

Fans, Ducts and Air Handling Systems: Design, Performance and Commissioning Issues

Evaporative Cooling



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Why Evaporative Cooling?

An Experiment

Data Logger

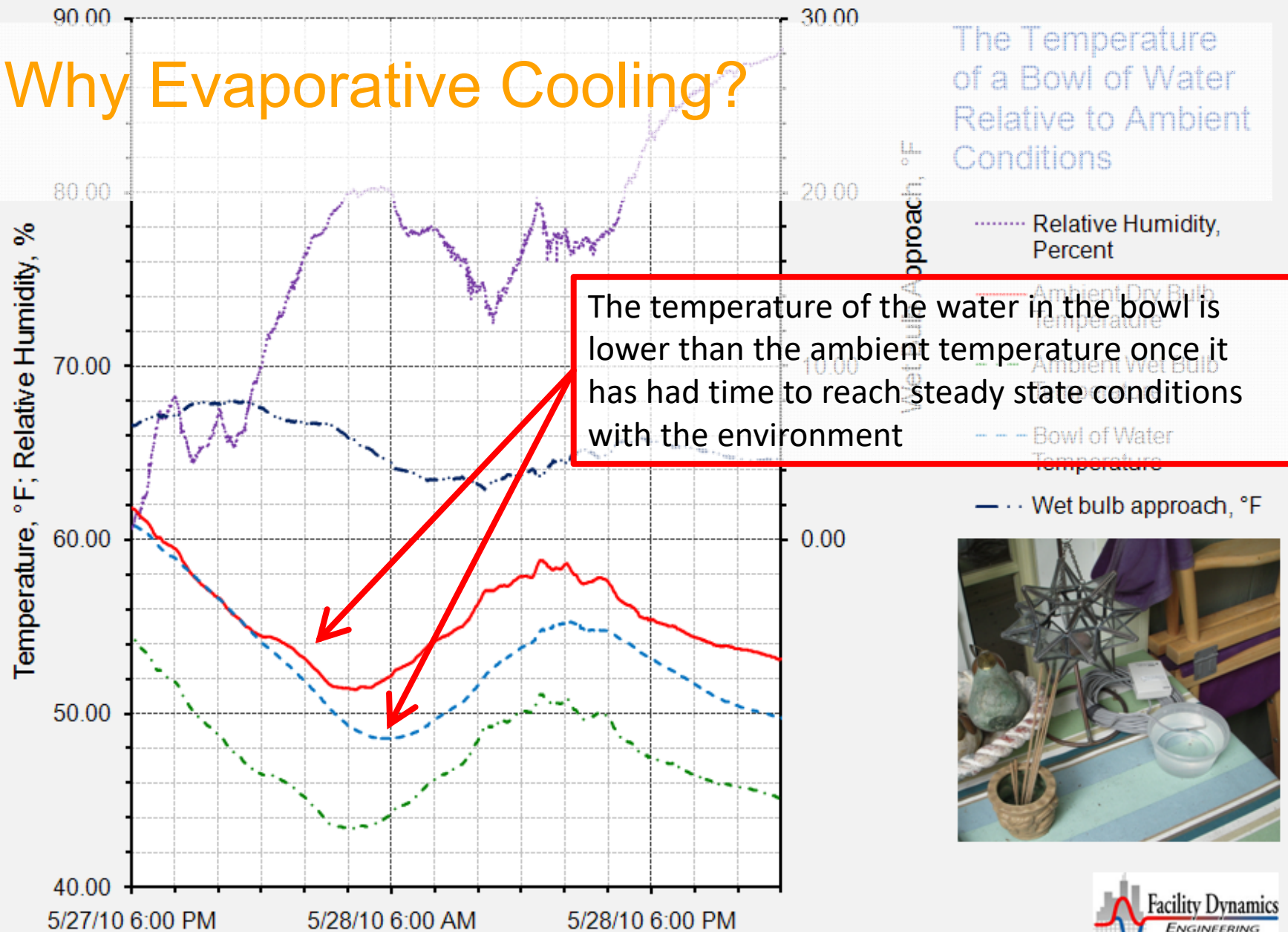
Ambient
Temperature and
Humidity Sensors

Water that has been Sitting
on the Table for a While

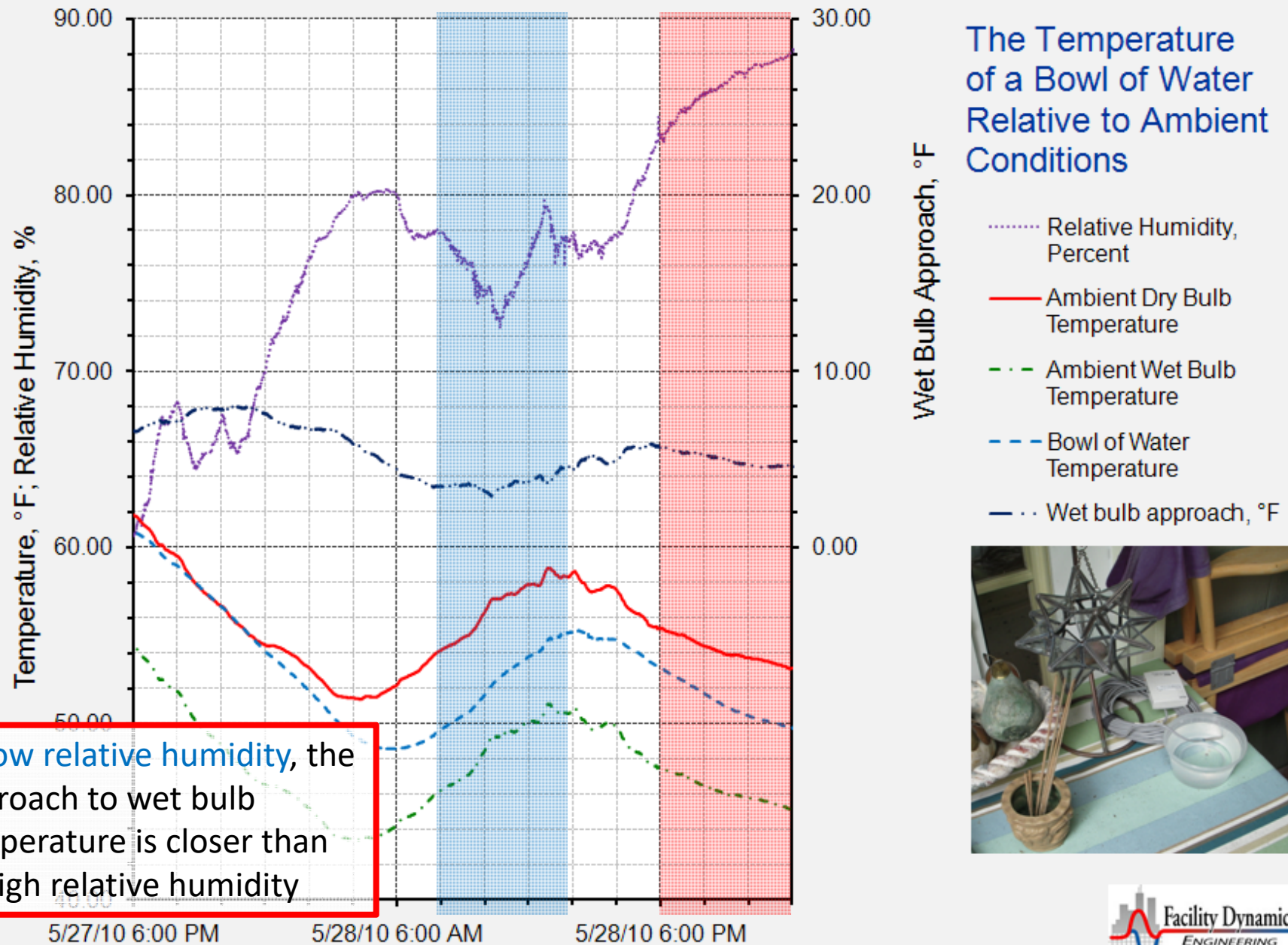
Water
Temperature
Sensor

Table on Deck

Why Evaporative Cooling?



The Temperature of a Bowl of Water Relative to Ambient Conditions



Characteristics of Free Cooling via an Evaporative Process

Water is not free

- Evaporating 1 lb of water at 75°F = 970 Btu
- 1 ton of cooling = 12,000 Btu/hr
- 1 ton of cooling = 1.49 gallons of water
- 1 ton of cooling = \$0.005 (Portland Oregon rates, 02/2011)

Producing 100 tons of cooling for 24 hours using an evaporative process would cost about \$13 for water



Characteristics of Free Cooling via an Evaporative Process

Water quality needs to be maintained

- Basin water is a good breeding ground for microorganisms
- Evaporation leaves minerals behind
 - Minerals will form scale
 - Scale degrades the equipment

Characteristics of Free Cooling via an Evaporative Process

- Maintaining water quality
 - Make up extra water to dilute solids by “blowing down” the system when solids build up
 - Inspection and cleaning required on a regular basis
 - Treat water for biological growth

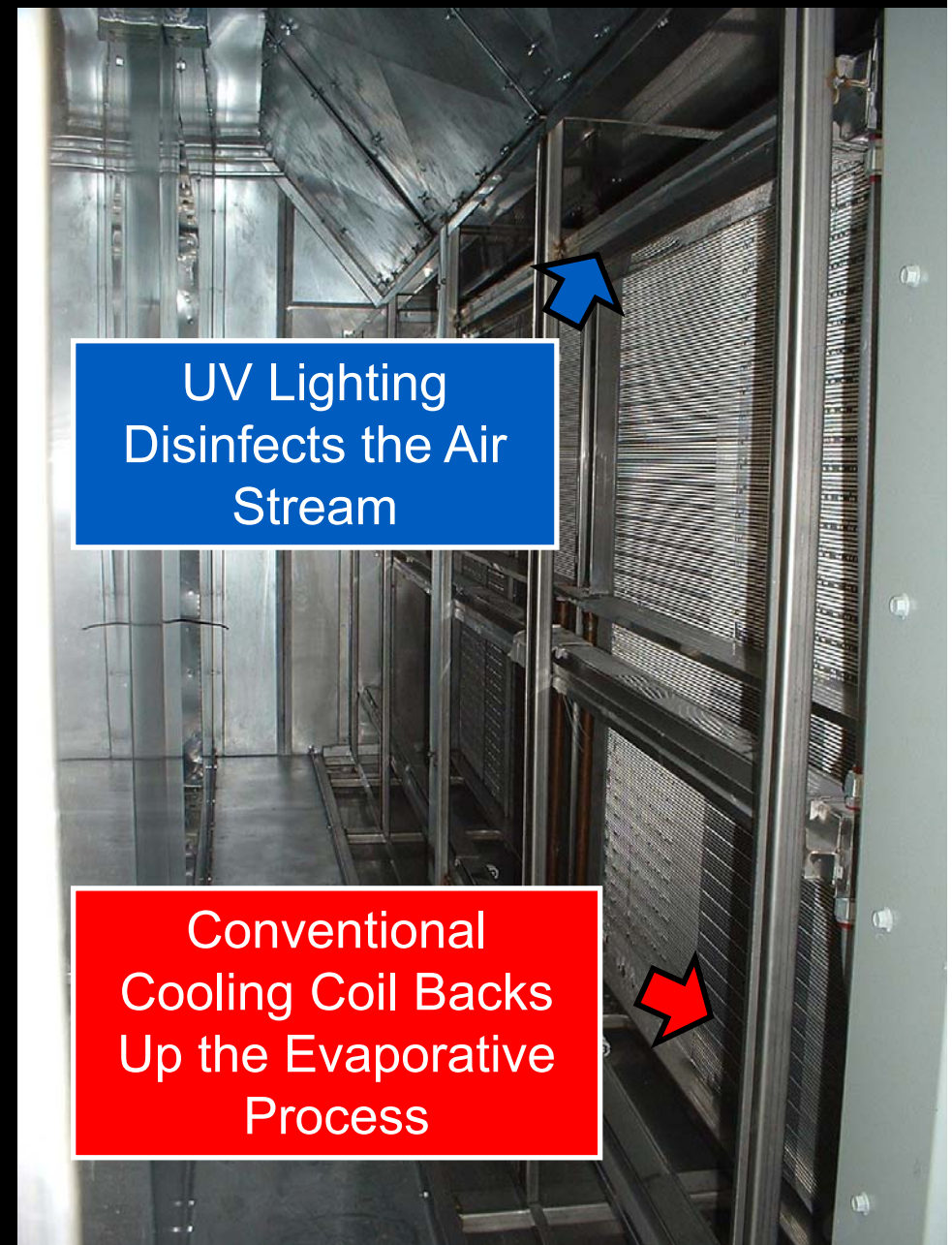
Characteristics of Free Cooling via an Evaporative Process

Systems will be 100% Outdoor air

- Other utilities will have to deal with 100% outdoor air
- Filters will have to deal with 100% outdoor air

Back up systems erode some of the savings potential

- Extra coils = Extra first cost = longer payback
- Extra coils = Extra pressure drop = More fan energy = More fan heat = More cooling required



Characteristics of Free Cooling via an Evaporative Process

Cooling is produced by evaporating water into the air stream

- Discharge temperatures approach the wet bulb temperature entering the evaporative process
- All other things being equal, spaces served with evaporative cooling will be more humid than spaces served by conventional cooling



Some Definitions

Direct Evaporative Cooling

The system air comes in direct contact with the water that is evaporated to provide cooling

- *Cools but adds humidity to the primary air stream*

Direct Evaporative Cooler
in Primary Air Stream



Some Definitions

Direct Evaporative Cooling

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Indirect Evaporative Cooling

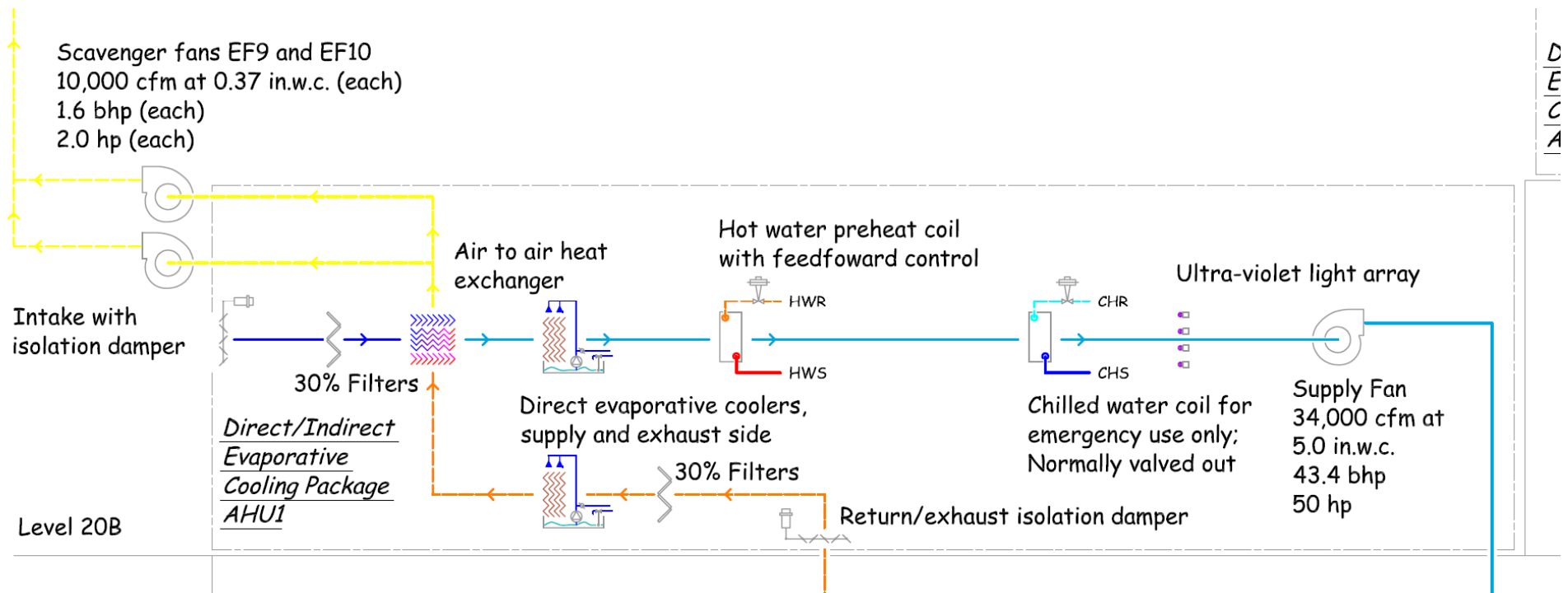
A secondary air stream is cooled by a direct evaporative process and then used to cool the primary air stream


- *Cools with out adding humidity to the primary air stream*

Access Panels to Air to Air Heat Exchanger that Couples the Indirectly Cooled Secondary Air Stream to the Primary Air Stream

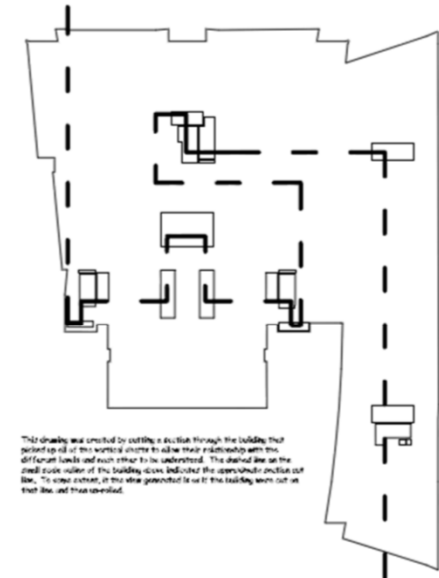
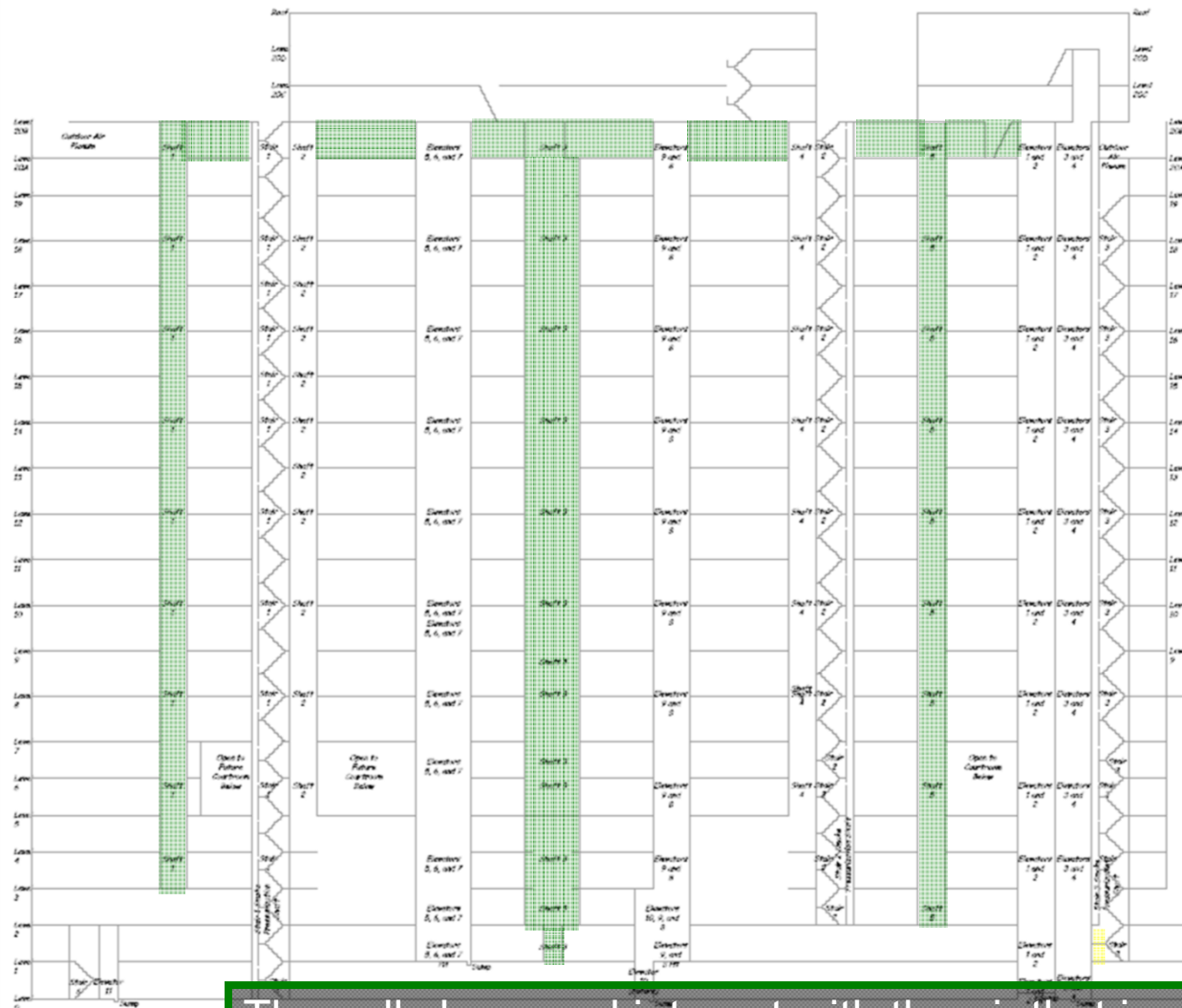


An Evaporatively Cooled AHU





What do the low temperature air VAV AHUs, the constant volume direct-indirect evaporative cooling AHUs, and the central chilled water plant that are being installed in this building have in common?



This drawing was created by cutting a section through the building that passed up all of the working shafts to allow their relationship with the different levels and each other to be understood. The dashed line on the small scale plan of the building shows indicated the approximate section cut line. To view extent, in this view generated to cut the building, were cut on their line and then overlaid.

They all share and interact with the air that is moving in these common return air shafts and plenums

A Bit of Information About the AHUs

Direct/Indirect Evaporative Cooling AHUs

- 120,000 cfm total
- Discharge air at 68.8/67.8 tdb/twb
- Space condition – 75°F/77% RH

Low Temperature Air AHUs

- 380,000 cfm total
- Discharge air at 42.9/42.7 tdb/twb
- Space condition – 72°F/38% RH

A Couple of Questions

Do you think the shared plenum space and related mixing of the various air streams is a good thing, a bad thing or doesn't matter?

If it does matter:

- Why?
- Is there an energy or performance issue?
- How would you assess any energy or performance issues?

Seattle Design Conditions

HDCLimatic - ASHRAE 1997 Fundamentals [Print] [Close]

☒ **COOLING** USA 449 Elevation, Feet ☒ English (IP)
☒ **HEATING** Washington 47.45 North Latitude ☐ Metric (SI)
☒ **WIND** Seattle, Int'l Airport 122.30 West Longitude

SUMMER COOLING				Evaporation			Dehumidification		
	DB °F	MWB °F	°F db	WB °F	MDB °F	°F db	DP °F	MDB °F	°F db
0.4%	85	65	84.92	66	83	82.94	60	71	70.88
1%	81	64	81.32	65	79	79.34	58	69	68.90
2%	78	62	77.72	63	76	76.10	57	68	67.64

Extr. Annual Max. DB °F 92 Std. Dev. °F 4 Mean Daily Range DB °F 18

WINTER HEATING				Coldest Month		Extreme Annual Daily	
	DB °F	RH %	°F db	WS mph	MCDB °F	DB °F	Std. Dev. °F
99.6%	23	50	23.36	0.4%	24	44	19
99%	28	50	28.04	1%	21	44	7

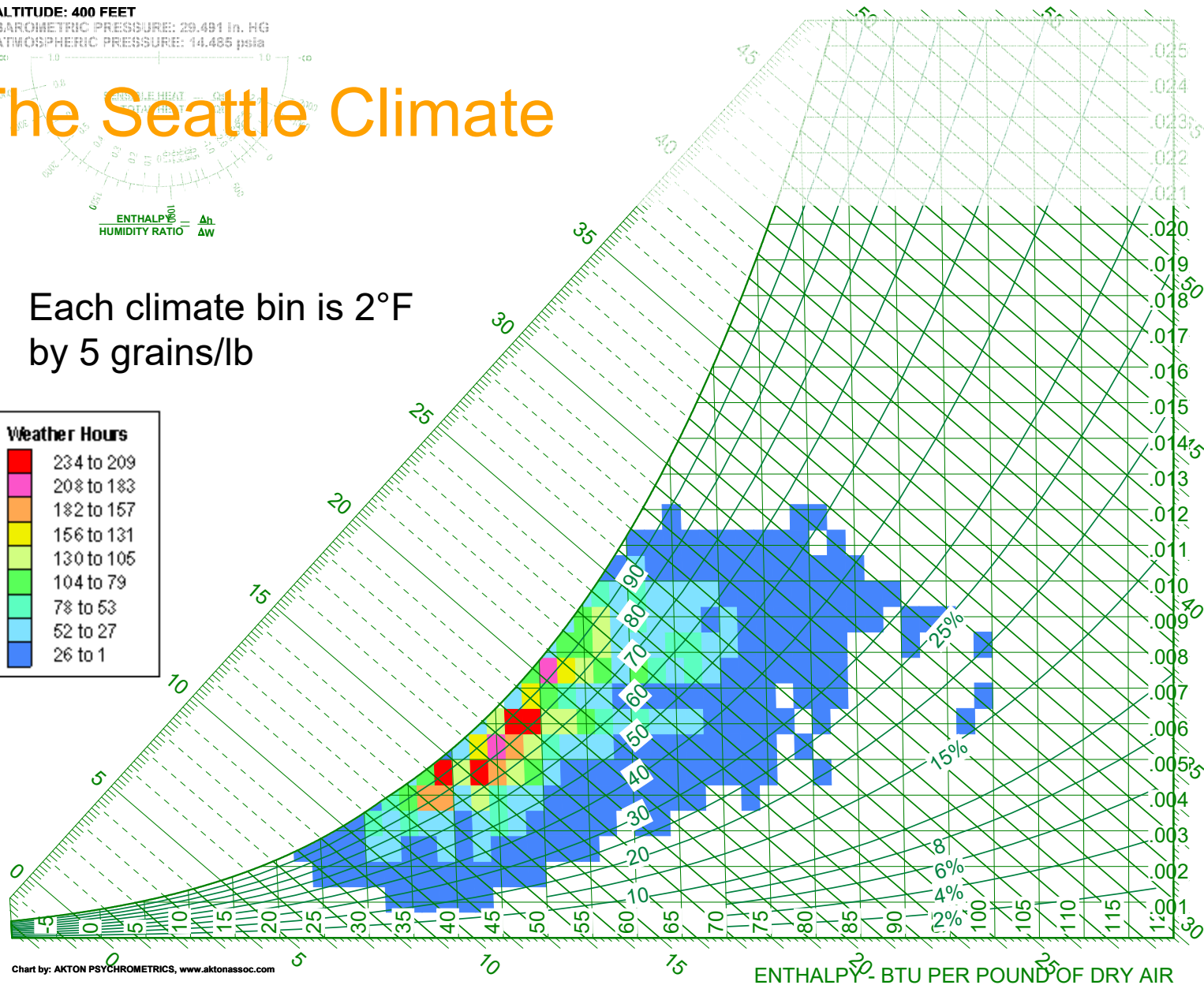
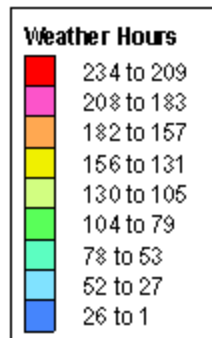
WIND Coincident with 0.4% DB (cooling) MCWS 10 mph PWD 350 deg.
 Coincident with 99.6% DB (heating) MCWS 10 mph PWD 10 deg.

Extreme Wind Speed 1% 22 mph 2.5% 19 mph 5% 17 mph

ALTITUDE: 400 FEET
 BAROMETRIC PRESSURE: 29.491 in. HG
 ATMOSPHERIC PRESSURE: 14.485 psia

The Seattle Climate

Each climate bin is 2°F
 by 5 grains/lb



ALTITUDE: SEA LEVEL
 BAROMETRIC PRESSURE: 29.921 in. HG
 ATMOSPHERIC PRESSURE: 14.696 psia

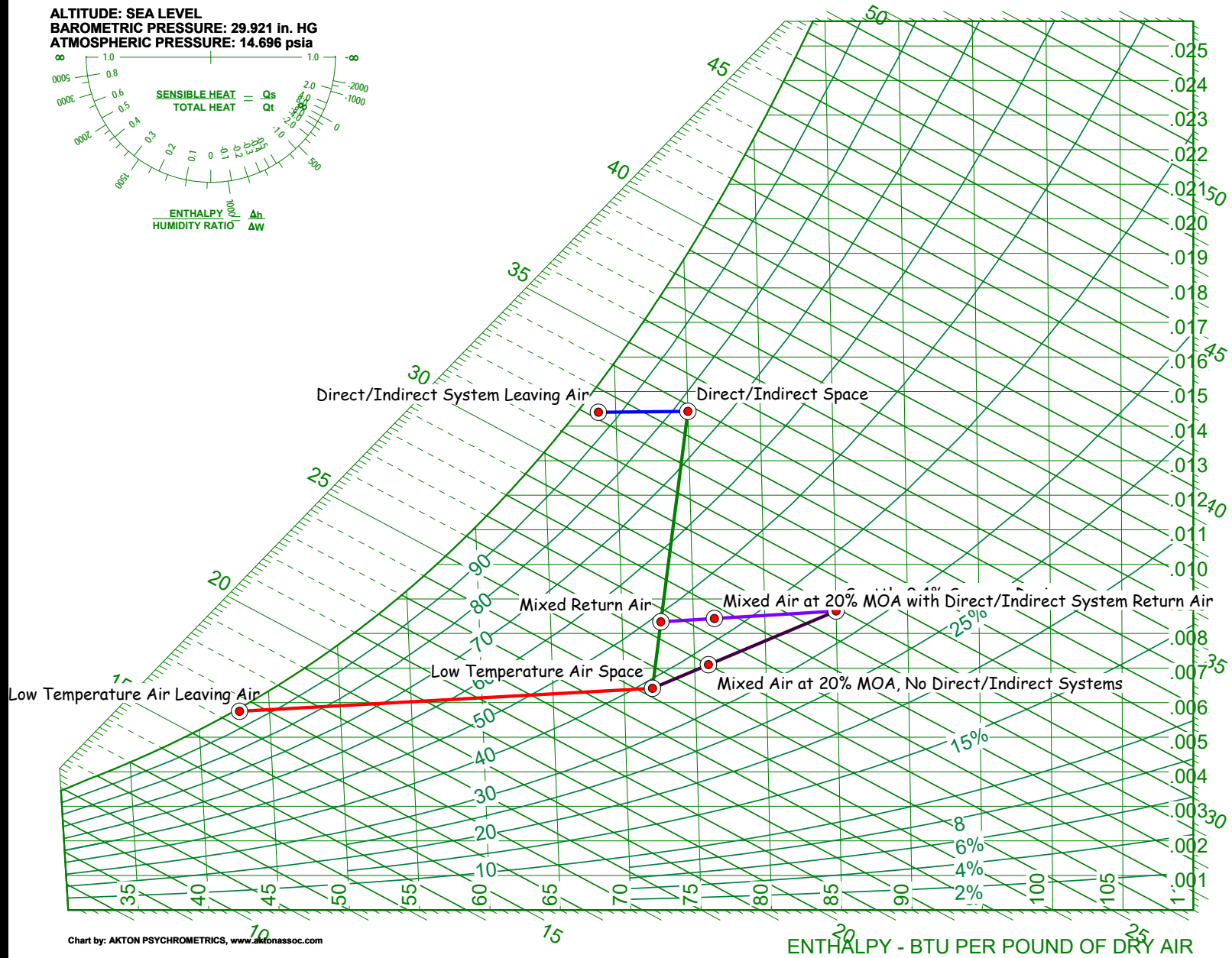
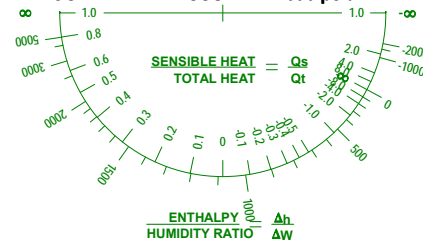


Chart by: AKTON PSYCHROMETRICS, www.aktonassoc.com

Total Load

$$Q = 4.5 \times cfm \times \Delta h$$

Where :

Q = Load in btu/hr

4.5 = A units conversion constant

cfm = Flow rate in cubic feet per minute

Δh = Enthalpy difference across the element in btu/lb/°F

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