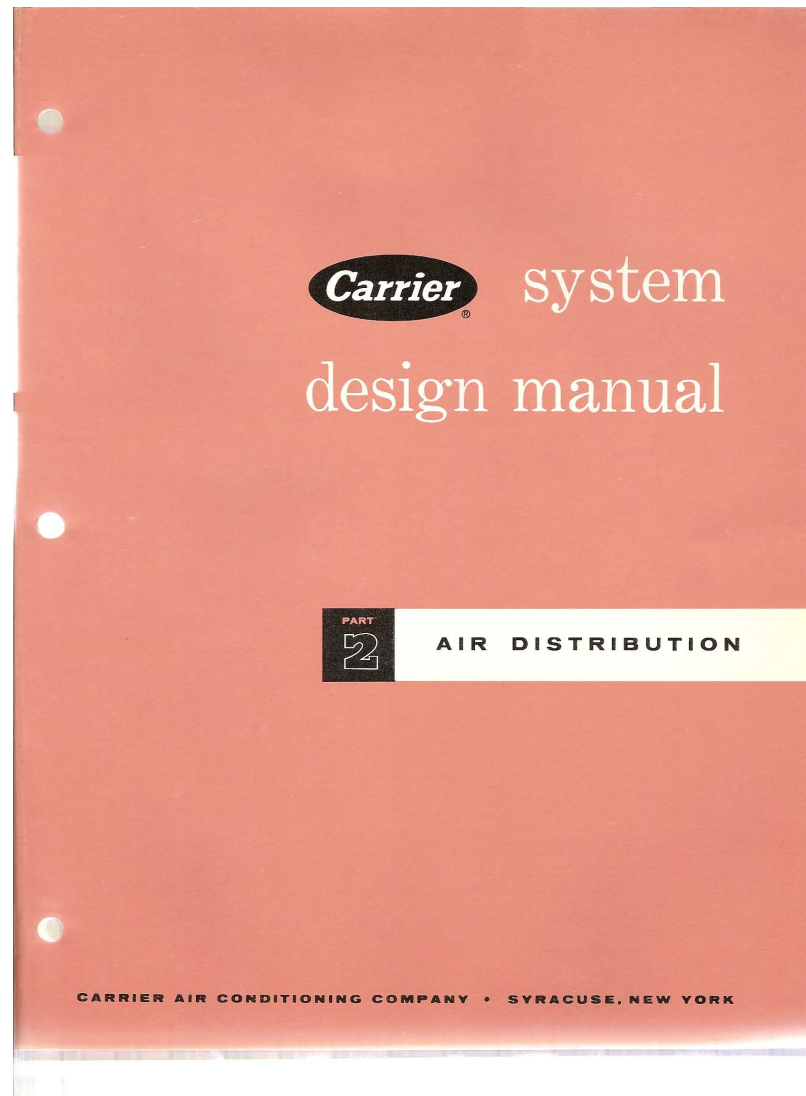


*(Except the planet)*

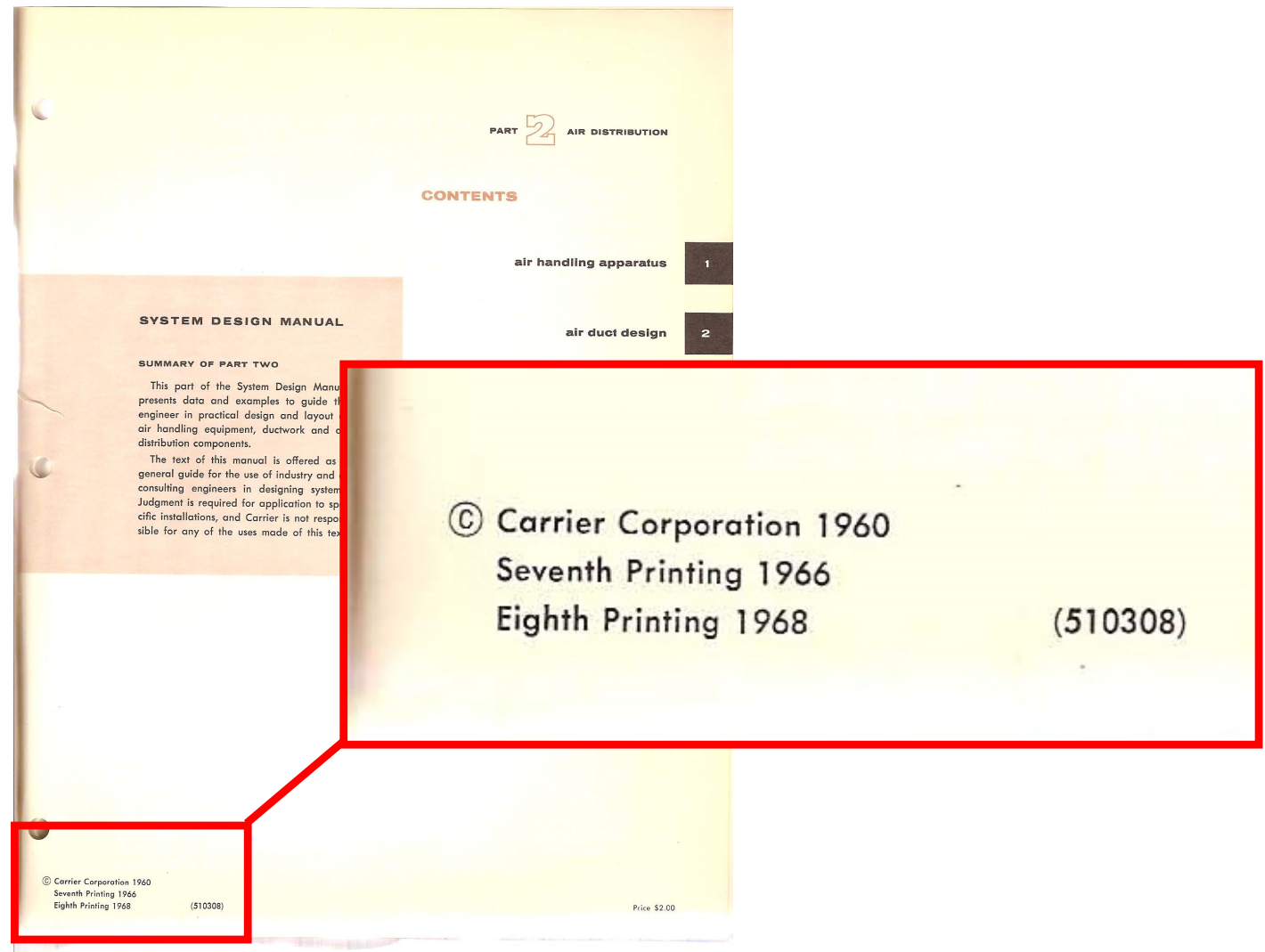
*We don't inherit the world  
from our ancestors;  
We borrow it from our children*

Unknown

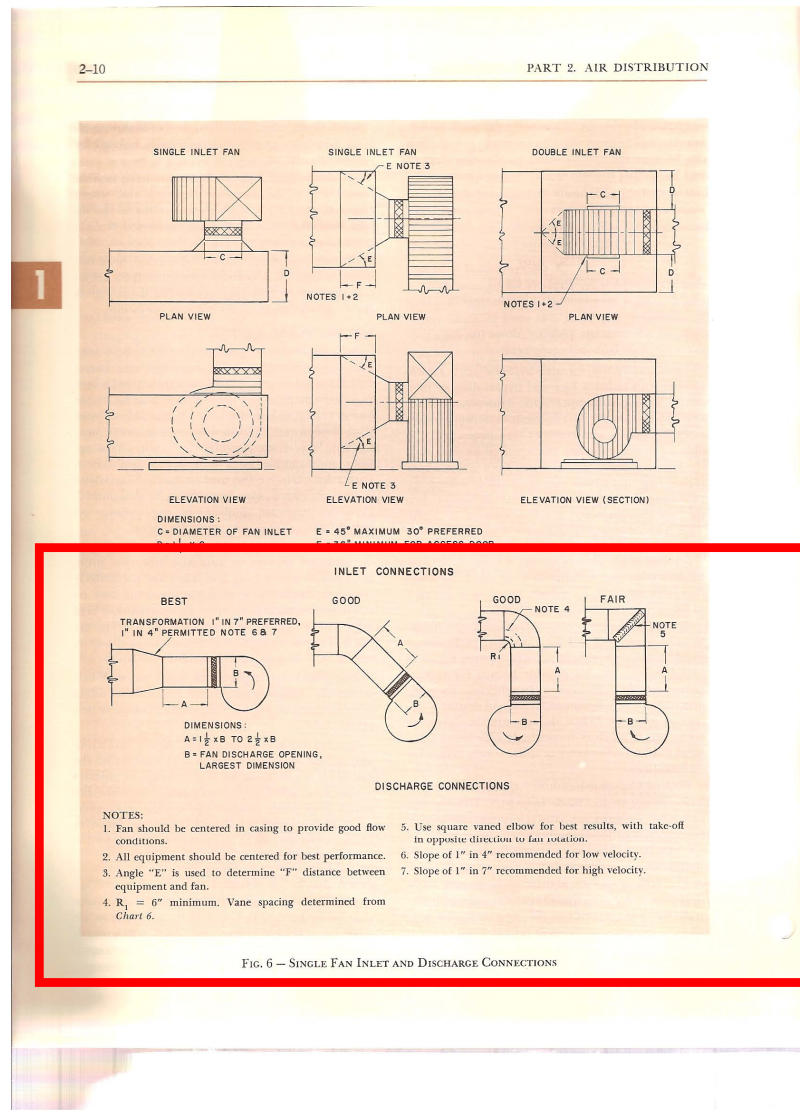
# A Known Phenomenon



# A Known Phenomenon

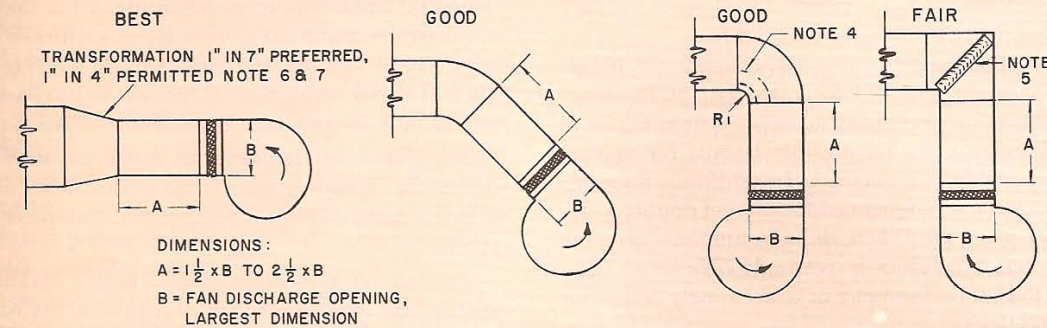
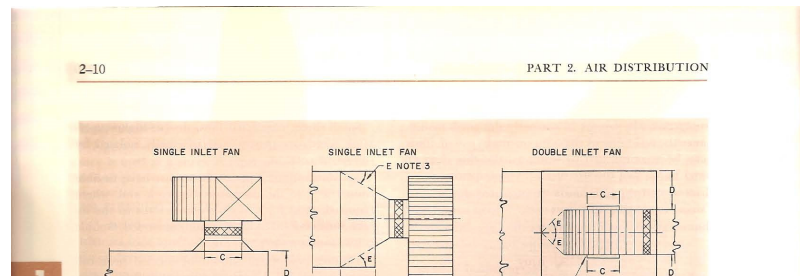


# A Known Phenomenon





# A Known Phenomenon



## NOTES:

1. Fan should be centered in casing to provide good flow conditions.
2. All equipment should be centered for best performance.
3. Angle "E" is used to determine "F" distance between equipment and fan.
4.  $R_1 = 6"$  minimum. Vane spacing determined from Chart 6.
5. Use square vaned elbow for best results, with take-off in opposite direction to fan rotation.
6. Slope of 1" in 4" recommended for low velocity.
7. Slope of 1" in 7" recommended for high velocity.

FIG. 6 — SINGLE FAN INLET AND DISCHARGE CONNECTIONS

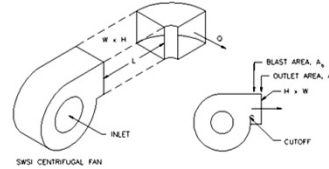
# Current ASHRAE Handbooks and Fitting Database

## Duct Design

34.67

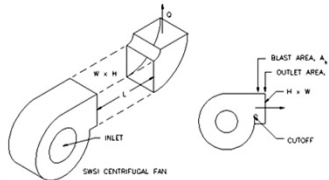
### SR7-6 Fan Outlet, Centrifugal, SWSL, with Elbow (Position B)

$A_p/A_e$	$C_p$ Values					
	$L/L_e$					
	0.00	0.12	0.25	0.50	1.00	10.00
0.4	3.80	3.20	2.20	1.00	0.00	0.00
0.5	2.90	2.20	1.60	0.67	0.00	0.00
0.6	2.00	1.60	1.20	0.53	0.00	0.00
0.7	1.40	1.00	0.67	0.33	0.00	0.00
0.8	1.00	0.80	0.53	0.26	0.00	0.00
0.9	0.80	0.67	0.47	0.18	0.00	0.00
1.0	0.67	0.53	0.40	0.18	0.00	0.00



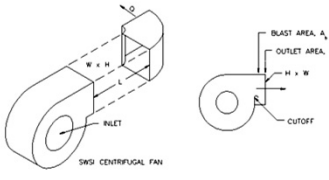
### SR7-7 Fan Outlet, Centrifugal, SWSL, with Elbow (Position C)

$A_p/A_e$	$C_p$ Values					
	$L/L_e$					
	0.00	0.12	0.25	0.50	1.00	10.00
0.4	5.50	4.50	3.20	1.60	0.00	0.00
0.5	3.80	3.20	2.20	1.00	0.00	0.00
0.6	2.90	2.50	1.60	0.80	0.00	0.00
0.7	2.00	1.60	1.00	0.53	0.00	0.00
0.8	1.40	1.20	0.80	0.33	0.00	0.00
0.9	1.20	0.80	0.67	0.26	0.00	0.00
1.0	1.00	0.80	0.53	0.26	0.00	0.00



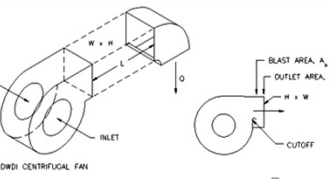
### SR7-8 Fan Outlet, Centrifugal, SWSL, with Elbow (Position D)

$A_p/A_e$	$C_p$ Values					
	$L/L_e$					
	0.00	0.12	0.25	0.50	1.00	10.00
0.4	5.50	4.50	3.20	1.60	0.00	0.00
0.5	3.80	3.20	2.20	1.00	0.00	0.00
0.6	2.90	2.50	1.60	0.80	0.00	0.00
0.7	2.00	1.60	1.00	0.53	0.00	0.00
0.8	1.40	1.20	0.80	0.33	0.00	0.00
0.9	1.20	0.80	0.67	0.26	0.00	0.00
1.0	1.00	0.80	0.53	0.26	0.00	0.00



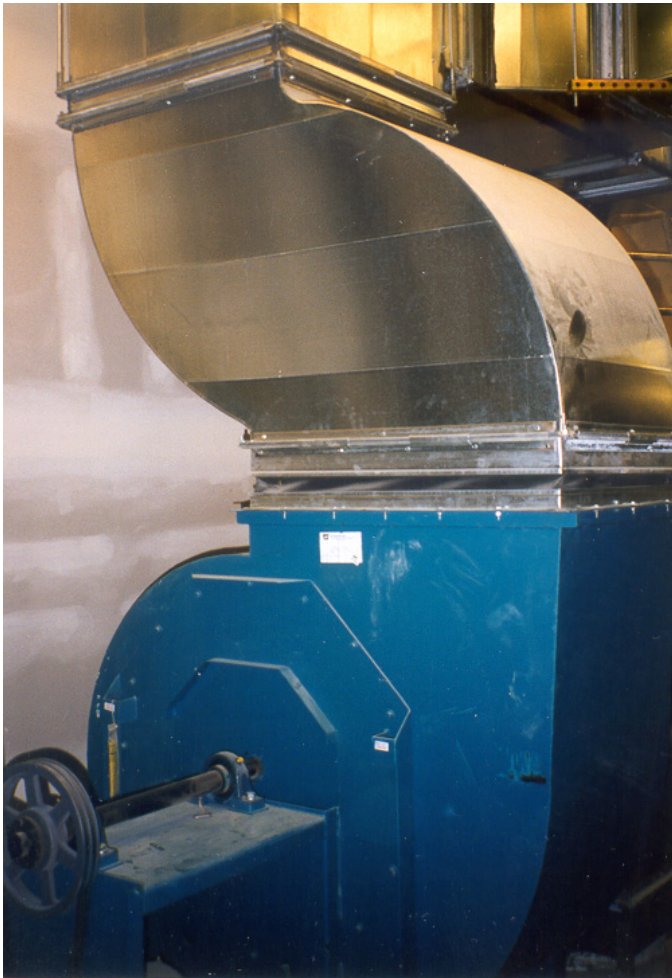
### SR7-9 Fan Outlet, Centrifugal, DWDL, with Elbow (Position A)

$A_p/A_e$	$C_p$ Values					
	$L/L_e$					
	0.00	0.12	0.25	0.50	1.00	10.00
0.4	3.20	2.50	1.80	0.80	0.00	0.00
0.5	2.20	1.80	1.20	0.53	0.00	0.00
0.6	1.60	1.40	0.80	0.40	0.00	0.00
0.7	1.00	0.80	0.53	0.26	0.00	0.00
0.8	0.80	0.67	0.47	0.18	0.00	0.00
0.9	0.53	0.47	0.33	0.18	0.00	0.00
1.0	0.53	0.47	0.33	0.18	0.00	0.00





# Field Experience Indicates Some Room for Improvement



*Left image courtesy of  
HPAC Editorial Advisory  
Board Member Ron  
Wilkinson*



# Why Does This Matter?



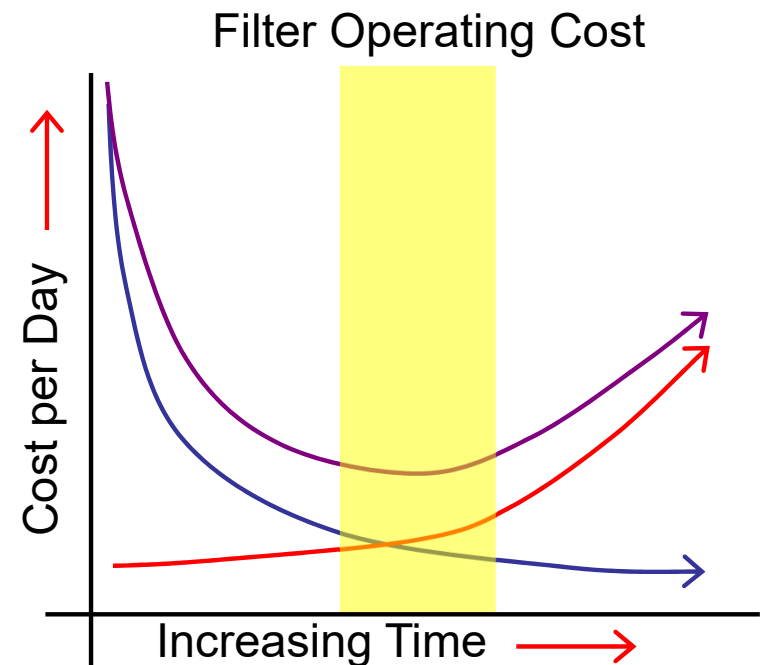
[http://www.energy.ca.gov/pier/final\\_project\\_reports/500-03-082.html](http://www.energy.ca.gov/pier/final_project_reports/500-03-082.html)

Recent PIER (California's Public Interest Energy Research program) found that:

- For small commercial buildings (30,000 – 50,000 sq.ft.)
  - Installed fan power exceeds ARI assumptions
    - Fan scheduling and control
    - Fan sizing and distribution system issues
  - Best practices savings potential – 10-15% over current approaches
- Similar conclusion for large commercial buildings

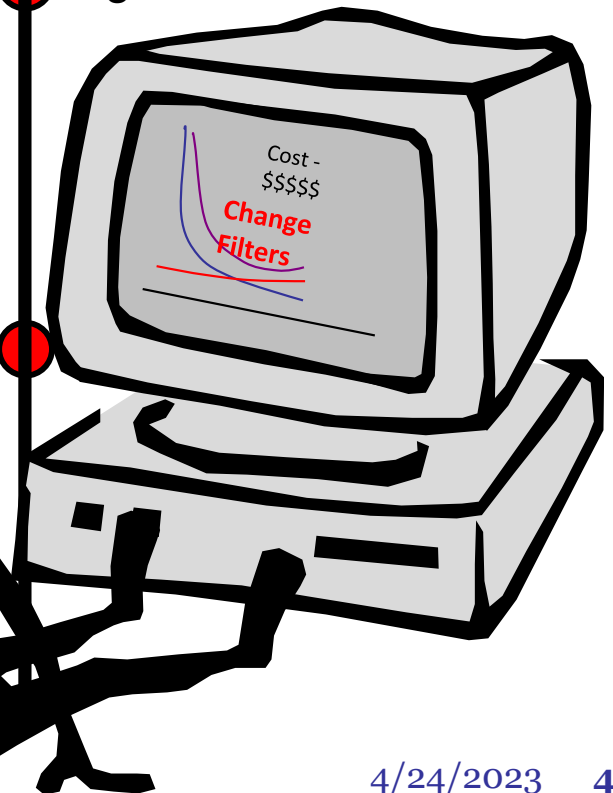
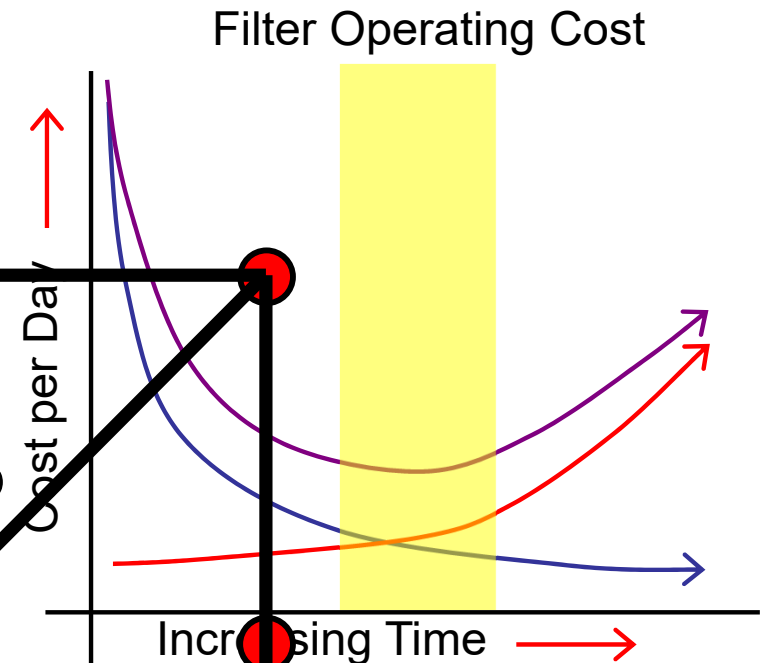
# Do You Know What you Need to Know for Optimum Operation?

- First cost component
  - Decreases over time
  - Non-linear
    - Day 1 – Cost per day = Cost of filter set
    - Day X – Cost per day = (Cost of filter set)/X Days
- Energy cost component
  - Increases over time
  - Non-linear
- Total cost component
  - Decreases then increases over time
  - Change filters at inflection point for best life cycle cost



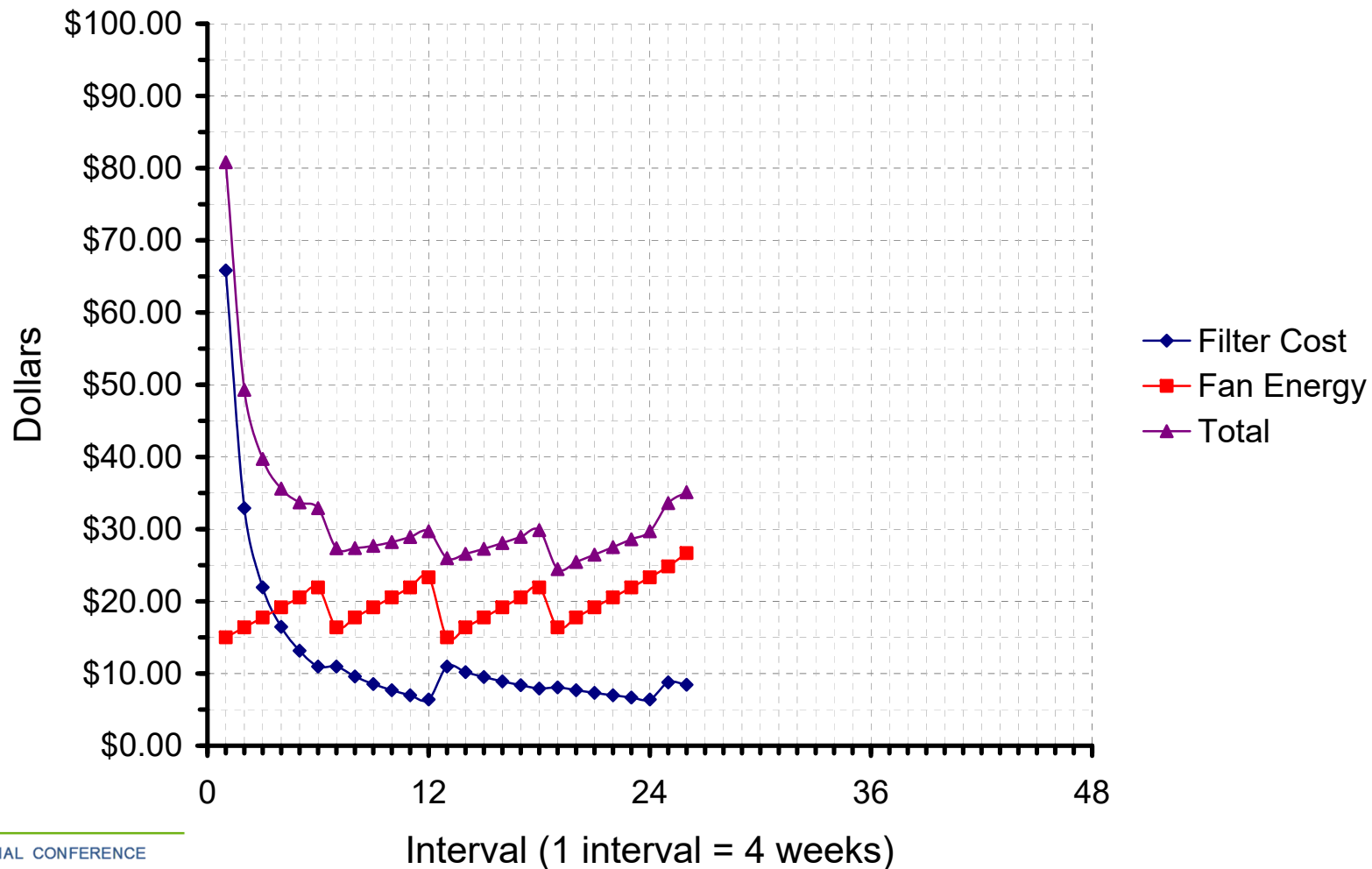
# Do You Know What you Need to Know for Optimum Operation?

- Measure/monitor filter pressure drop
- Measure air flow
- Calculate filter failure power
- Create a graphic that projects what your filter operating costs are based on a manually entered filter cost and alarms when you hit the inflection point



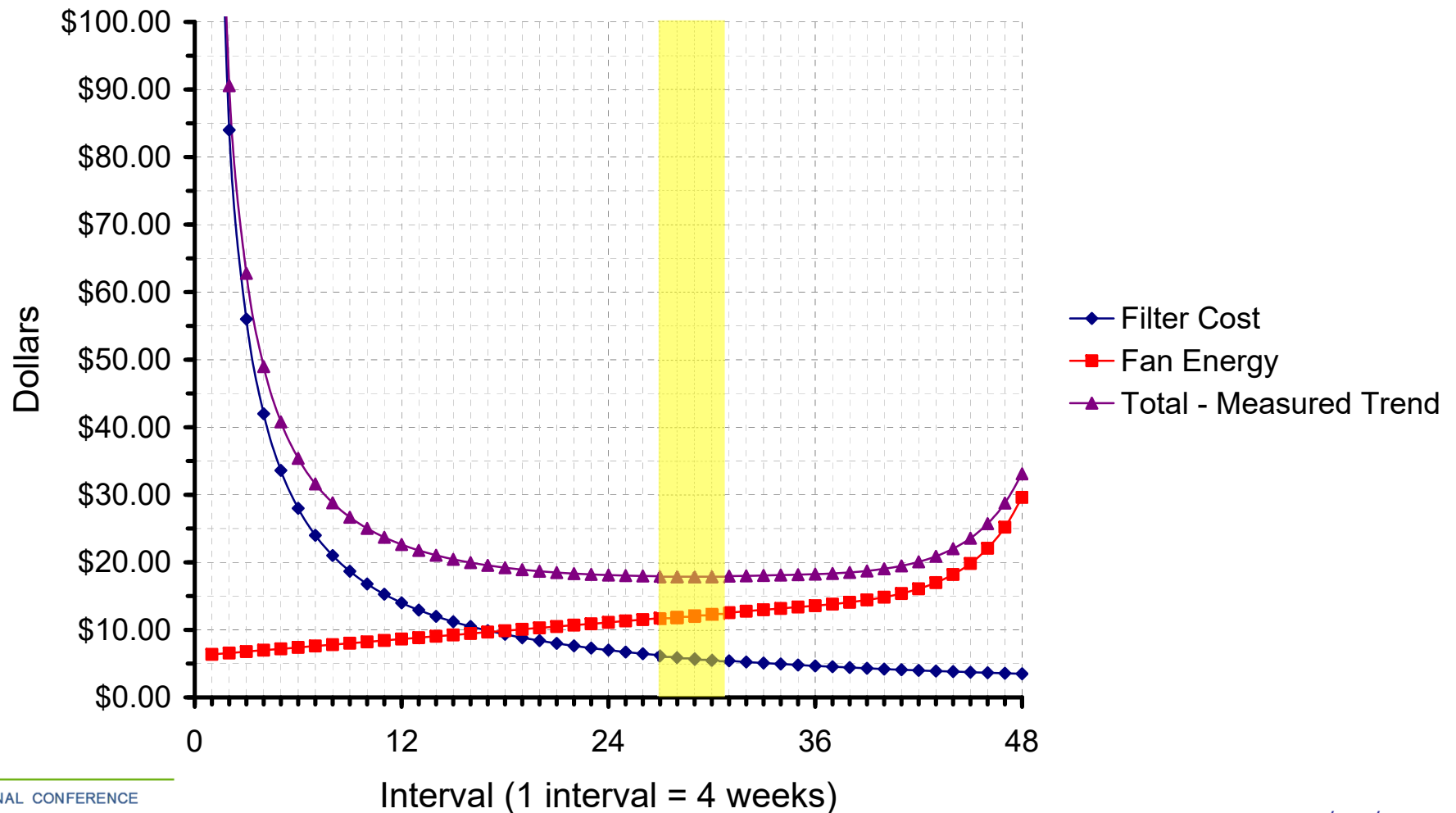
# Why Does This Matter?

Filter Cost per Average Day - AHU C  
Original Filters and Approach to Change-out



# Why Does This Matter?

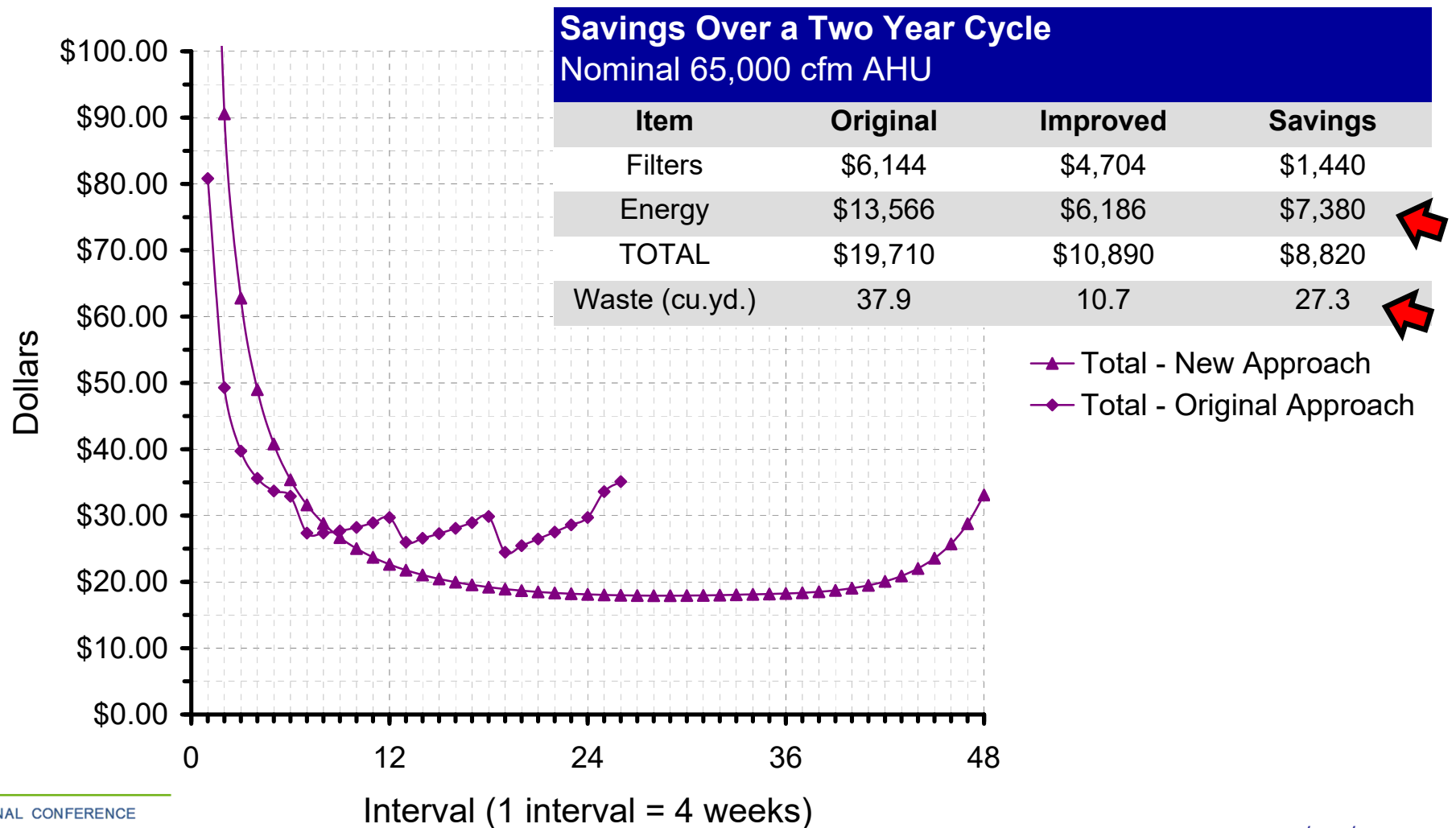
Filter Cost per Average Day - AHU C  
Extended Surface Area Filters



# Why Does This Matter?

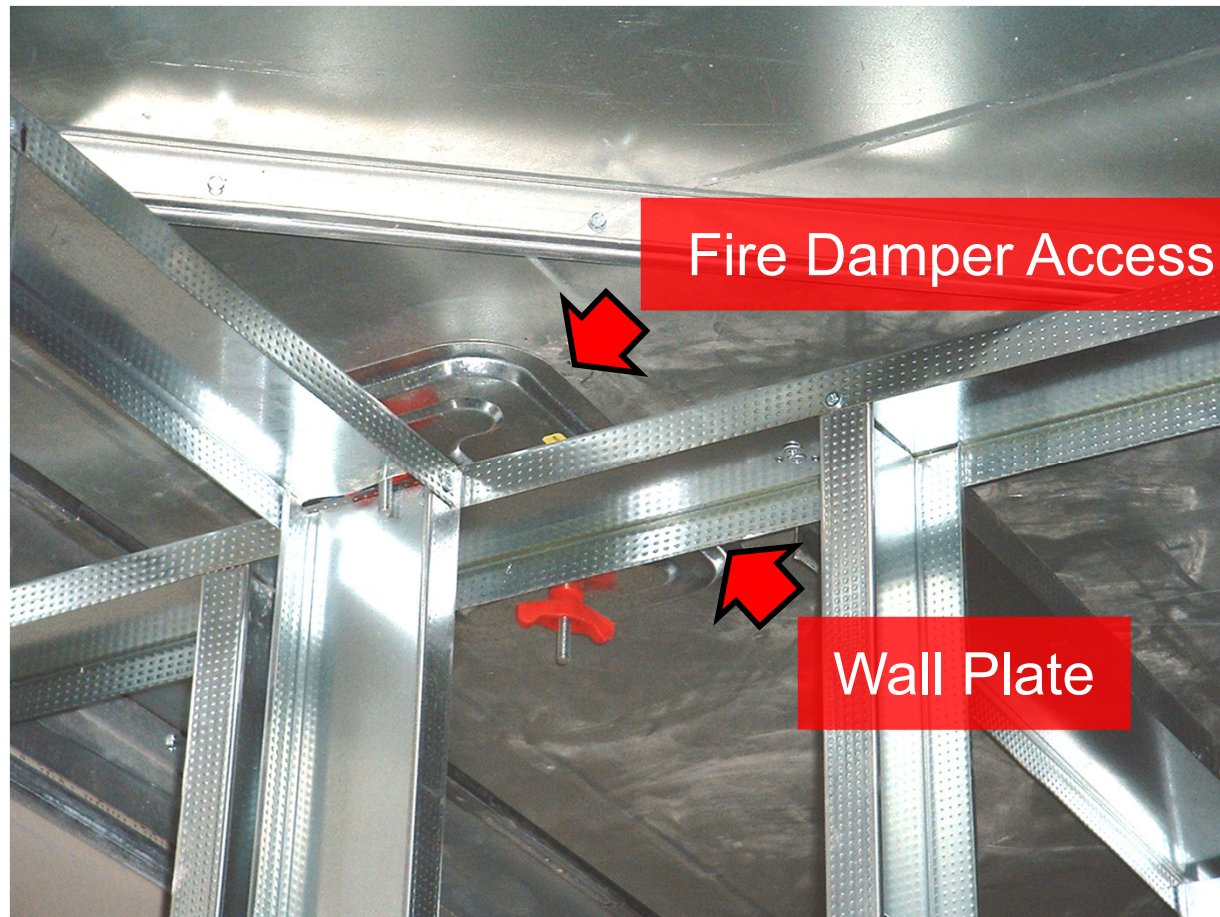
## Filter Cost per Average Day - AHU C

Original Approach vs. New Approach





# Construction Observation



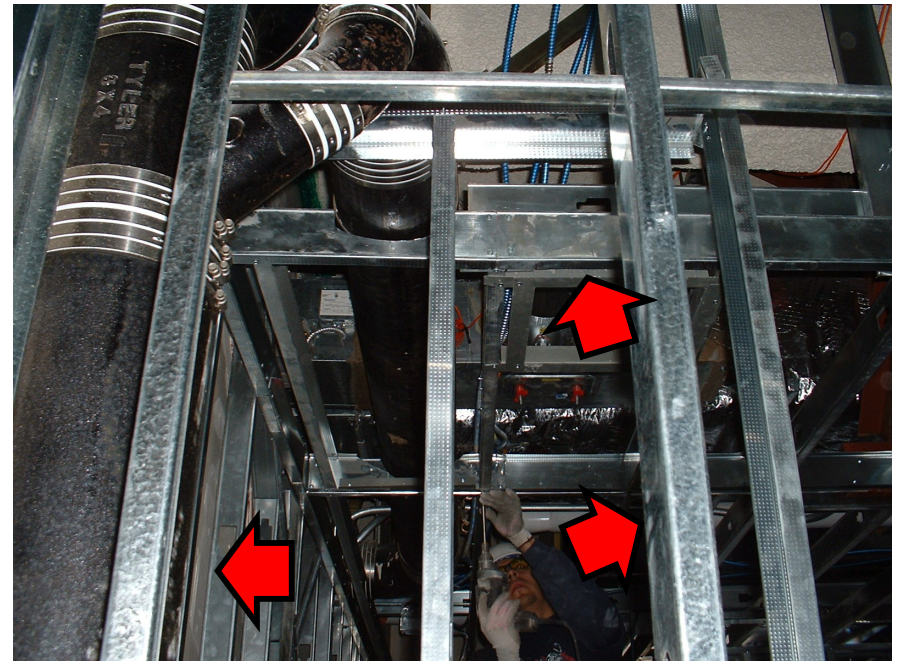
*Because inspecting and servicing a combination fire and smoke damper should not involve removing a wall.*



# An Architect, an Engineer, and a Commissioning Provider Go to Inspect a Fire Damper ....

What the Architect sees:

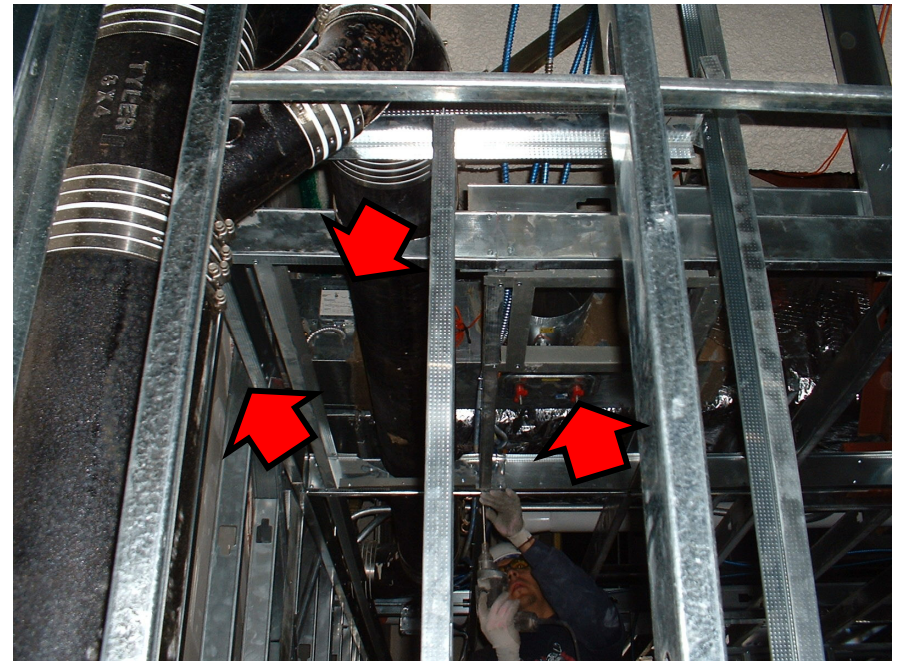
- Properly rated shaft wall
- Properly framed partitions
- Ceiling heights look about right based on the framing



# An Architect, an Engineer, and a Commissioning Provider Go to Inspect a Fire Damper ....

What the Engineer sees:

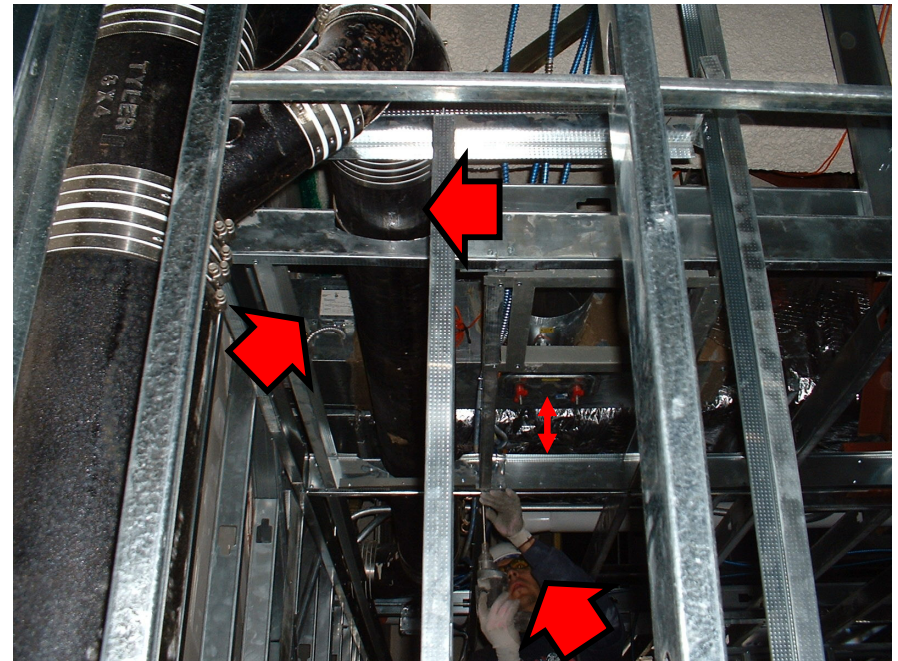
- Fire damper properly installed in the rated partition
- Duct inspection access provided
- Test/manual reset switch provided



# An Architect, an Engineer, and a Commissioning Provider Go to Inspect a Fire Damper ....

What the Commissioning Provider sees:

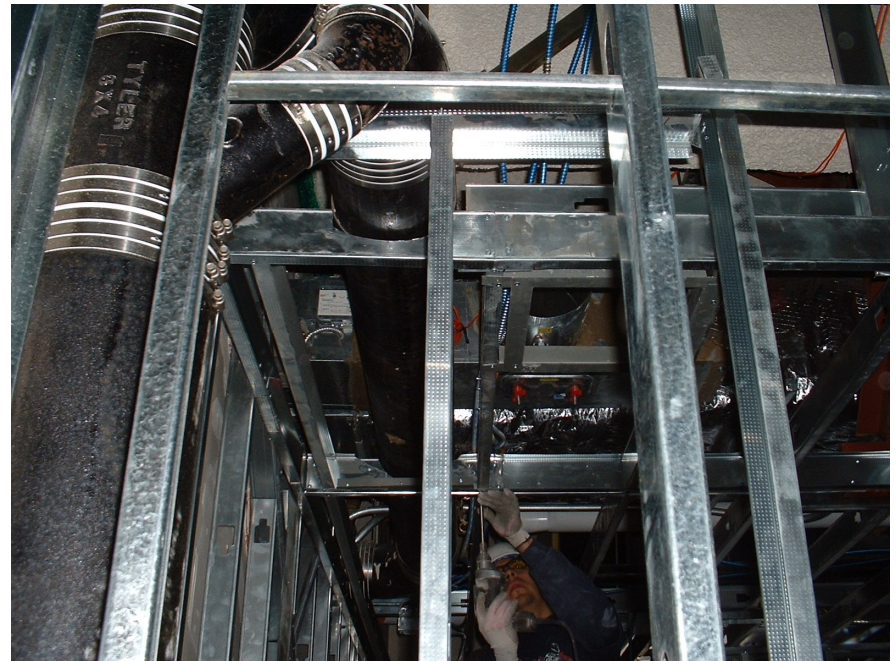
- Ceiling access framing that is several feet from the duct access
- 6" of clearance between the top of the ceiling framing and the bottom of the duct access
- An 8" line blocking what little access there is to the test/manual reset switch





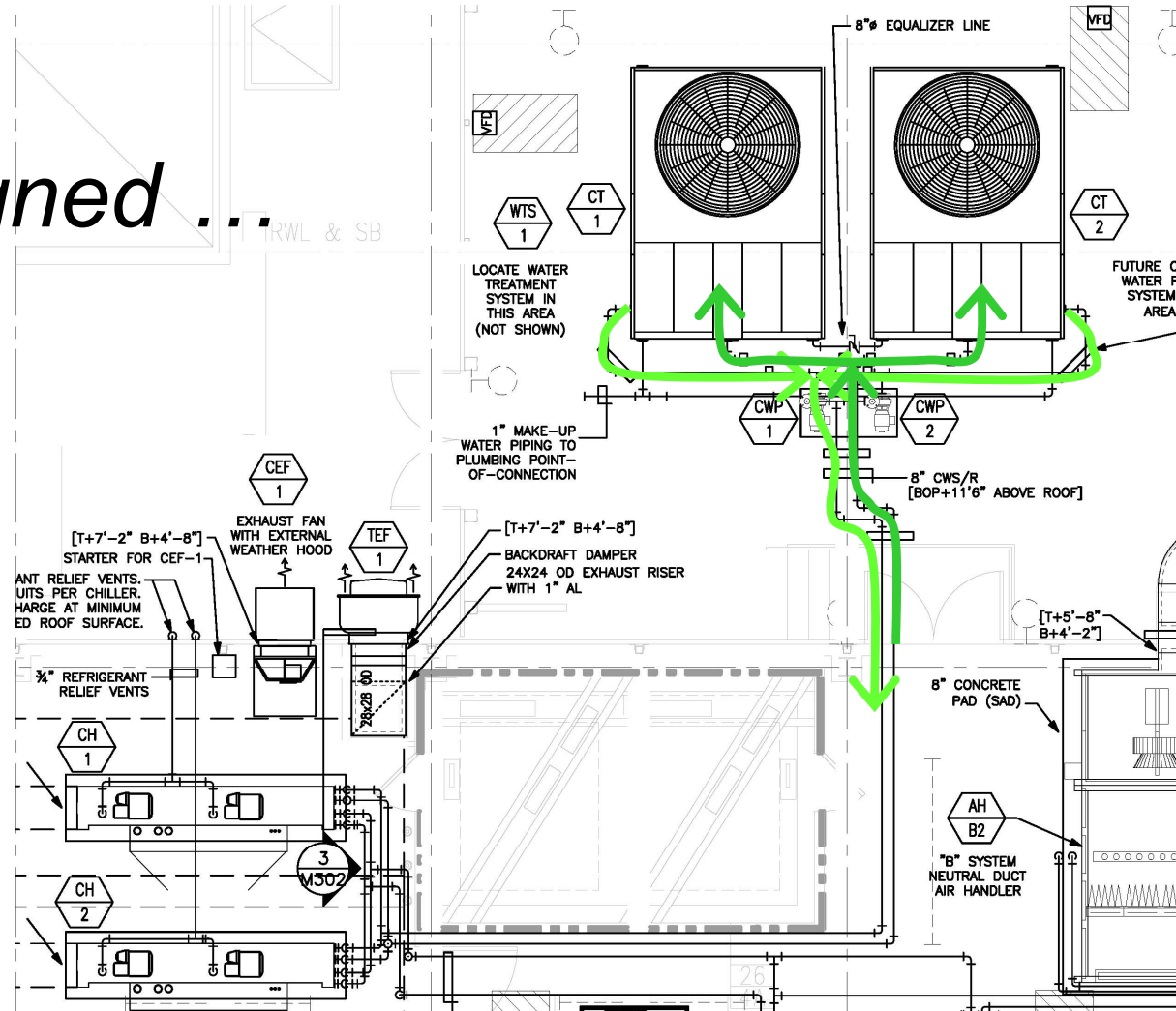
# Everyone Did Their Job

- The Design Professionals verified compliance with critical design requirements
- The Commissioning Provider brought additional valuable perspectives
  - Integration
  - Operations and maintenance
- Some of these things are not apparent until you get into construction



# Capturing Design Intent via Construction Observation

*As designed . . .*



# As Installed



*... not so symmetrical!*



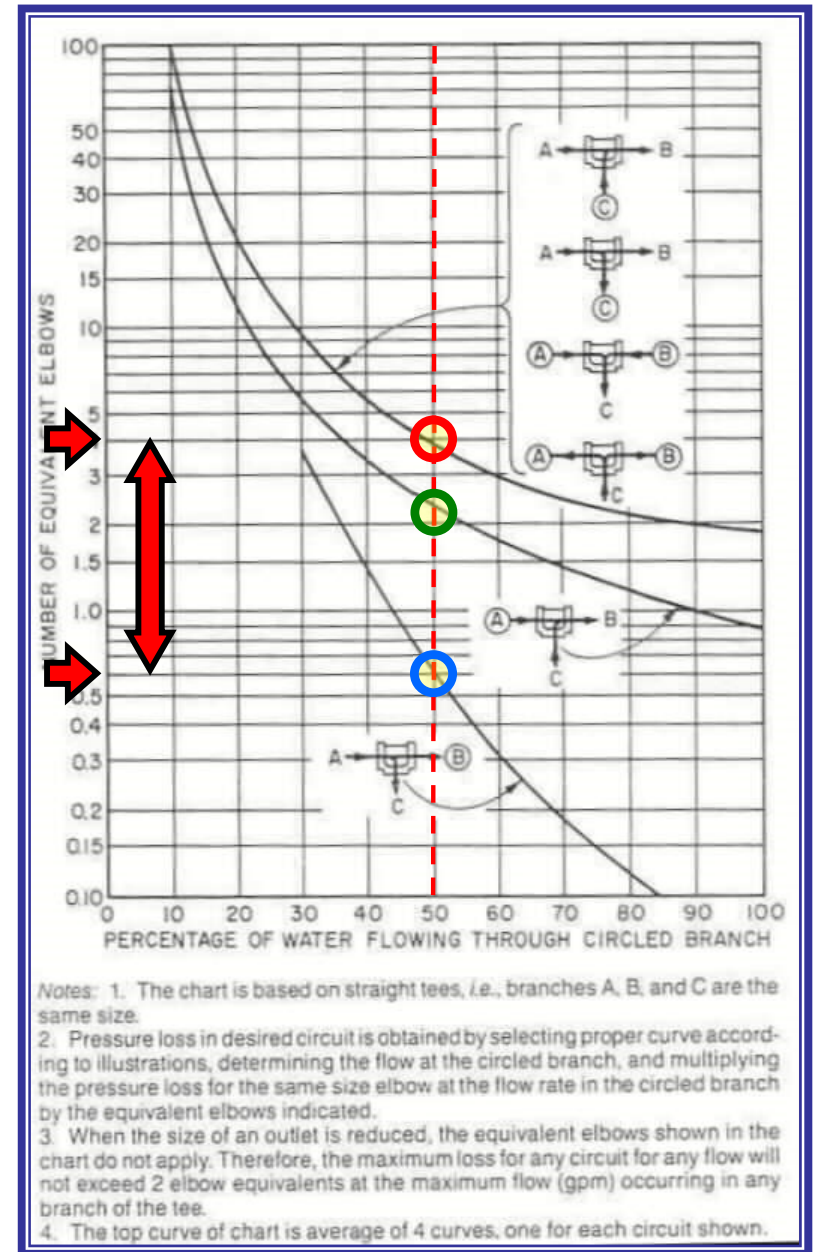


# Tee Details

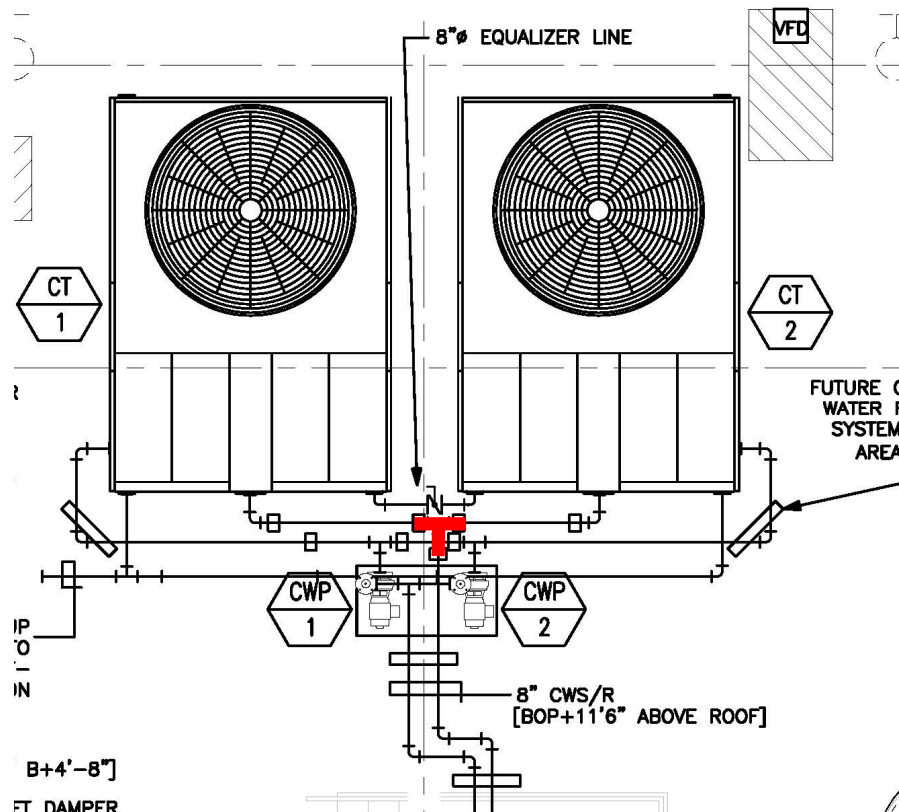
Tee Orientation has a significant impact on pressure drop

Factor of 6.7 Difference!

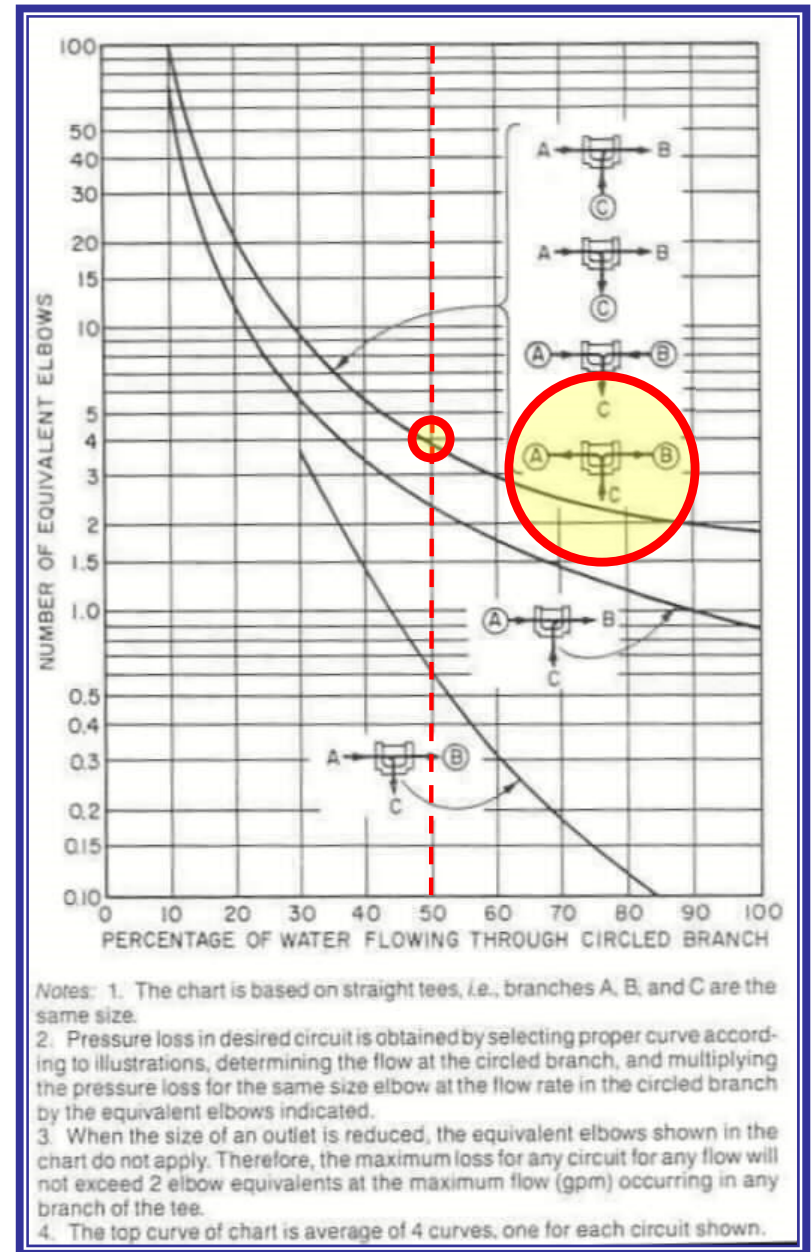
*Image from the ASHRAE Handbook of Fundamentals*



# As Designed

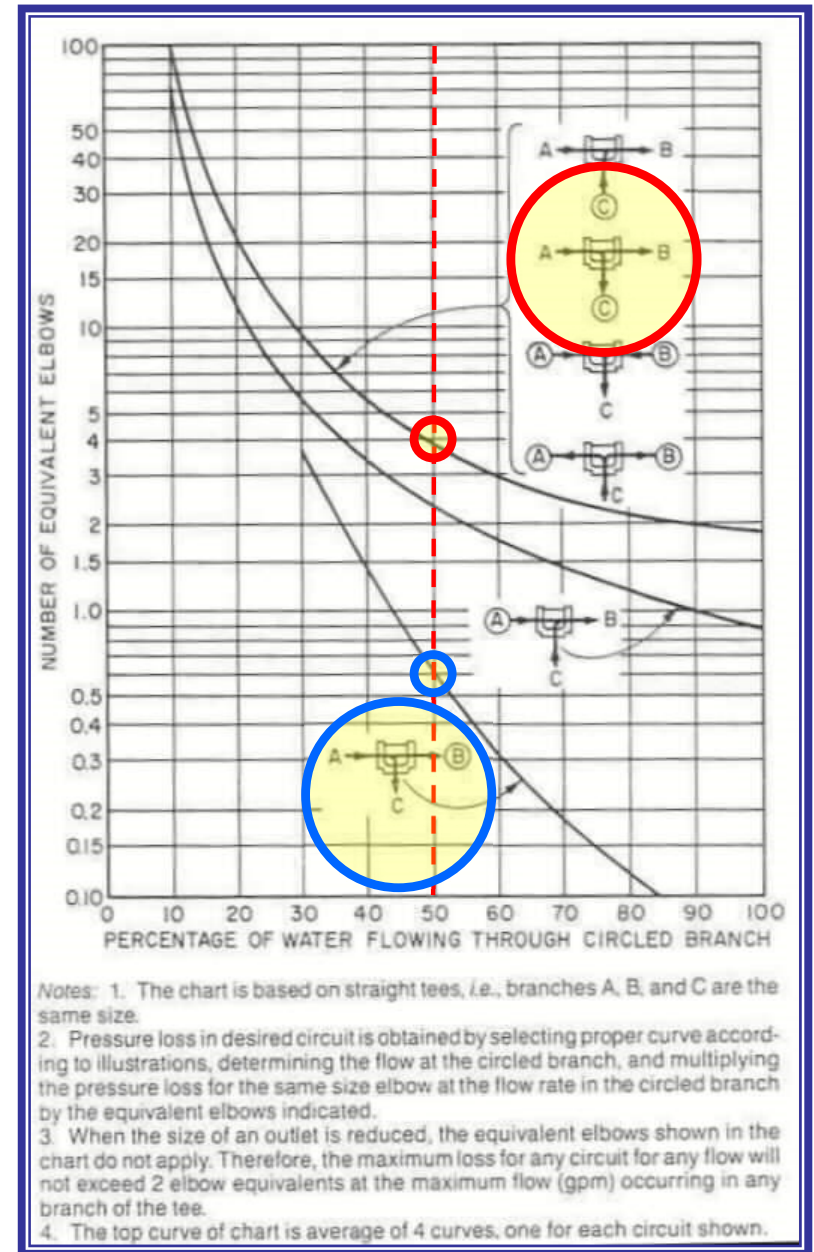


*...higher, but equal pressure drops*



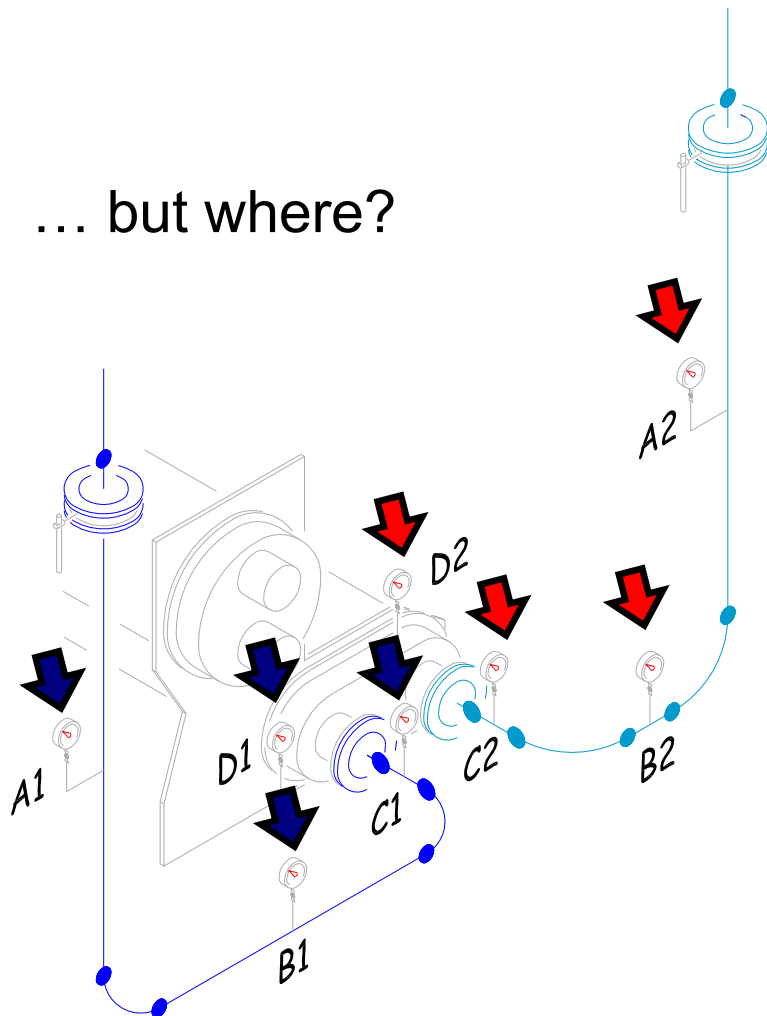


# As Installed

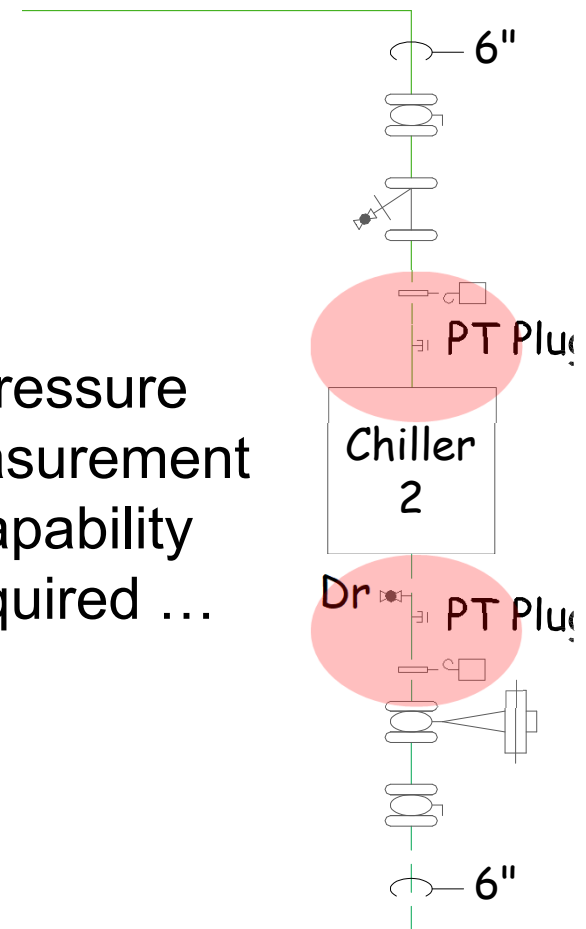


# Another Situation Where Small Details can Make A Big Difference

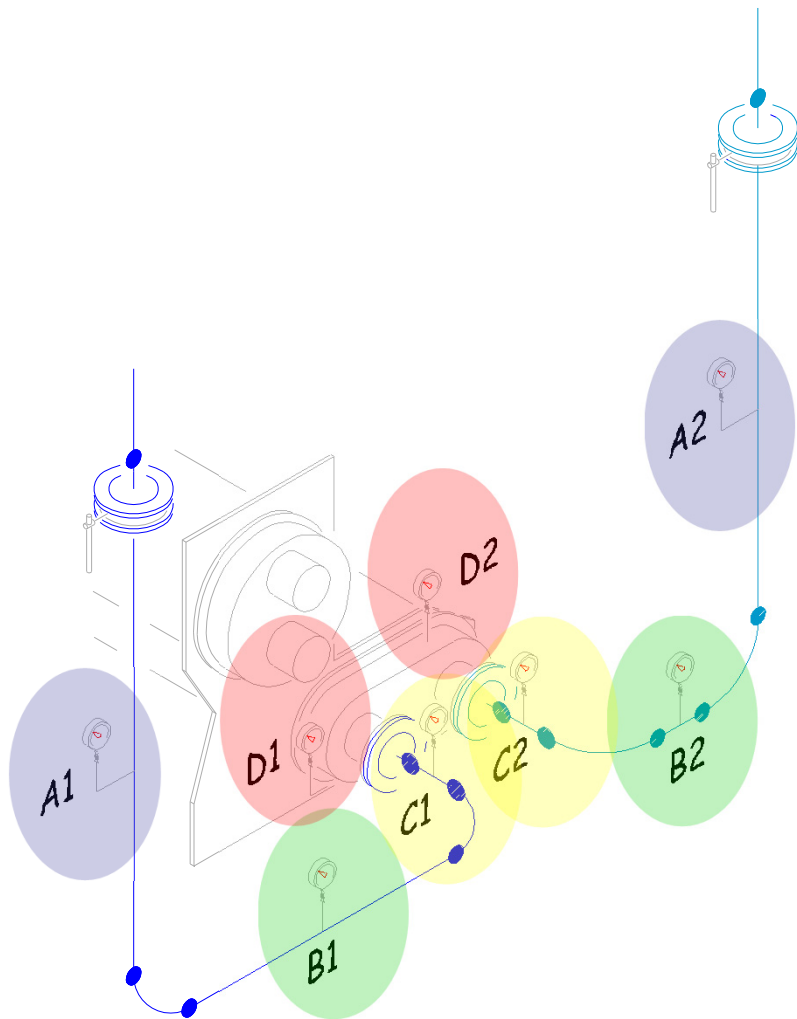
... but where?



Pressure measurement capability required ...



# Another Situation Where Small Details can Make A Big Difference

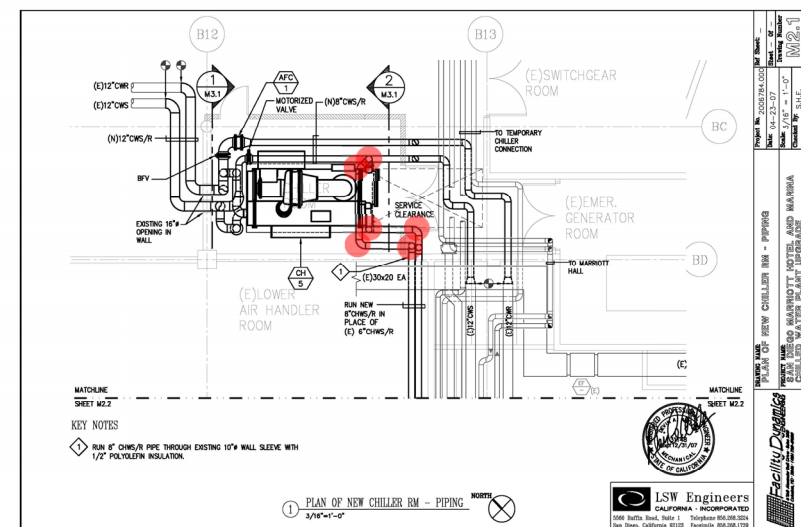
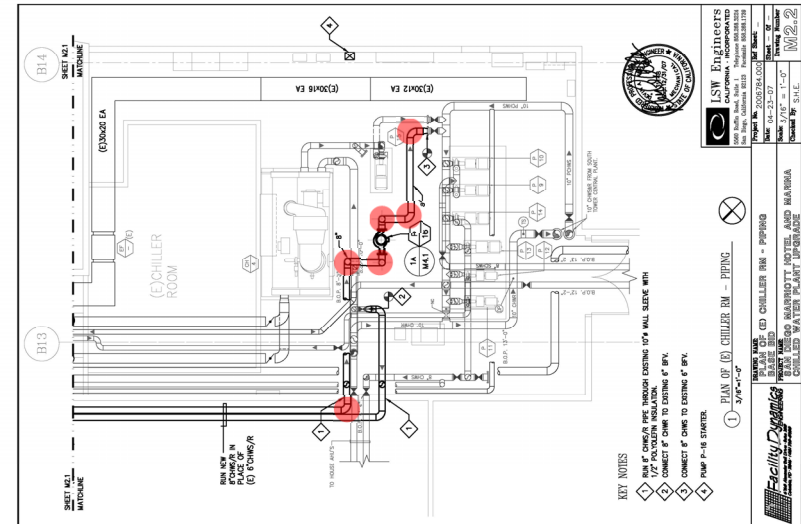


**Flow Based on Differential Pressure**

Location	Pressure Difference, ft.w.c.	Flow Based on Pressure Difference	
		gpm	%
D1 - D2	14.48	1,363	85%
C1 - C2	17.00	1,600	100%
B1 - B2	18.26	1,718	107%
A1 - A2	19.78	1,862	116%

# Short Radius vs. Long Radius Elbows

- Number of elbows = 26
- Difference in head for the circuit with long radius versus short radius elbows
  - 3+ ft.w.c.
  - 9+% difference in pump head (A.K.A. the safety factor)



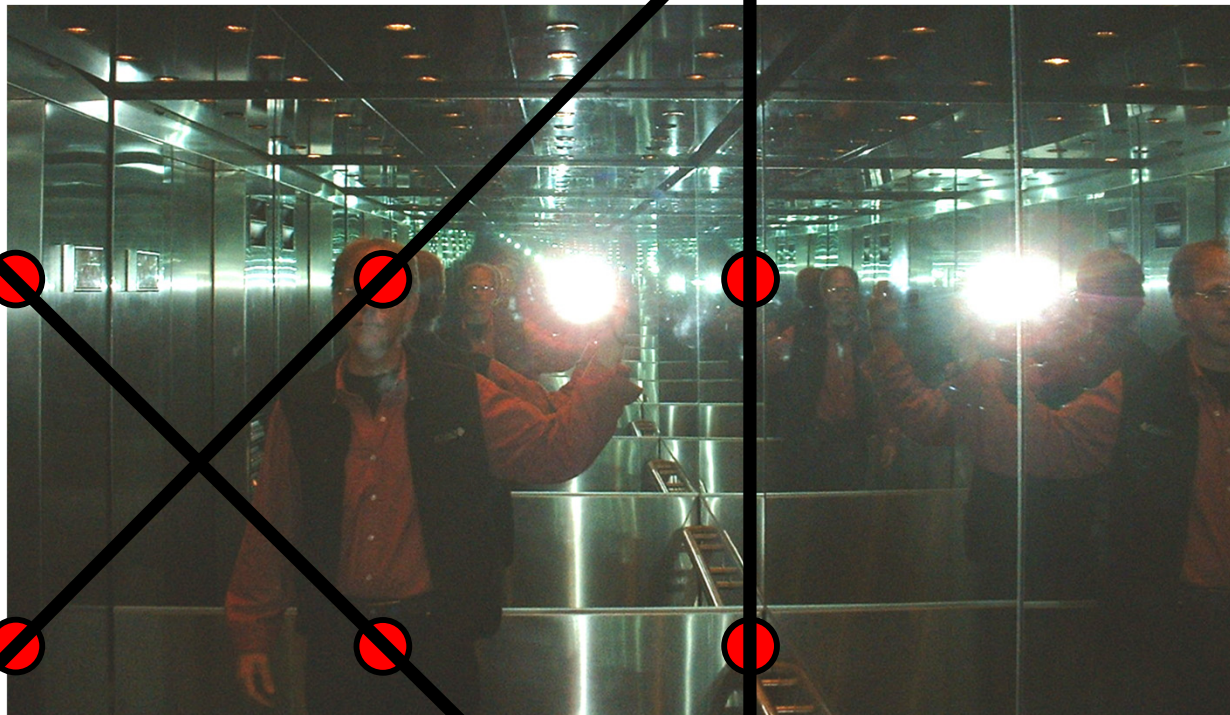
# Bottom Lines

- Design phase is the ideal time to hit critical Cx targets
  - Plan and set up the process
  - Capture design intent
  - Optimize efficiency
  - Ensure access and other O&M considerations
- Unresolved design opportunities become field opportunities
- Getting it right on paper does not necessarily mean it will be right in the field

# Persistence



## Resources



*... and you'll discover a key source of innovation and persistence!*

# Integrated Design and Design Review Resources

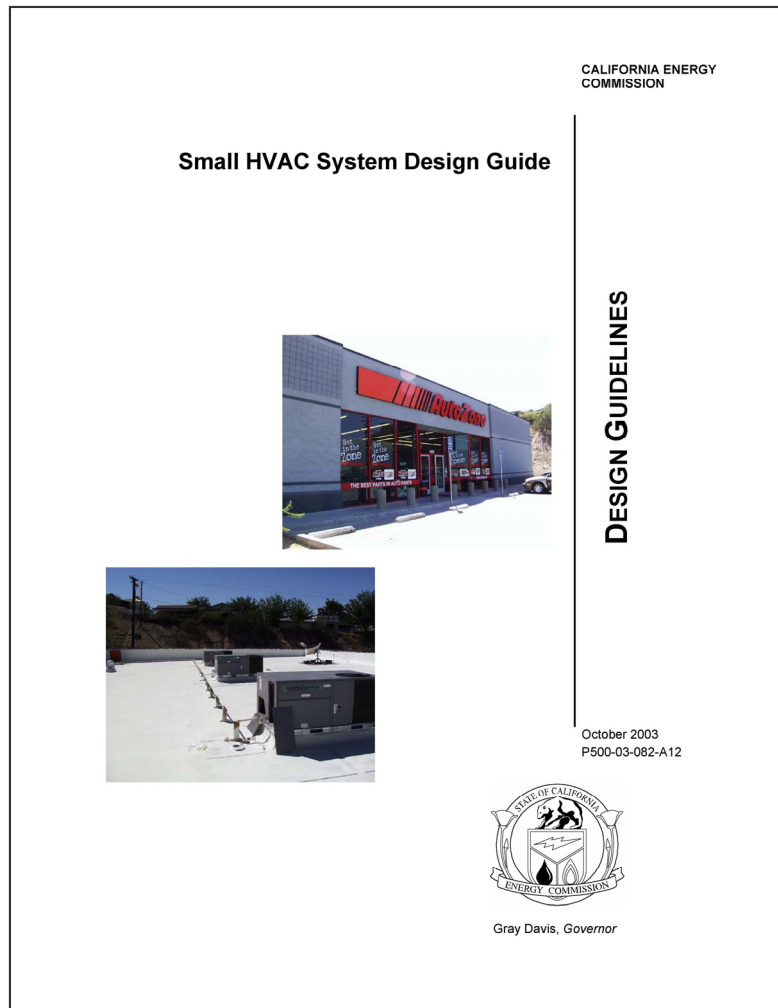
## Energy Design Resources Design Briefs

- *Design Details*
- *Design Review*
- *Field Review*
- *Improving Mechanical System Energy Efficiency Through Architect and Engineer Coordination*



Free Downloads from [www.energydesignresources.com](http://www.energydesignresources.com)

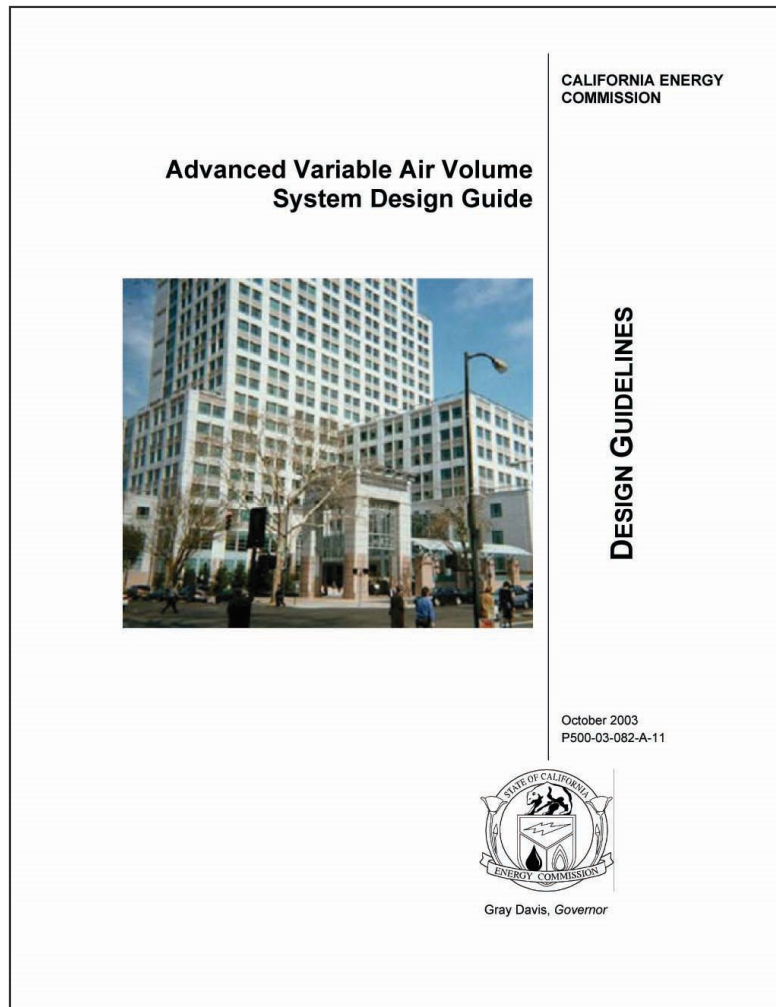
# Small Commercial HVAC Design Guideline



- Good design paves the way for successful Cx
- Design review is an essential component of Cx
- Topics include
  - Integrated design
  - Unit sizing
  - Unit selection
  - Distribution systems
  - Ventilation
  - Thermostats and controls
  - Commissioning
  - Operations and maintenance

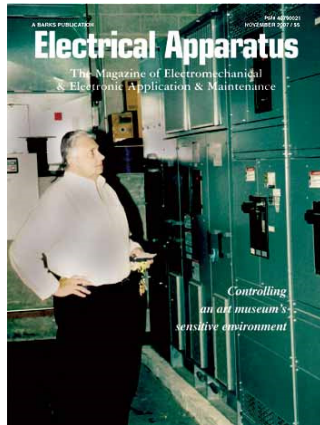


# Advanced VAV Design Guideline



- Good design paves the way for successful Cx
- Design review is an essential component of Cx
- Topics include
  - Early design issues
  - Zone issues
  - VAV box selection
  - Duct design
  - Supply air temperature control
  - Fan size, type and control
  - Coils and filters
  - OA, RA, relief and exhaust control

# Take Advantage of Free Trade Publications



- Consulting Specifying Engineer

[www.csemag.com](http://www.csemag.com)

- Heating, Piping, and Air Conditioning

[www.hpac.com](http://www.hpac.com)

- Engineered Systems

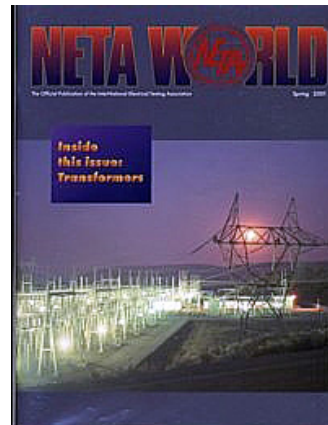
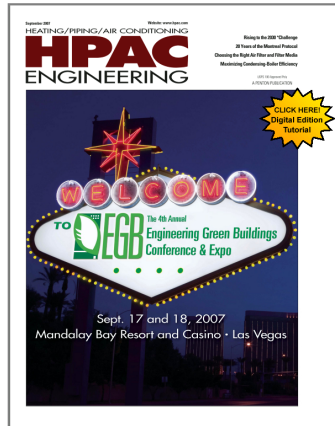
[www.esmagazine.com](http://www.esmagazine.com)

- Electrical Apparatus

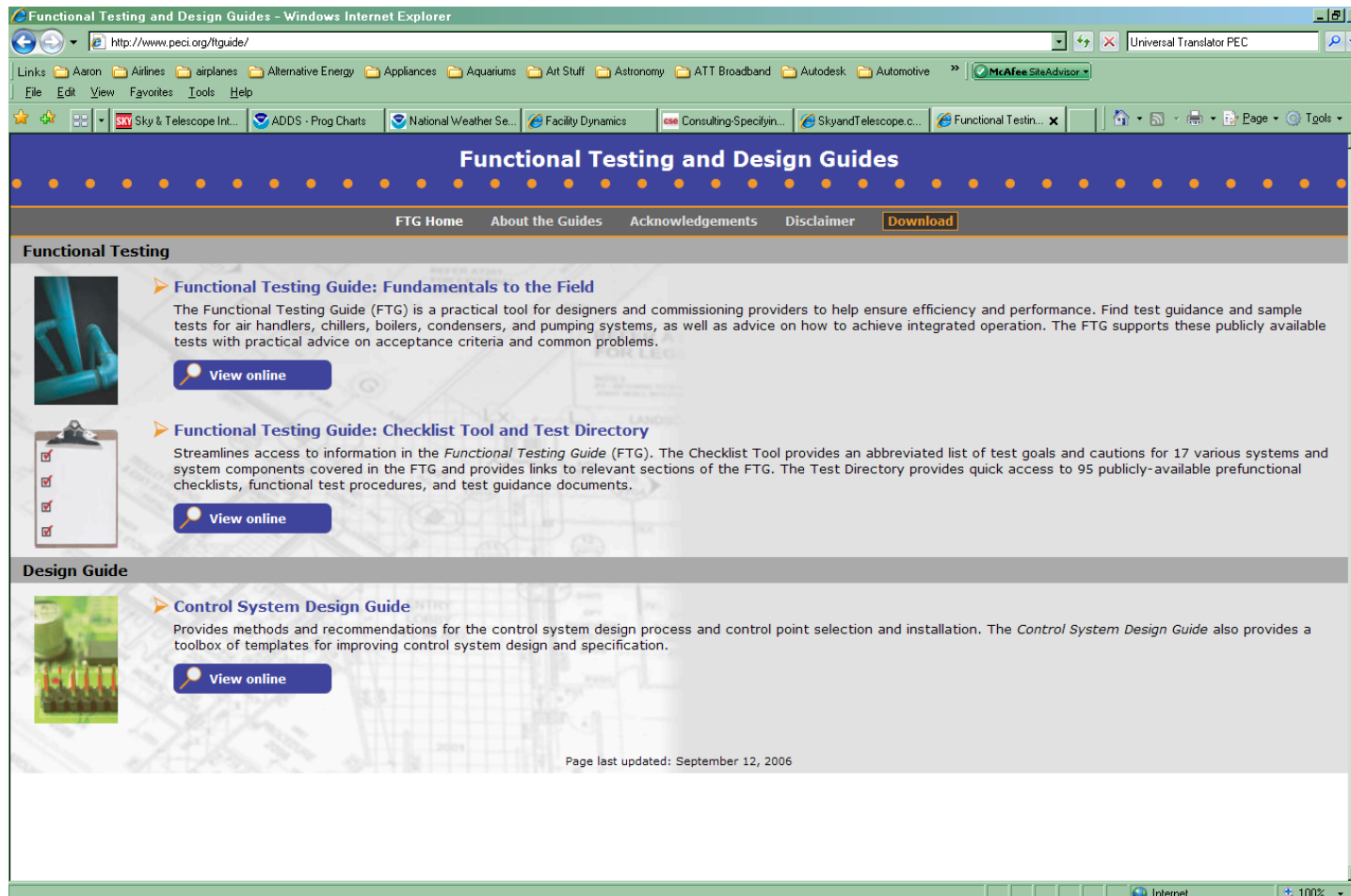
<http://www.barks.com/index.htm>

- NETA World

<http://www.netaworld.org/>



# The Functional Testing Guide Checklist Tool



The Functional Testing and Design Guide; [www.peci.org/ftguide](http://www.peci.org/ftguide)

# EDR's Design Review Checklist Tool



**EnergyDesignResources.Com**

**Cx Assistant™**

**Design Review Tool Module  
Master Reference Guide**

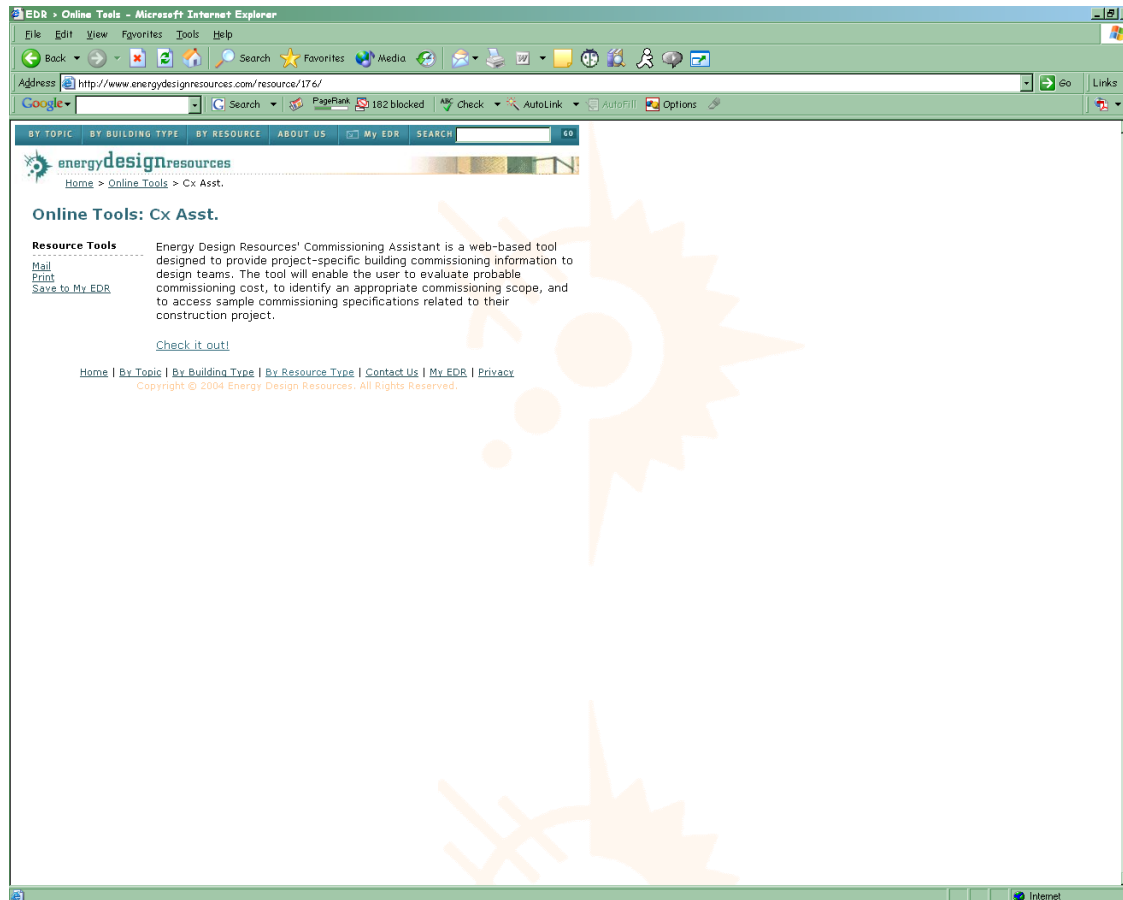
Developed by  
**Portland Energy Conservation, Inc.**

Under contract to  
**Pacific Gas and Electric Company**

**March 2007**

<http://resources.cacx.org/library/>,  
Search for Design Review Tool

# The Commissioning Assistant



- Develop design intent
- Evaluate probable commissioning cost
- Identify an appropriate commissioning scope
- Access sample commissioning specifications

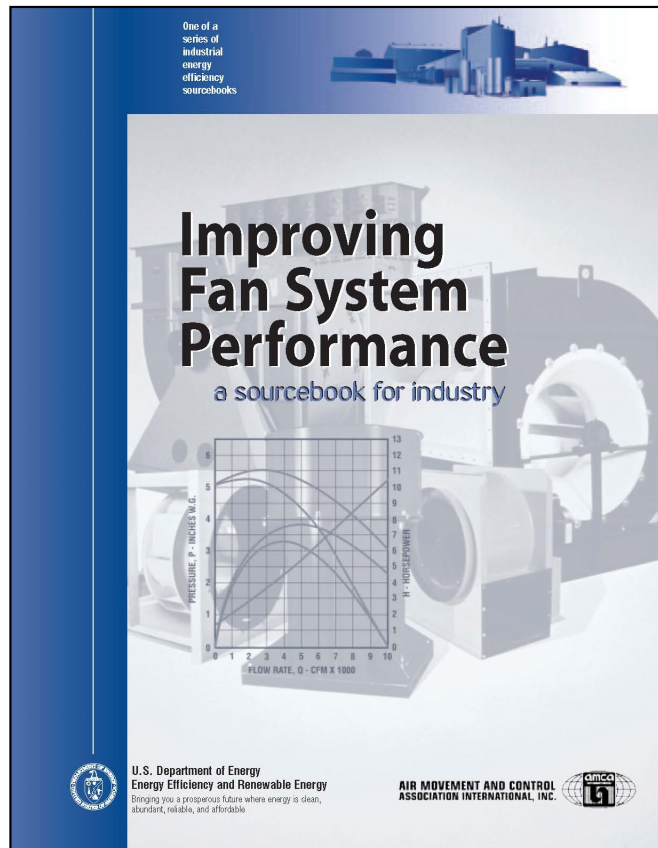


# Watch for Web Based Training Opportunities



<http://207.67.203.54/p40007staff/opac/index.asp>

# Want To Learn More?



Make use of publicly available resources:

- The DOE/AMCA sourcebook *Improving Fan System Performance* can be downloaded at <http://industrial-energy.lbl.gov/node/297>

# Motor Selection and Comparison Tool

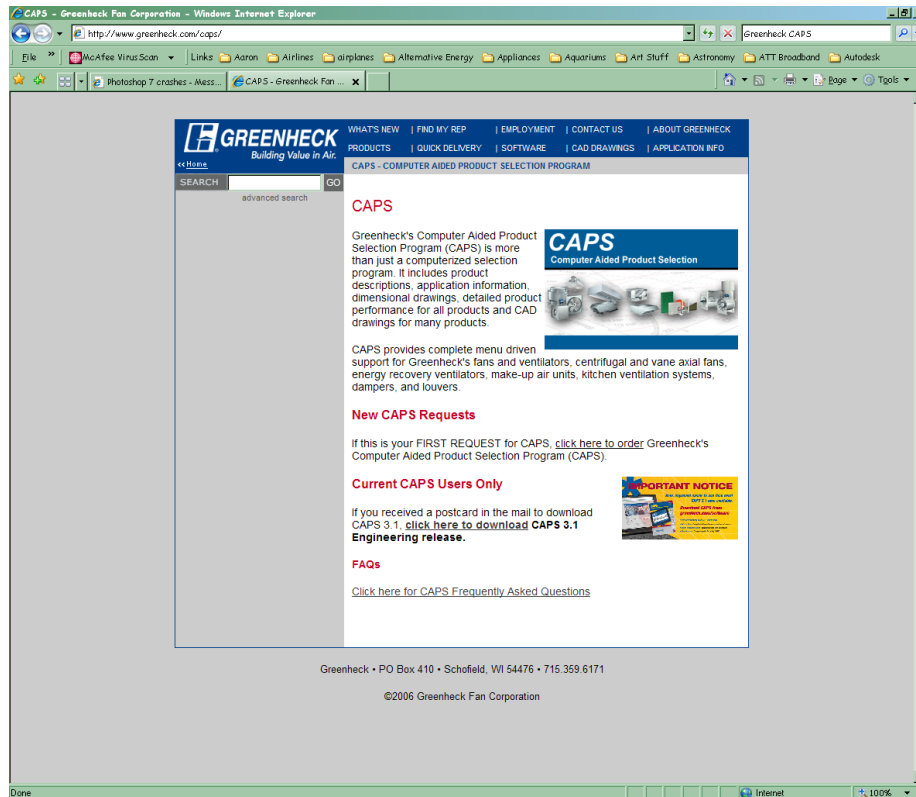


Compare motor purchase and replacement options

[www.eere.energy.gov/industry/bestpractices/software.html](http://www.eere.energy.gov/industry/bestpractices/software.html)



# Vendor Tools Provide Field Insights



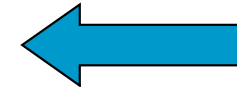
Design and Operations;  
Same idea, different direction

What should  
it do?



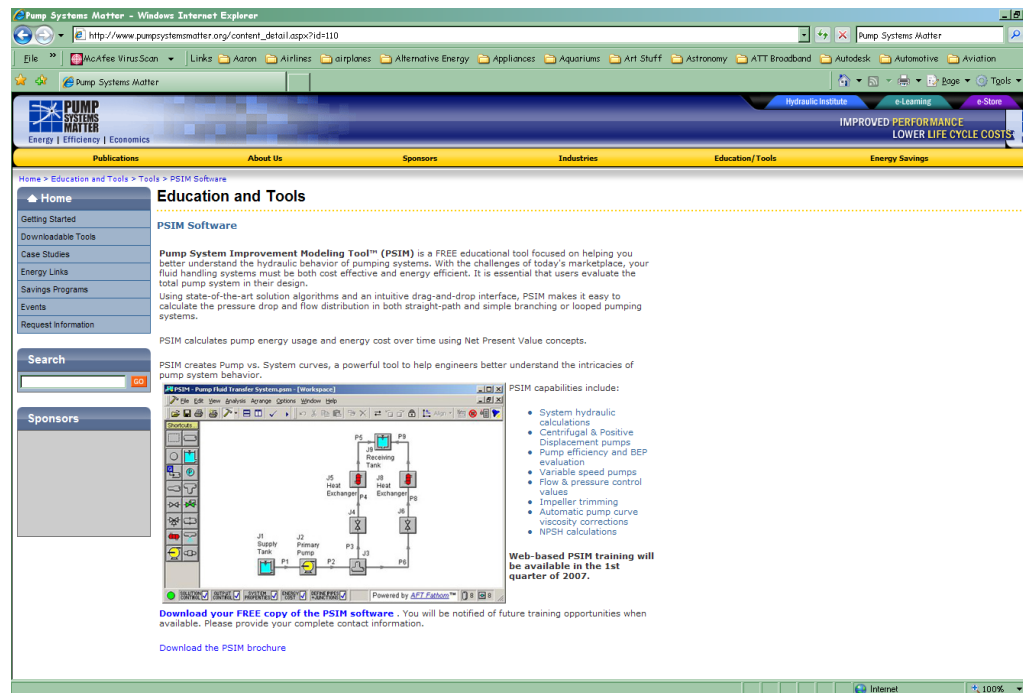
What does it  
look like?

What should  
it do?



What does it  
look like?

# Use PSIMT to Assess Pump Head



*This tool is a free download at the Pump Systems Matter website at:*

[www.pumpsystemsmatter.org](http://www.pumpsystemsmatter.org)

# Another Free Pump Performance Resource

