

How Water and Steam Systems; Basic Principles, Ongoing Commissioning, Operation, and Optimization

#### How We Generate Heat in Buildings



Presented By: David Sellers Senior Engineer, Facility Dynamics Engineering

#### Your Basic Boiler Plant

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# Your Basic Boiler

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#### https://tinyurl.com/SaturatedSystemsP4PotBoils

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#### A Modern Boiler Plant

A Really Big Modern Boiler Plant

### The Steam Drum

### The Combustion Air Fan

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### Combustion Air and Fuel Control Linkages

#### Adjustable Cam and Follower for the Modulating Gas Control Valve

#### Flue Gas Energy Recovery System



#### PanelView Plus 1000



#### **The Process Control Graphic**





The Heat Recovery Pumps

#### The Heat Recovery Heat Exchangers



### The Flue Gas Condensate Pumps



°CTU-2°

PUMP #2

PUMP #1

#### They go hand in hand

- Condense a pound of steam, gain a pound of condensate
  - Condensate needs to return to the plant
    - Gravity and pipe pitch
    - Pumps







### The Combustion Process

#### They go hand in hand

 Condense a pound of steam, gain a bunch of energy

## $CH_4 + 2O_2 \rightarrow CO_2 + 2H_2O + Heat$

Since air is more than just Oxygen, what happens to the other stuff?

#### They go hand in hand

 Condense a pound of steam, gain a bunch of energy

## $CH_4 + 2O_2 \rightarrow CO_2 + 2H_2O + Heat$

What would happen to boiler efficiency if we condensed this water vapor?

#### They go hand in hand

 Condense a pound of steam, gain a bunch of energy

# $CH_4 + 2O_2 \rightarrow CO_2 + 2H_2O + Heat$

How do we get the water vapor to condense?





## **Combustion Efficiency**



#### All Boilers are Condensing Boilers

... at some point in their operating cycle

#### Not All Boilers are Designed to Operate Continuously In Condensing Mode

#### A Boiler Designed to Not Operating in Condensing Mode Courtesy Christian Weber





### An Example

#### **Applying These Concepts**



#### General Note:

This drawing has been developed using a combination of field notes and project documents. It has not been fully field verified as of 2015-11-20 although the general arrangement and order of connection is based on field notes and sketches.

Columbus Hotel

Heating Hot Water System Diagram 2015-11-19 DS

• 22 Floors, 408 Guest Rooms, 485,000 sq.ft.



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 22<sup>nd</sup> Floor – Engineering Office and Major Mechanical Space



• Floor plan Expands at Level 3



 Note that parallax/perspective/vanishing point makes the top of tower look wider than the bottom



• Three Basement Levels with Parking, Laundry, Lockers, and other "Back of the House" Areas (Level 3 Shown)



• Three Basement Levels with Parking, Laundry, Lockers, and other "Back of the House" Areas (Level 1 Shown)



 1st Floor; Main Lobby, Food and Beverage, Management, Loading Dock



 2<sup>nd</sup> Floor; Ballrooms, Pre-function, Meeting Rooms, Kitchen, Storage



 3<sup>rd</sup> Floor: Meeting Rooms, Pre-function, Restrooms, Mechanical Space, Ball Room Upper Level, DHW Heaters


#### **General Building Arrangement**

• Guest rooms Start on the 4<sup>th</sup> Floor



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• Guest rooms Start on the 4<sup>th</sup> Floor



### Thinking Through a Condensing Boiler Application

- Existing boiler provide 180°F water running with 550-560°F flue gas temperatures
- Combustion air is entering the burner at 70-80°F
- Annual gas consumption 2014 basis 365,139 therms
- Current annual gas cost 2014 basis \$213,452 per year
- Existing boilers are about to be replaced with condensing boilers
- An EBCx project is about to take place in the facility

1. What are some of the opportunities?

#### **New Boiler Efficiency Curves**



### **New Boiler Efficiency Curves**

#### lf:

- Current boiler efficiency is in the 75% range
- Current gas cost is in the range of \$213,452 per year

Can we project a conservative estimate of the savings to be achieved?



#### Taking a Closer Look at the System



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Columbus Hotel

Heating Hot Water System Diagram 2015-11-19 DS



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#### Is There a Constraint on the Temperature We Can Operate the New Boilers At?



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Columbus Hotel Heating Hot Water System Diagram

2015-11-19 DS

# Was the 20% Premium for Condensing Boilers Worth It?



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Columbus Hotel

Heating Hot Water System Diagram 2015-11-19 DS

### **Another Potential Opportunity**





Boiler Flue Temperature vs. Time Friday, November 27, 2015





Burner Fan On, Purge, Airflow Ramping Up, 30 sec. +/-; This is the of the purgey cycle which blows air through the boiler to remove unburned gas in the combustion chamber due to gas valves leaks, thus prevention an explosion when the pilot is igniated. Purge, Full Airflow 60 sec.: This is the main portion of the purge cycle with the combustion air fan running at high fire air flow. Purge, Airflow Ramping Down, 30 sec.+/-: The end of the purge cycle; sets up the ignition cycle at the low fire airflow rate. Ignite and Prove Pilot, 10 sec,: The pilot gas valve opens and an ignites the pilot flame. An electronic sensor verifies ignition before allowing the cycle to proceed.

**Ignite and Prove Main Flame**, **15 sec**: The main gas valve is opened and the burner operates on low fire ; main flame is verified. Firing, Modulating to Match the Load, time varies with load: This is the heart of the firing cycle and the burner modulates between low and high fire as necessary to match the load.

Flame Off: Post Cycle Purge, 14 sec. +/-: The fuel cycles off because at low fire, the burner capacity exceeds the current load; the burner fan remains in operation to purge any unburned gas from the combustion chamber.

**Burner Fan Off, Boiler Off Cycle, time varies with load:** The boiler stands by until the load drops the entering water temperature below set point. Then, the cycle repeats.

#### Potential Savings via Optimizing Boiler Staging to Minimize Short Cycling and Purge Requirements

Maximum Short Cycle Rate (Note 1)	Lead Boiler Continuous Firing Temperature,	% Excess Air, (Note 3)	Boiler Efficiency, % (Note 3)	Annual Purge Cost (Note 4)			Potential Savings (Note 5)	
	°F (Note 2)			Therms		% of Total		Dollars
7	52	9.5%	78.9%	2,929	\$1,712	0.80%	1,414	\$827
7	52	9.5%	72.6%	3,181	\$1,859	0.87%	1,536	\$898
14	52	9.5%	78.9%	3,801	\$2,222	1.04%	1,850	\$1,082
14	52	9.5%	72.6%	4,127	\$2,413	1.13%	2,009	\$1,175
7	52	81.6%	78.9%	4,857	\$2,840	1.33%	2,346	\$1,371
7	52	81.6%	72.6%	5,275	\$3,083	1.44%	2,547	\$1,489
14	52	81.6%	78.9%	6,303	\$3,685	1.73%	3,069	\$1,794
14	52	81.6%	72.6%	6,845	\$4,001	1.87%	3,332	\$1,948

Notes

- 1. This is how many times the boilers cycle during warm weather. The higher number is based on assuming both the East and West boiler cycle at the same rate as the East boiler does currently. The low rate is an arbitrary reduction to 50% of this.
- 2. This is the temperature below which, with the current load profile, it is assumed the lead boiler will fire continuously. It is based on a curve fit extrapolated from the logger data, which included days at this temperature.
- 3. This is based on a combustion efficiency assessment that was done using the DOE Steam Boiler Tip Sheet #4 and the observed flue gas temperature.
- 4. Percent of total refers to the percentage of the total boiler gas use that is represented by the purge losses.
- 5. This is based on adjusting the staging of the boilers to eliminate short cycling of the lag boiler under conditions when the lead boiler should be able to carry the load by itself (temperatures above the Lead Boiler Continuous Firing Temperature).

A detailed explanation can be found here: <u>https://tinyurl.com/PurgeLosses</u>







# Steam vs. Hot Water

## Hot Water Boilers

and

#### **Steam Boilers**



- Similar in form
- Similar safeties
- Different controlled variable
  - Hot water fired for a temperature set point
  - Steam fired for a pressure set point



• Different operating limitations

#### Steam Systems

http://www.armstronginternational.com/files/products/traps/pdf/P-101.pdf



Screen shot of Figure CG-6 from P-101

© Armstrong International

### Hot Water Systems are Basically Pumping Systems

#### energy design resources design hri CENTRIFUGAL PUMP APPLICATION AND OPTIMIZATION energy designresources design brief Who Should Read This Brief? This brief presents practical pump theory, principles. The concepts discussed will be o Design professionals who want to bette theory and design practice apply to day environment Summarv Identifying and solving pump Commissioning providers who wish to system problems commonly This brief explores practical and proven troubleshooting and assessment pumping systems, and the evolution of techniques for identifying and solving pump system problems commonly encountered in existing systems encountered in existing buildings during retro-commissioning processes. buildings will improve Facilities engineering professionals who It focuses on pump and system interaction, with a primary focus on the performance and translate parallel pump arrangements that are common in heating, ventilation, the fundamental design and application and air-conditioning (HVAC) applications. into energy savings. equipment they operate A case study format is presented for readers. Each example is presented Key points for each of these areas of profess in the following context: in checklists at the end of each major topic. What were the indicators of the problem and how can the problem be corrected? Are there any "ripple" effects associated with the problem? Be How can the costs and benefits be assessed and can persistence be ensured? How can the problem be prevented in future projects? CONTENTS Introduction Introduction Based on a manufacture's survey and as reported by the Pump Matter Pump Tests Initiative, 60 percent of all pumps are improperly applied. Of those, 90 percent are not specified for the proper operating point. Parallel Pumps Troubleshooting, assessing, and solving pump system problems will

Design professionals, commissioning agents, and facilities engineering professionals will benefit from the techniques and examples presented in this brief. The information provides a fundamental understanding of

improve performance and energy efficiency. Reliability and

redundancy issues also may be addressed.

Dissimilar Pumps in Parallel

Conclusion

Notes

For More Information

Other Parallel Pump Examples

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Pumps; Design, Performance, and Commissioning Issues

**Chilled and Condenser Water** Systems: Design, Performance, and Commissioning Issues

- **EDR Design Briefs**
- Centrifugal Pump Application and Optimization
- Pump System Troubleshooting

#### www.EnergyDesignResources.com

#### Transferring 10,000,000 Btu/hr with Hot Water

Given:

- The load
- A design supply temperature of 180°F
- A design temperature drop of 20°F

Determine

- Required flow rate
- Line size
- Insulation thickness

#### The Waterside Load Equation

$$Q_{Btu/Hr} = 500 \times Flow_{gpm} \times (t_{Entering,°F} - t_{Leaving,°F})$$

Where:

 $Q_{Btu/Hr} = Load in Btu/hr$  500 = Units conversion constant, good for water between 30 and 200°F  $Flow_{gpm} = Flow through the heat exchanger in gallons per minute$   $t_{Entering,°F} = Temperature entering the heat exchanger in °F$  $t_{Leaving,°F} = Temperature leaving the heat exchanger in °F$ 

#### Transferring 10,000,000 Btu/hr with Hot Water

 $Q = 500 \times gpm \times \Delta t$ 

- Q = 10,000,000 Btu/hr
- Supply = 180°F
- Return = 160°F
- $\Delta t = 20^{\circ} F$
- gpm = 1,000
- Line size = 8"
- Number of
   lines = 2
- Insulation = 1-1/2"

#### Transferring 10,000,000 Btu/hr with Steam

**Q** = Pounds per hour x  $\Delta \eta$ 

- Q = 10,000,000 Btu/hr
- $\Delta \eta$  = 1,000 Btu/lb
- Pounds per hour = 10,000
- Supply line size = 6"
- Temperature = 240°F
- Insulation = 2-1/2"
- gpm returned = 20
- Return line size = 1-1/2"
- Temperature = 200°F
- Insulation = 2"

#### Transferring 10,000,000 Btu/hr with Hot Water

 $Q = 500 \times gpm \times \Delta t$ 

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#### Transferring 10,000,000 Btu/hr with Steam

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Processes require steam
Humidification
Cooking
Production
Power Generation

#### http://jensensteamengines.com





### **High Temperatures**

An Issue Common to Both Steam and Hot Water

 Missing insulation = Common Low Hanging Fruit



Energy Savings From Installing Removable Insulated Valve Covers (Btu/hr)								
Temperature,	Valve Size, inches							
°F	3	4	6	8	10	12		
200	800	1,090	1,560	2,200	2,900	3,300		
300	1,710	2,300	3,300	4,800	6,200	7,200		
400	2900	3,800	5,800	8,300	10,800	12,500		
500	4,500	6,200	9,000	13,000	16,900	19,700		
600	6700	9,100	13,300	19,200	25,200	29,300		
Parad on DOE Steam Tip Shart #17								

# **High Temperatures**

An Issue Common to Both Steam and Hot Water

#### Leverage Existing Resources

- US DOE Industrial  $\bullet$ **Technologies** Program
  - Best Practices Web Page
  - Steam Energy System Link
  - www1.eere.energy.gov

#### https://tinyurl.com/InsulationTools





Based on installat on of a 1-inch thick insulating pad on an ANSI 150-pound class flanged valve with an ambien temperature of 65°F and zero wind speed

#### Example

Using the table above, calculate the annual fuel and dollar savings from installing a 1-inch thick insulating pad on an uninsulated 6-inch gate valve in a 250 pounds per square inch gauge (psig) saturated steam line (406°F). Assume continuous operation with natural gas at a boiler efficiency of 80% and a fuel price of \$4.50 per million British thermal units (MMBtu).

U.S. Department of Energy 

from information provided by the

Industrial Energy Extension Service of Georgia Tech and reviewed by the

DOE BestPractices Steam Technica

Subcommittee. For additional information

on steam system efficiency measure

contact the EERE Information Center

at 877-337-3463.

Energy Efficiency and Renewable Energy Bringing you a prosperous future where energy is clean, abundant, reliable, and affordable

Heat Loss pe	r 100 feet of	f Uninsulated	Steam Line	(MMBtu/yr)
Distribution		Diameter	(inches)	
Line Steam	15	150	300	600
1	140	285	375	495
2	235	480	630	840
4	415	850	1,120	1,500
8	740	1,540	2,030	2,725
12	1,055	2,200	2,910	3,920

Based on horizontal steel pipe,  $75^{\rm o}{\rm F}$  ambient air, no wind velocity, and 8,760 operating hr/yr.

Based on DOE Steam Tip Sheet #2

## It Works for Pipes Too

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## Are These Uninsulated Pressure Reducing Valves Opportunities?





# Steam Traps

# Similar Techniques can be Applied to Steam Traps

In

Out



# Similar Techniques can be Applied to Steam Traps

In

Out



Condensate and Traps An Issue Particular to Steam Systems

### A Inverted Bucket Trap
# A Bucket Trap

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# A Bucket Trap

- Operating Principle
  - Condensate enters the trap by gravity at the bottom
  - Vapor trapped in the inverted bucket causes it to float on the liquid
  - As the vapor cools and condenses, the bucket sinks
  - The sinking bucket opens the discharge valve
  - Steam pressure forces the condensate out
  - New vapor entering the trap causes the bucket to float again, closing the trap
- Two position operation (open or closed)
- Capable of "lifting" condensate

### A Bucket Trap









- Operating Principle
  - Condensate enters the trap by gravity at the bottom
  - The float "floats" on the liquid, modulating the flow of condensate out to the return system to match the load
  - Air rises to the top and is cooler than the steam
  - A thermostatic bellows vents the air
- Modulating capability is a good match for modulating loads
- Gravity must push condensate through the trap
  - Loads typically operating below atmospheric pressure
  - Trap requires the load to be elevated above it to move condensate through it

# A Float and Thermostatic Trap (F&T Trap)

Leaking Steam Trap Discharge Rate				
Trap	Steam Loss (lbs/hr)			
Orifice	Steam Pressure (psig)			
Diameter	15	100	150	300
1/32	0.85	3.3	4.8	-
1/16	3.4	13.2	18.9	36.2
1/8	13.7	52.8	75.8	145
3/16	30.7	119	170	326
1/4	54.7	211	303	579
3/8	123	475	682	1,303

Data taken from the Boiler Efficiency Institute Assumes steam is discharging to atmospheric pressure Based on DOE Steam Tip Sheet #1

#### A Condensate Issue Unique to Planets with Gravity









# Returning Condensate

# Vacuum Condensate Return Pump

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Operating Principle

- Condensate returns by gravity to an isolated chamber

- Pumps run continuously circulating water through a venture which pulls a vacuum on the chamber and thus, the condensate return system
- A float switch in the chamber triggers a discharge cycle when the chamber fills up and the condensate is pumped out to the boiler feed water system
- Minor leaks in the return system tend to eradicate the ability of the pumps to pull a vacuum
- RCx Opportunity
  - Give up on the vacuum approach
  - Allow the pumps to cycle on the float switch

Vacuum Condensate Return Pump

- Reasons for vacuum
  - Remove air from the steam system
    - Air is not a good heat transfer media
    - Air displaces steam which is the intended heat transfer media
    - Air can cause corrosion
  - Older systems
    - Lower steam system saturation pressure and thus lower operating temperature as a temperature control strategy
- Not Intended to Lift Condensate

#### Vacuum Condensate Return Pump

Steam Powered Condensate Pump

- Operating Principle
  - Condensate returns to the tank by gravity
  - When the tank fills up, level switches open a steam valve that pressurizes the tank, forcing the steam out the discharge pipe
  - A check valve prevents back-flow into the condensate return system
  - Could be powered by compressed air also
- Required no electricity to run

But the boiler or compressor making the

steam or air still use energy

#### Steam Powered Condensate Pump



Steam Powered Condensate Pump The Video: Watch the Sight Glass and Listen

#### Pressure Reducing Valves

- Found on both hot water and steam systems
- Different application considerations









# Pressure Relief and Pressure Reducing Valves

### Pressure Relief Valves

- Found on both hot water and steam systems
- Different application considerations











# One Pipe Steam Systems







# Taking a Look at Equipment

# Inside A Fire Tube Boiler

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# Inside a Water Tube Boiler

NATURA

GAS







### Inside a Condensing Boiler










# Inside a Condensing Boiler that was not really "or equal"





### The Difference Between Boilers



### A Typical Neutralization Basin

A Typical Neutralization Basin

### An Electric Boiler





Maintaining Circulation; An Important Hot Water Boiler Design Consideration

TITIT

EATING WATER RETURN

Variable Flow/Variable Volume Systems



#### Boiler Loop – A Constant Flow Requirement



### Merging Two Concepts





#### Variable Flow Primary/Secondary



#### Variable Flow Primary/Secondary – Full Load



#### Variable Flow Primary/Secondary – 50% Load



#### Variable Flow Primary/Secondary with Protection for the Non-condensing Boiler







#### Model 4010

### Three-Way Thermostatic Valve



**Features** 

Self-Contained

Non-Adjustable

Tamper-Proof

Wide Range of

Temperatures

Compact

4010 4" Flange 4010M 4" Flange with Manual Override

FPE Thermostatic Valves utilize the principle of expanding wax, which in the semi-liquid state undergoes large expansion rates within a relatively narrow temperature range. The self-contained element activates a stainless steel sleeve, which directs flow. All FPE Thermostatic Valves are factory set at predetermined temperatures: no further adjustments are necessary. A wide range of temperatures are available for water and oil temperature control applications.

When used in a diverting application, on start-up the total fluid flow is routed back to the main system. As fluid temperature rises to the control range, some fluid is diverted to the cooling system. As fluid temperature continues to increase, more flow is diverted. When the thermostat is in a fully stroked condition, all fluid flow is directed to the cooling system. FPE Thermostatic Valves may also be used in a mixing application.

In a mixing application, hot fluid enters the "B" port and colder fluid enters the "C" port. The flows mix and the thermostat adjusts to reach the desired temperature, exiting the "A" port.

Standard FPE thermostatic valve housings are made from aluminum and grey iron castings, however, ductile iron, bronze, steel and stainless steel housings are available.

Available Connections: 125 # FF Flange, 150 # RF Flange, 300 # RF and FF Flange, and Navy Flanges

Optional features: Manual Override, High Over Temp element, Plated element

FLUID POWER ENERGY, INC. W229 N591 Foster Court • Waukesha, WI 53186 262 • 548 • 6220 Fax 262 • 548 • 6239 www.fpevalves.com



#### Model 4010

MODEL NUMBER	BODY MATERIAL (*)	NOMINAL PIPE SIZE	PRINCIPAL DIMENSIONS (UNITS in. & (mm))				MAX WIDTH	FLANGE DRILLING		LING	NO. OF	APPROX.	NOTES OR
			"X"		.m.	"Z"	OTHER	OF HOLES	OF HOLES	BOLT CIRCLE	ELEMENTS	WEIGHT	ENDNOTES
*4010	A, B, D	4" 125# FF FLANGE	N/A	7 15/16 (201.61)	15 7/8 (403.23)	8 9/16 (217.49)	11 (279.40)	8	3/4 (19.05)	7 1/2 (190.50)	4	A=95#, B=116# D=90#	
	S, SS	4" 150W RF FLANGE	N/A	7 15/16 (201.61)	15 7/8 (403.23)	8 9/16 (217.49)	11 (279.40)	8	3/4 (19.05)	7 1/2 (190.50)	4	S & SS=104#	
	A, B, D	4" 125# FF FLANGE	N/A	7 15/16 (201.61)	15 7/8 (403.23)	8 9/16 (217.49)	11 (279.40)	8	3/4 (19.05)	7 1/2 (190.50)	4	A=97#, B=118# D=92#	Manual Override
-4010M	S, SS	4" 150# RF FLANGE	N/A	7 15/16 (201.61)	15 7/8 (403.23)	8 9/16 (217.49)	11 (279.40)	8	3/4 (19.05)	7 1/2 (190.50)	4	S & SS=106#	Manual Override
*4010X	S, SS	4" 300# RF FLANGE	N/A	7 15/16 (201.61)	15 7/8 (403.23)	8 9/16 (217.49)	11 (279.40)	8	7/8 (22.23)	7 7/8 (200.03)	4	S & SS=126#	
*4010XM	S, SS	4" 300/ RF FLANGE	N/A	7 15/16 (201.61)	15 7/8 (403.23)	8 9/16 (217.49)	11 (279.40)	8	7/8 (22.23)	7 7/8 (200.03)	4	S & SS=127#	Manual Override







PART #	DESCRIPTION
*4010	VALVE BODY (*See table for material)
*4020	VALVE COVER (*See table for material)
2071	LIP SEAL
4080-C	GASKET
2050-Temp	THERMOSTAT (Temp to follow dash)
1604	HEX BOLT
1605	LOCK WASHER
1590	NAMEPLATE
FPE Model 400	0 Replacement Kit (Includes the following:)
4080-0	GASKET

(4) 2071 LIP SEAL THERMOSTAT (Temp to follow dash) (4) 2050-Temp

APPLICATION CHARTS



\*4010M \*4010XM



DIVERTING SYSTEM

### EPE-4010-11503

FLUID POWER ENERGY, INC. W229 N591 Foster Court · Waukesha, WI 53186 262 • 548 • 6220 Fax 262 • 548 • 6239 www.fpevalves.com

#### To Order

Specify house temperature desired, and mo-nal. For Model coding information, the website or consult your consentative







\*4010, \*4010X

200

16

Pressure Drop, PSID

0

100

\*4010M, \*4010XM

500

Flow vs. Pressure Drop

300



DESCRIPTION
VALVE BODY (*See table for material)
VALVE COVER (*See table for material)
LIP SEAL
GASKET
THERMOSTAT (Temp to follow dash)
HEX BOLT
LOCK WASHER
NAMEPLATE

FPE Model 4000 R	eplacement Kit (Includes the following:)
4080-C	GASKET
(4) 2071	LIP SEAL
(4) 2050-Temp	THERMOSTAT (Temp to follow dash)



600



400

FLOWIN U.S. GPM SAE 30 @ 140° F



DIVERTING SYSTEM



#### Design Day, Cold Start



#### Design Day, Cold Start, Boiler LWT Reaches 140°F



### More Information on Variable Flow Systems

https://tinyurl.com/VariabeFlow





## Controlling the Fuel; Critical for All Applications





### Why Deaerators and Water Treatment Matter



Condense steam to create hot water

Heat Exchangers; Steam Converters

### A Heat Exchanger Tube Bundle







### Heat Exchangers; Cogeneration Equipment

Steam or hot water recovered from power generation equipment

 Heat balance is critical to over-all plant efficiency

### Heat Exchangers; Heat Exchangers/Recovery Equipment

- "Higher grade thermal source" = relative term
- Renewable energy is frequently low grade thermal energy



### Questions?



"PG&E" refers to Pacific Gas and Electric Company, a subsidiary of PG&E Corporation. ©2017 Pacific Gas and Electric Company. All rights reserved. These offerings are funded by California utility customers and administered by PG&E under the auspices of the California Public Utilities Commission.



### Break Time We will return at 10:25 AM Pacific Time



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