

Facility Dynamics

ENGINEERING

Inputs and Outputs – The Field Perspective

Outputs and Actuator Considerations

Presented By:

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Proportional Actuators

Electric

- Tristate
 - Two sets of contacts
 - Clockwise rotation
 - Counterclockwise
 - Can be used as two position
- Proportional signal
 - 4-20 milliamp
 - 2-10 vdc
 - 1-5 vdc

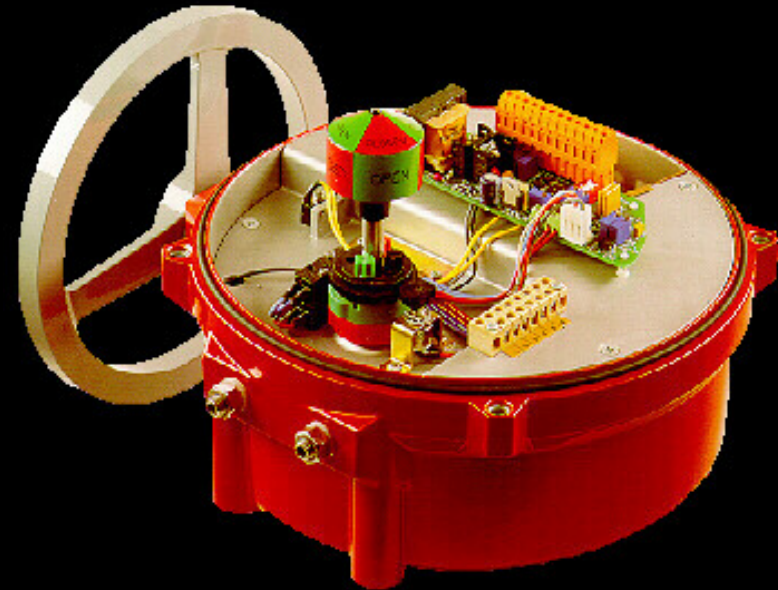
Image courtesy Bray Controls; used with permission



Proportional Actuators

Electric

- Electronic controller boards required to integrate the drive system with the control and power system



Proportional Actuators

Electric

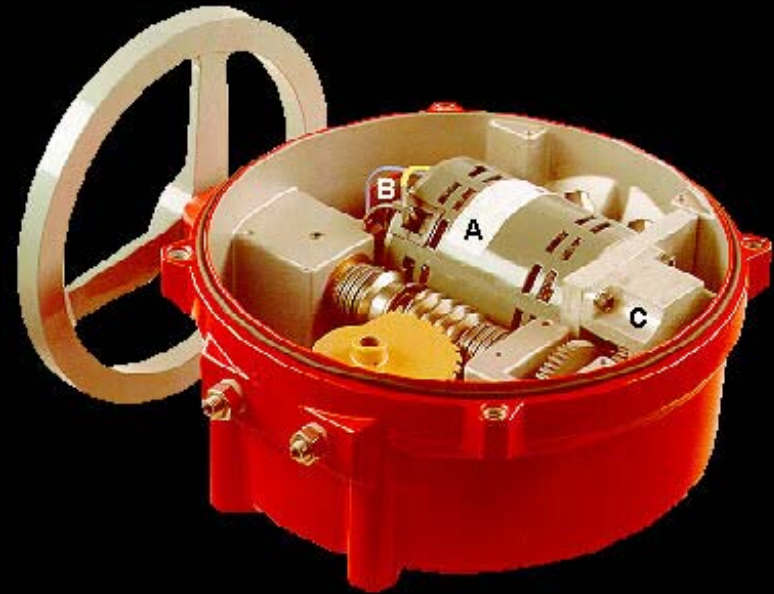
- Frequently powered with low voltage cabling system to minimize cost
 - Power limited circuits mean that the instantaneous power available to move something is limited
 - Small motors are geared to manage the available power
 - Torque may be limited unless high voltage power is provided



Proportional Actuators

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 - Torque may be limited unless high voltage power is provided
- Speed is compromised
 - 15 seconds is fast
 - 30-60 seconds is common
 - 90 – 120 seconds is not unheard of in large actautors



Proportional Actuators

Pneumatic

- Power is a function of diaphragm or piston area
- Even with low pressure (20 psi) air you can get a lot of force and torque
- 50-150 psi operation is possible
- Can be used for modulation or two position operation

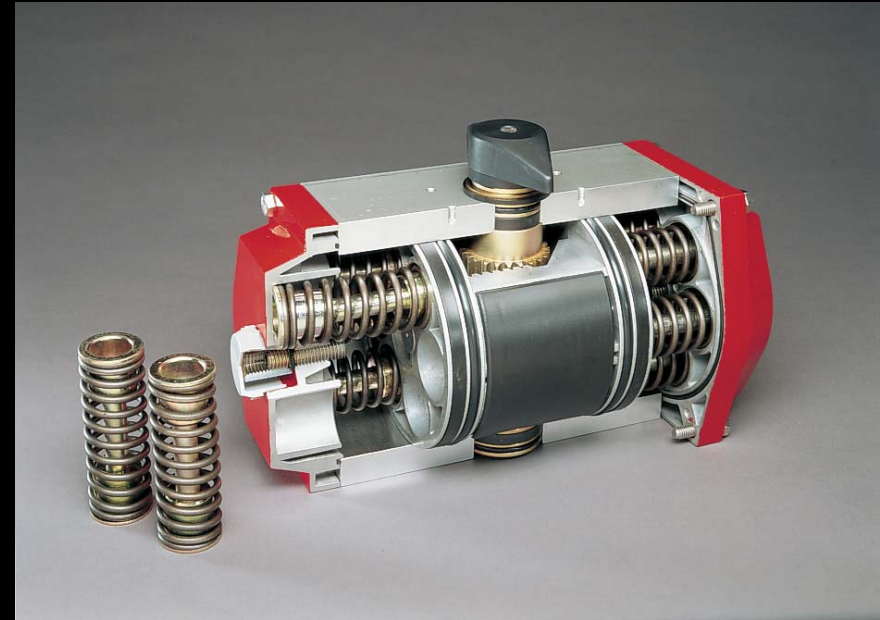
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Proportional Actuators

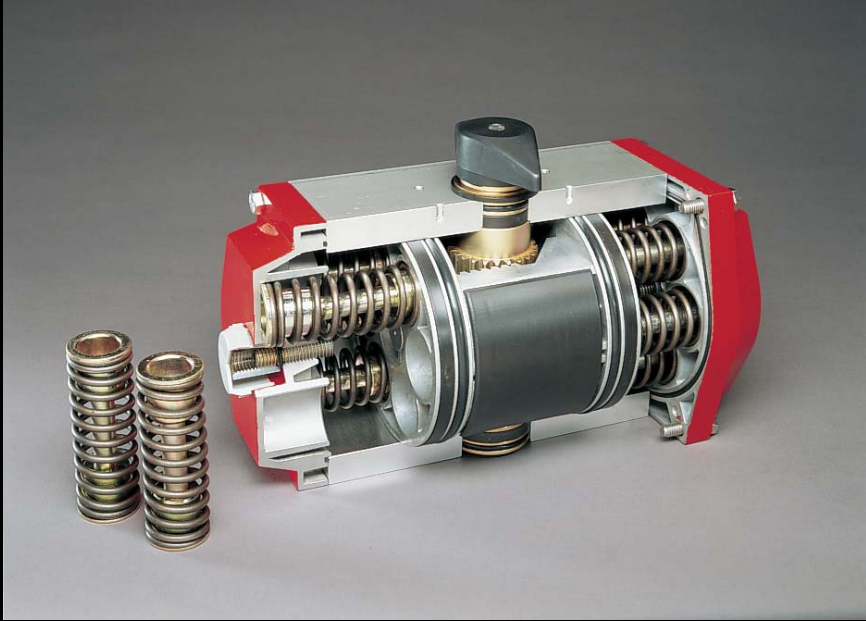
Pneumatic

- Relatively simple construction
- Easy to understand and repair
- Actuating speed can be a matter of seconds to go full stroke
 - The air line needs to be sized to provide the volume required for the desired speed
 - Beware of water hammer and air hammer



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- 
- Double acting cylinders can use air to provide equal power in either direction
 - Accumulators with check valves provide fail safe operation

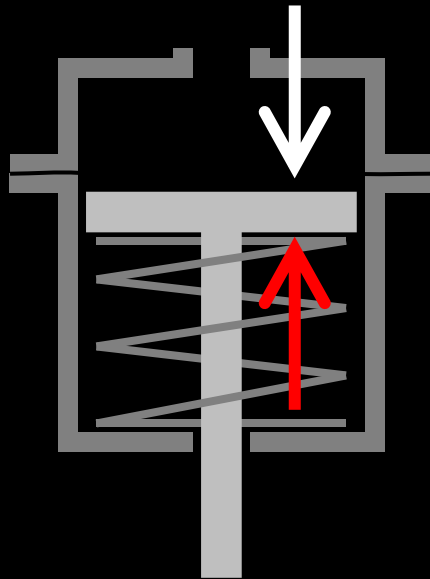
Proportional Actuators

Pneumatic

- 3-15 psi = standard signal
- Spring ranges allow sequencing
 - HW Coil – 3-5 psi = Full flow to no flow
 - Economizer – 7 – 10 psi = MOA to Maximum OA
 - CHW Coil – 12-15 psi – No flow to full flow



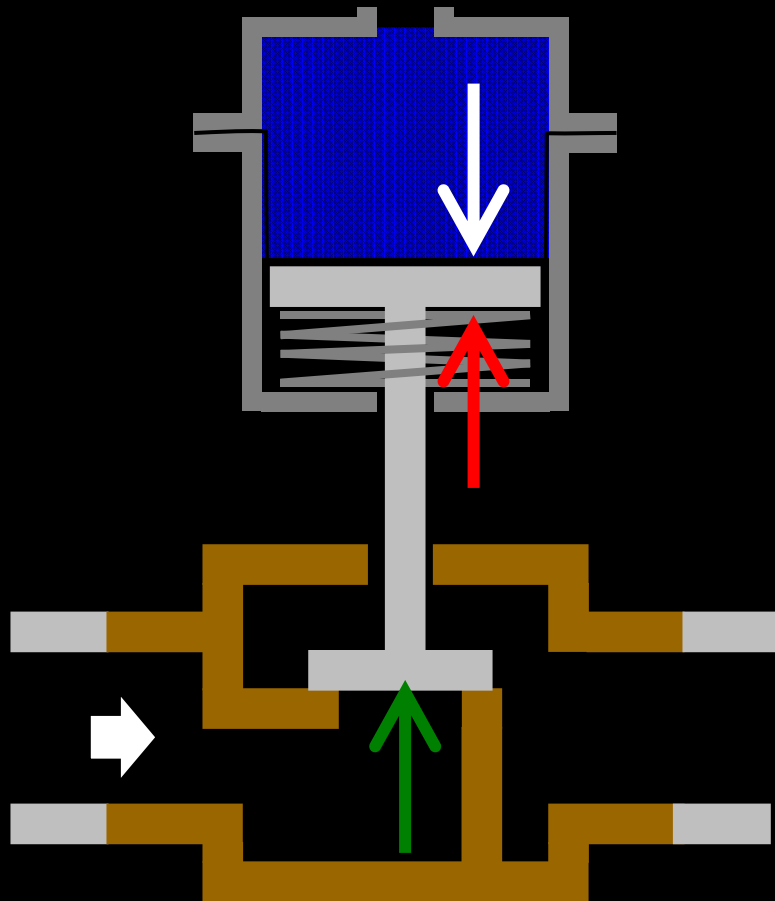
The Forces Acting on a Valve Actuator as Described in the Literature



Control Signal
 $\text{Force} = \text{Pressure} \times \text{Area}$

Spring
 $\text{Force} = \text{Spring Rate} \times \text{Deflection}$

The Forces Acting on a Valve Actuator in the Field



Control Signal

$$\text{Force} = \text{Pressure} \times \text{Area}$$

Spring

$$\text{Force} = \text{Spring Rate} \times \text{Deflection}$$

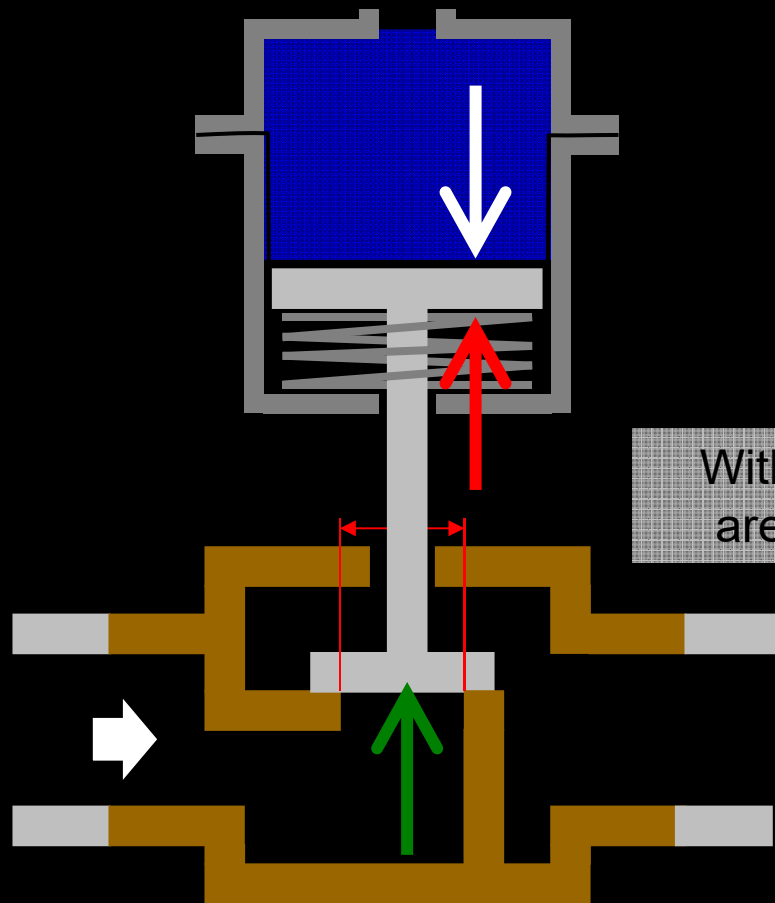
Valve Plug Differential Pressure

$$\text{Force} = \text{Effective Area} \times \Delta P$$

The Forces Acting on a Valve Actuator in the Field

Plug force will vary as the result of a number of operating variables

- Effective area can change

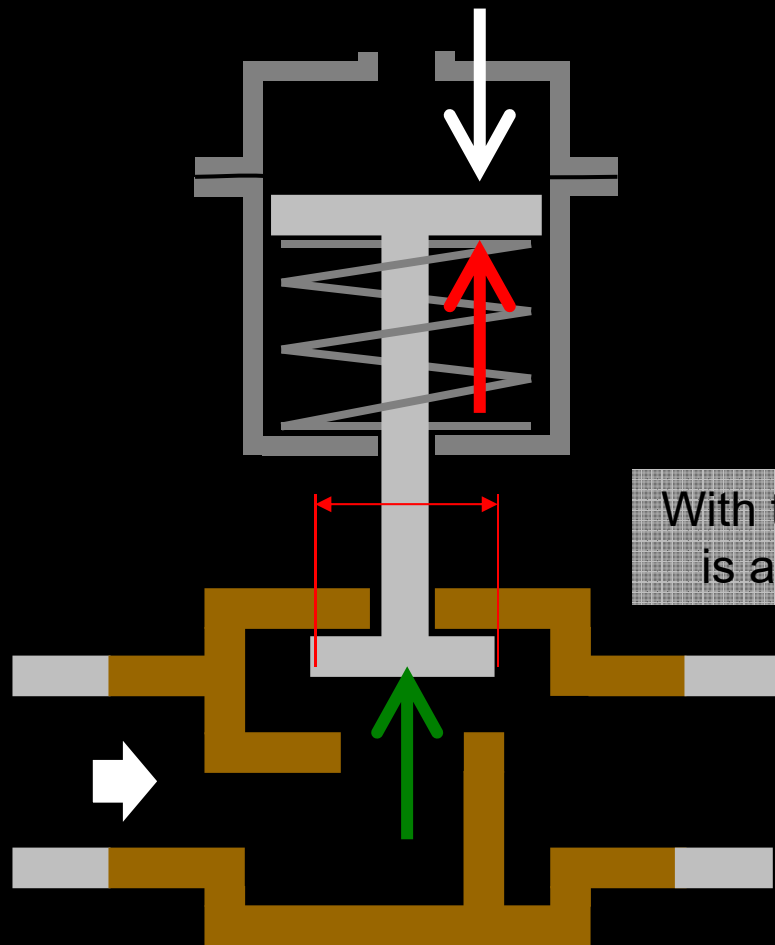


With the valve closed, the effective area is a function of this diameter

The Forces Acting on a Valve Actuator in the Field

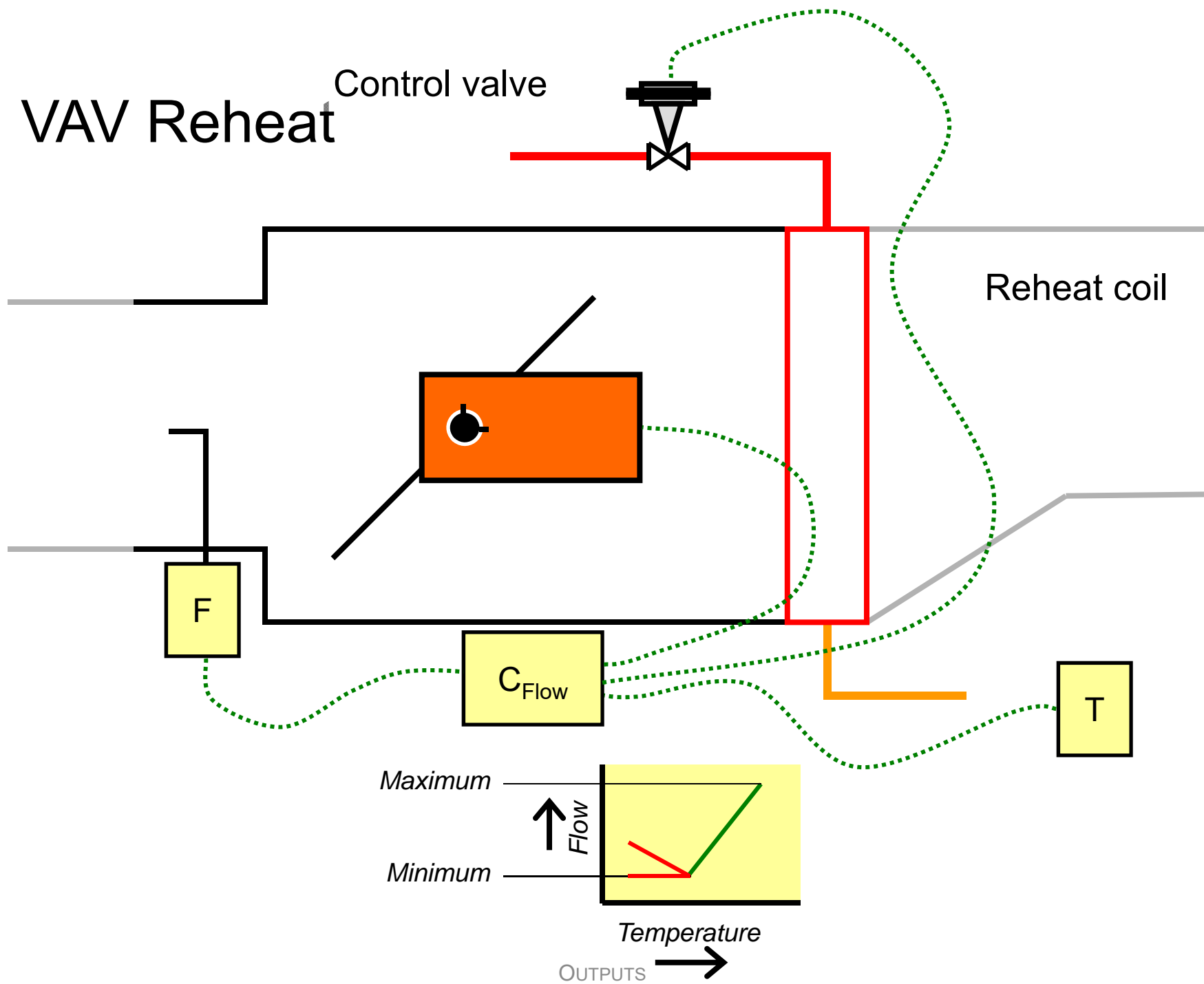
Plug force will vary as the result of a number of operating variables

- Effective area can change



With the valve open, the effective area is a function of a different diameter

VAV Reheat



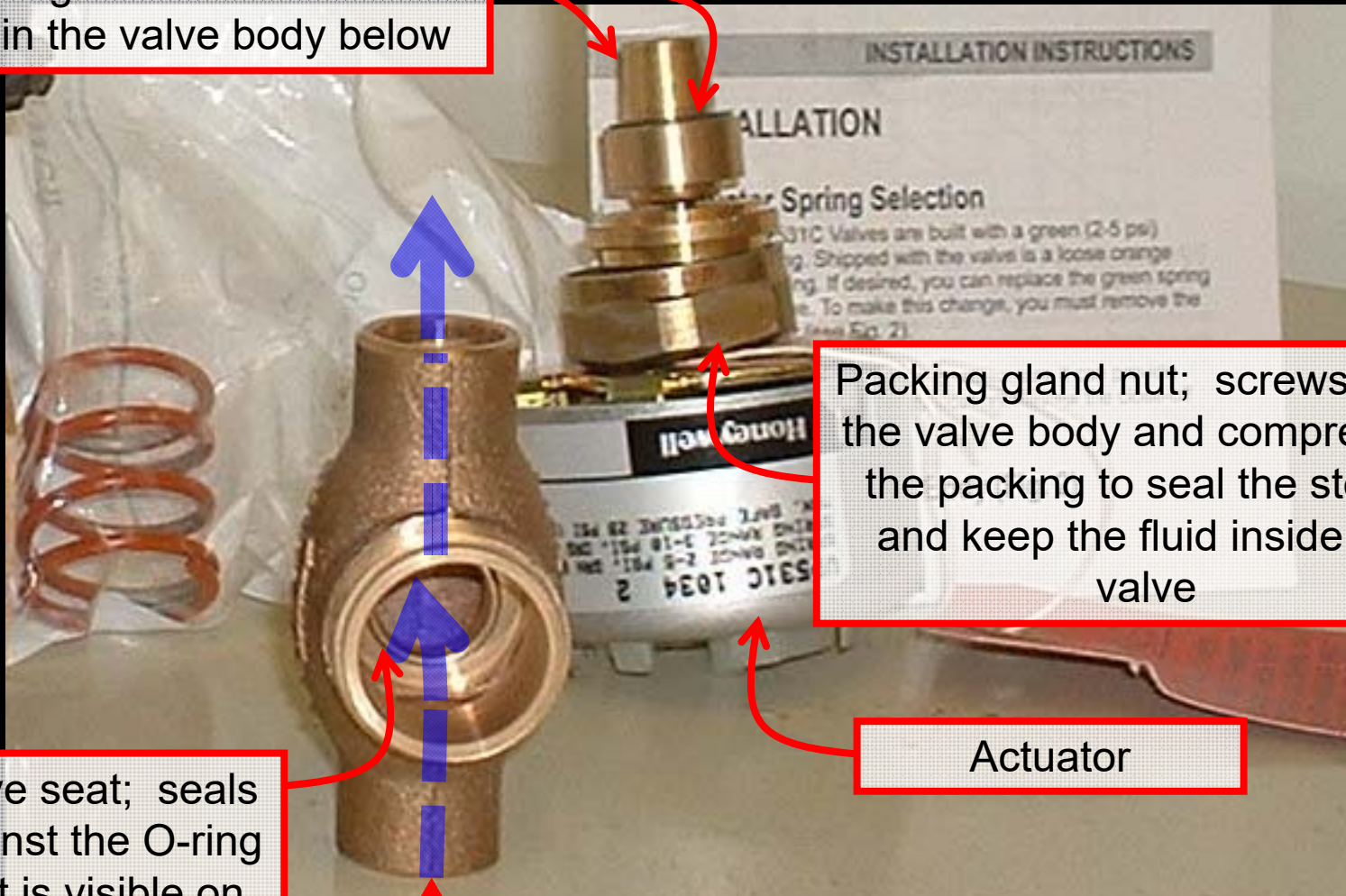
Valve plug; note the black O-ring that provides the sealing surface against the brass seat visible in the valve body below

Packing gland nut; screws on to the valve body and compresses the packing to seal the steam and keep the fluid inside the valve

Valve seat; seals against the O-ring that is visible on the valve plug

Actuator

Flow enters the valve through the lower connection, flows down and through the valve seat, and then out the upper connection



A Typical Pneumatic VAV Controller

Maximum
Flow

Increasing VAV Air
Terminal Flow



Minimum
Flow

A Typical Pneumatic
VAV Terminal -
Air Flow Damper
Operation
10-15 psi span

0 psig

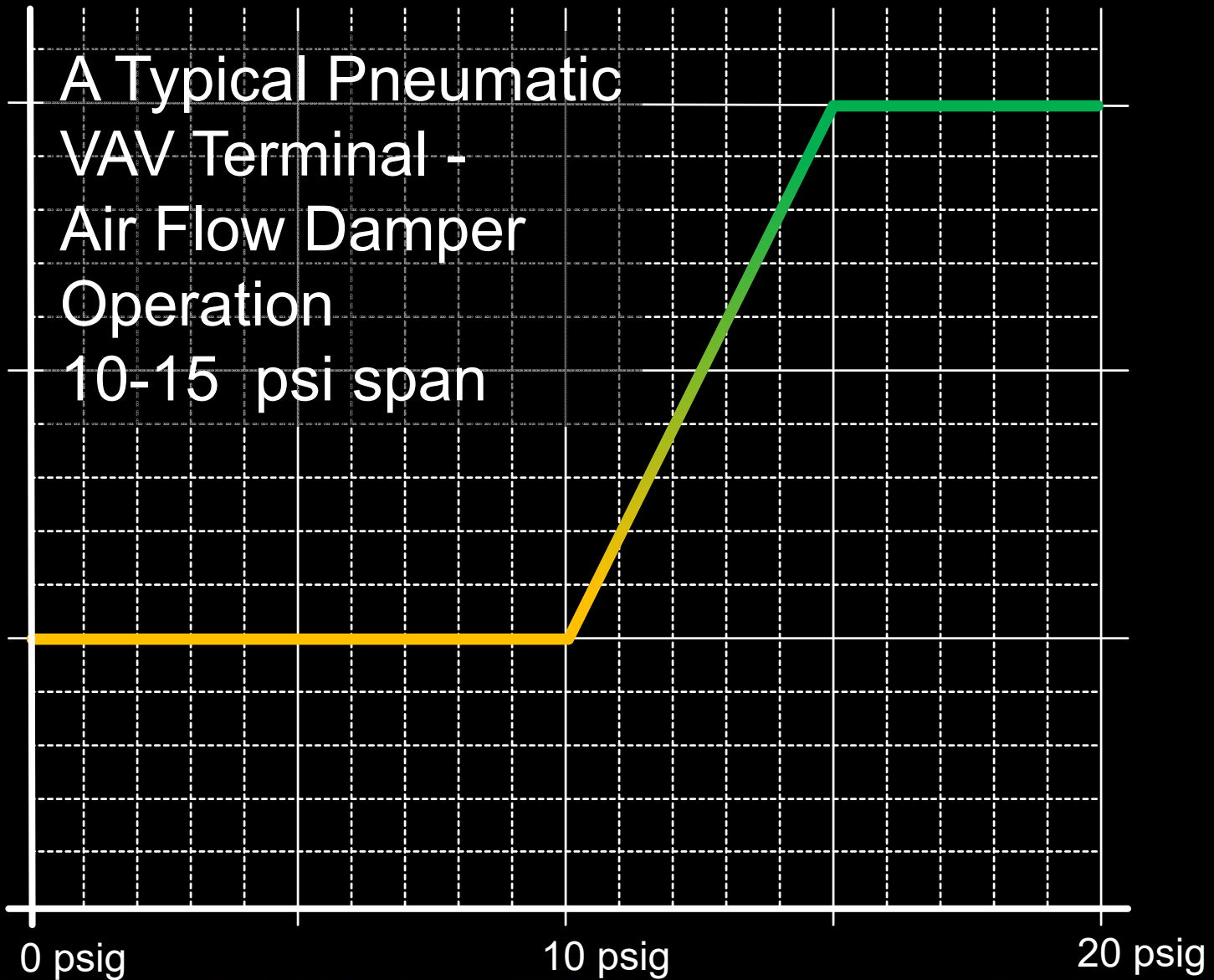
10 psig

20 psig

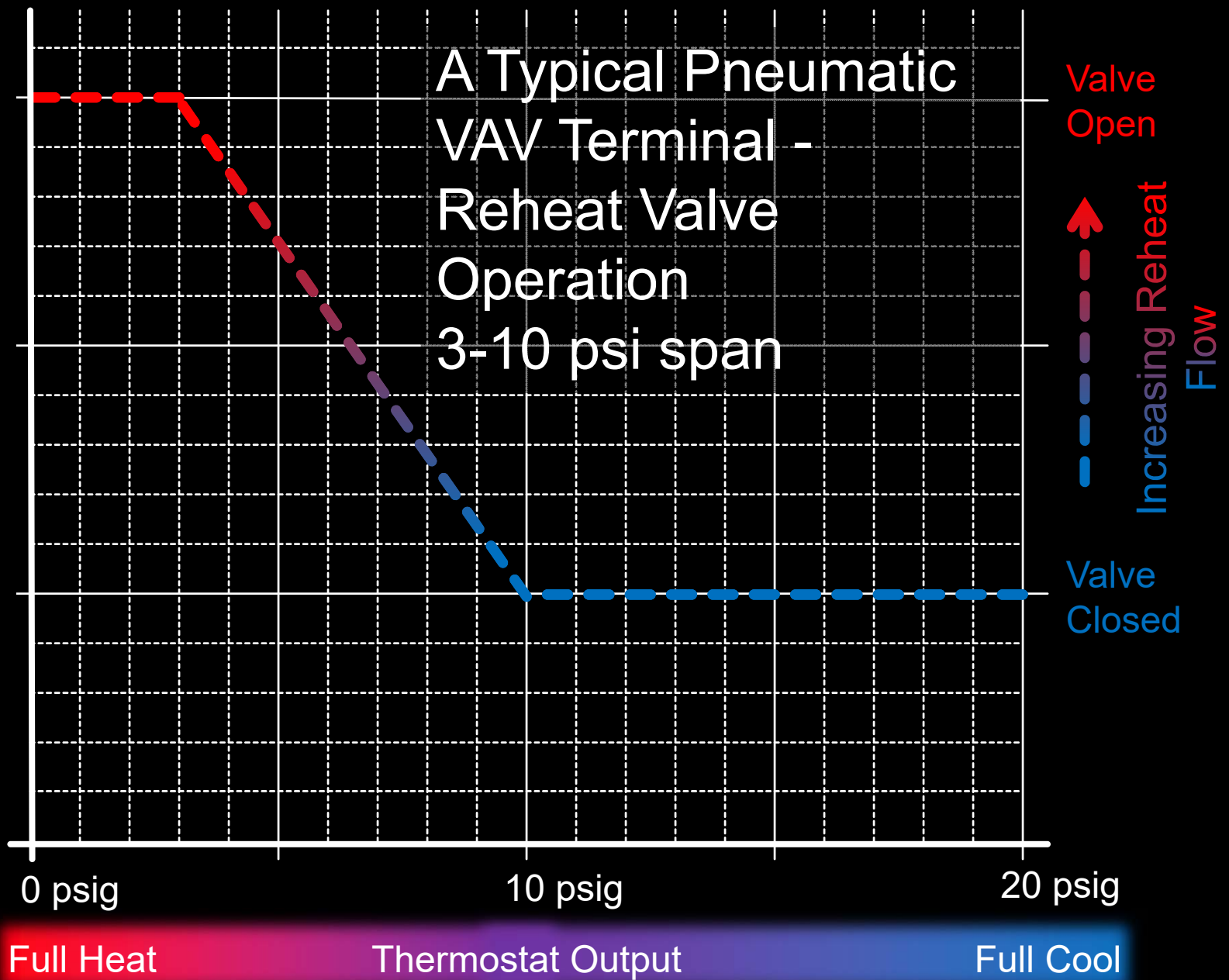
Full Heat

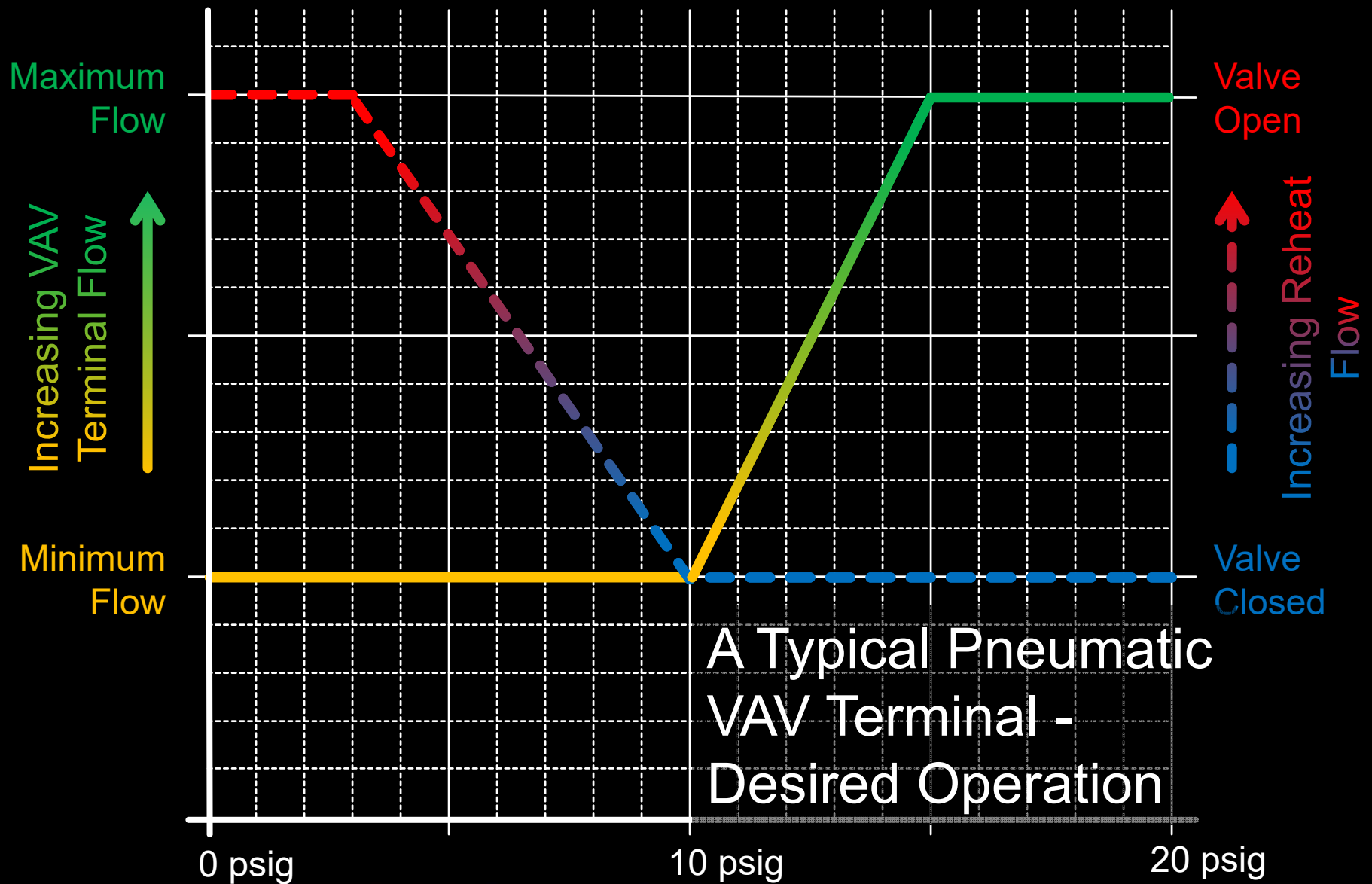
Thermostat Output

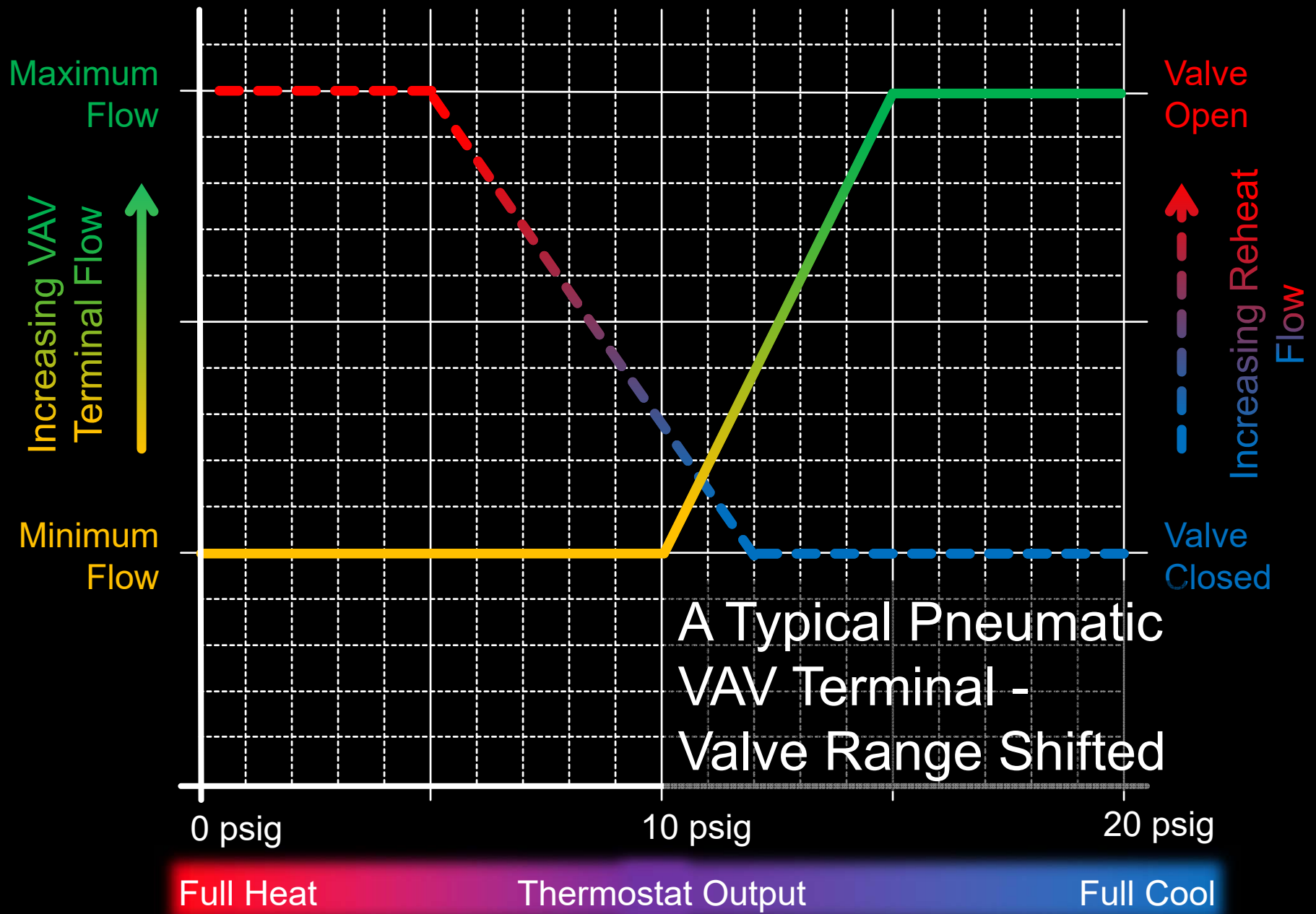
Full Cool

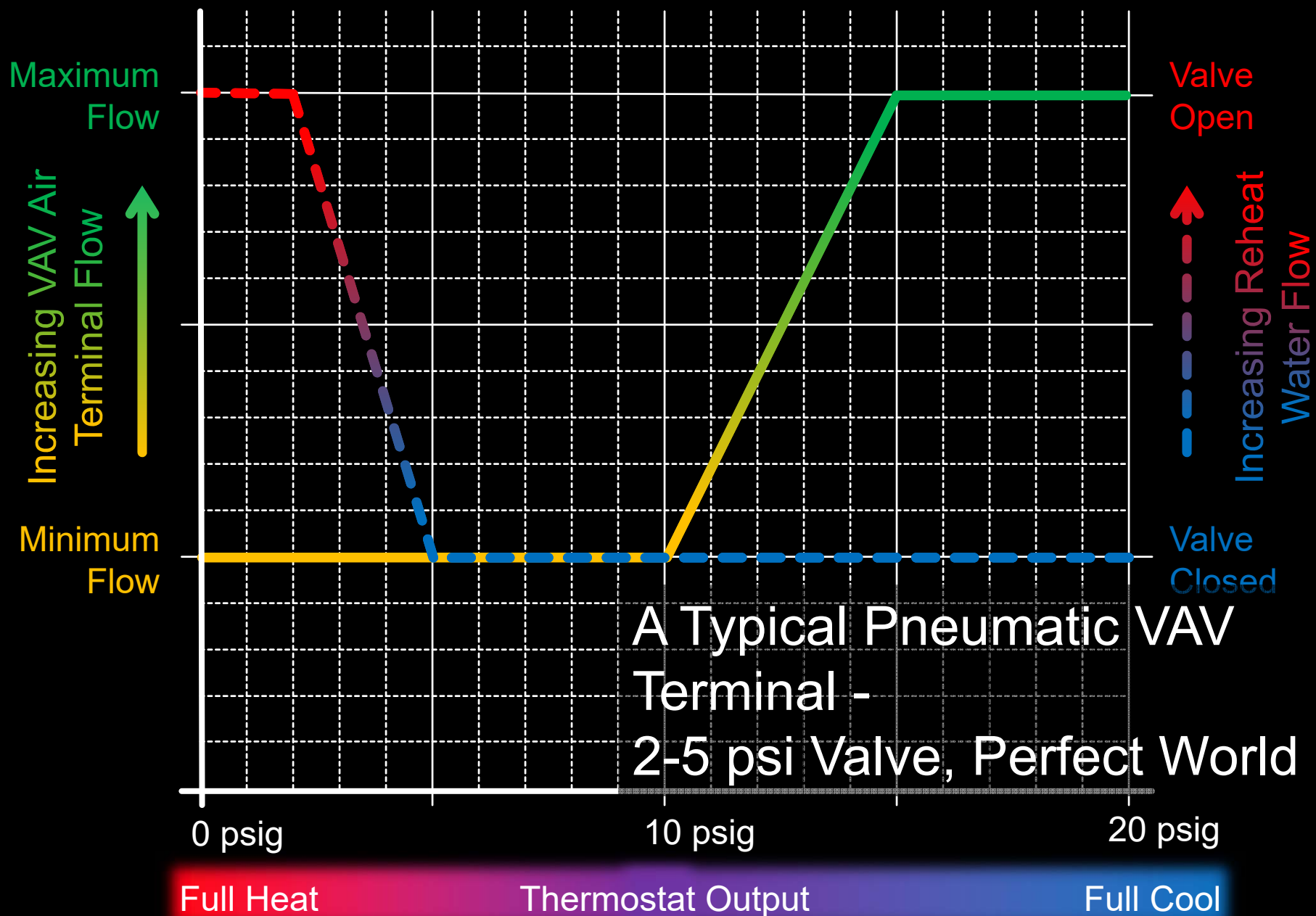


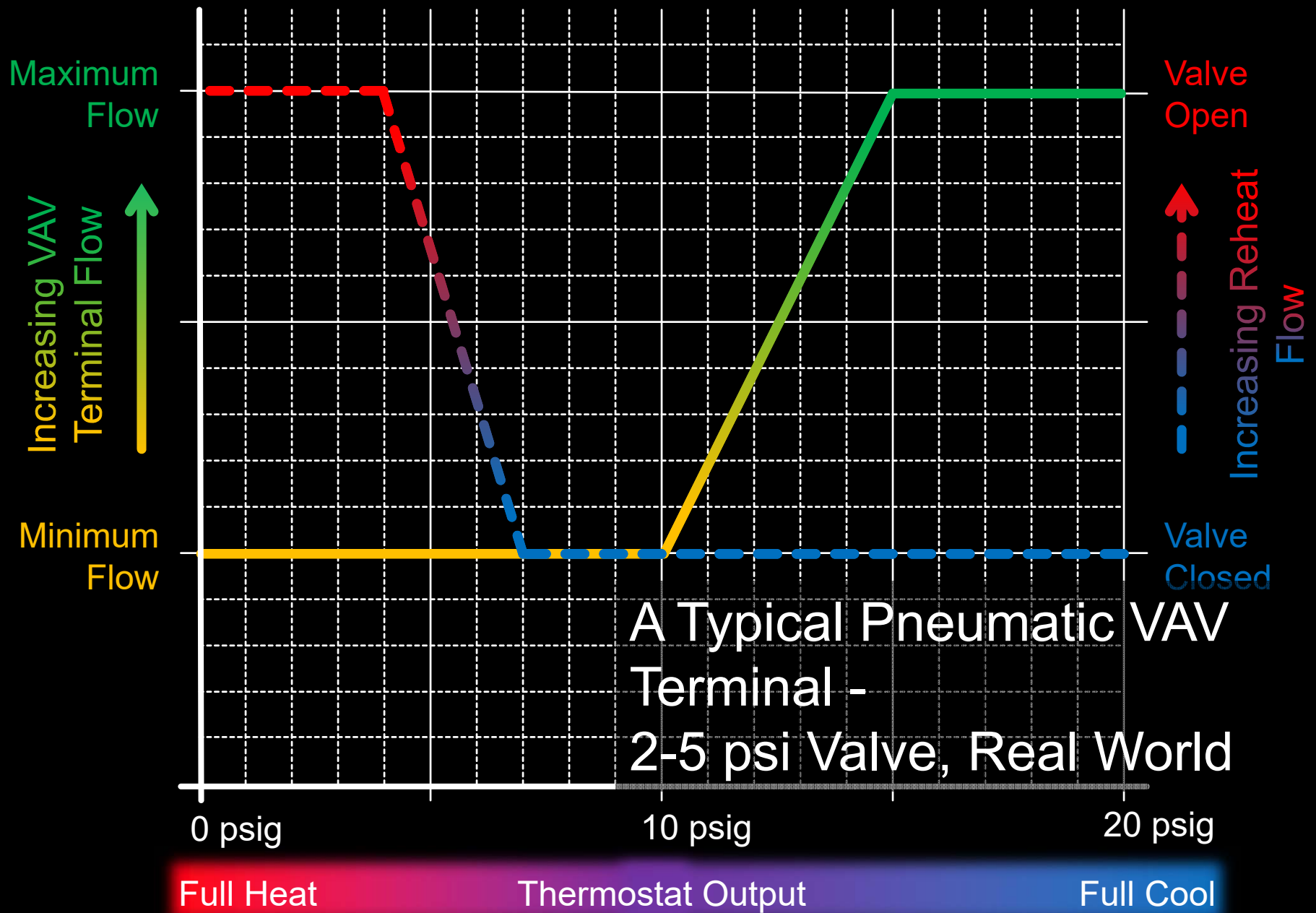
A Typical Pneumatic
VAV Terminal -
Reheat Valve
Operation
3-10 psi span











Dampers Can See Similar Variation in Power Required vs. Stroke

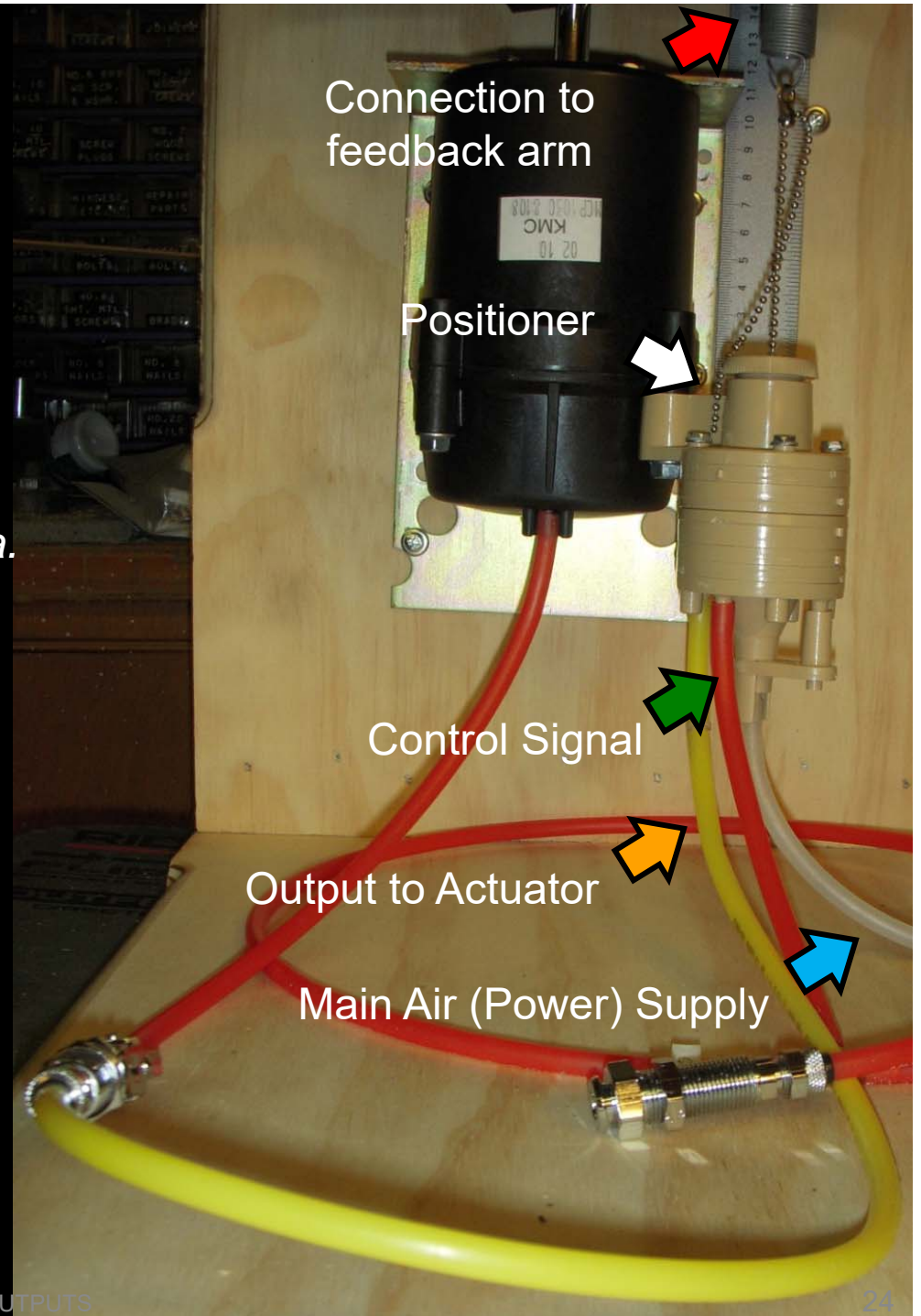
- Aerodynamic loads can vary with stroke
- Jamb seal loads can increase as the blades become aligned with the frame
- Blade seals require a very specific torque applied to the damper shaft to compress them and achieve rated leakage



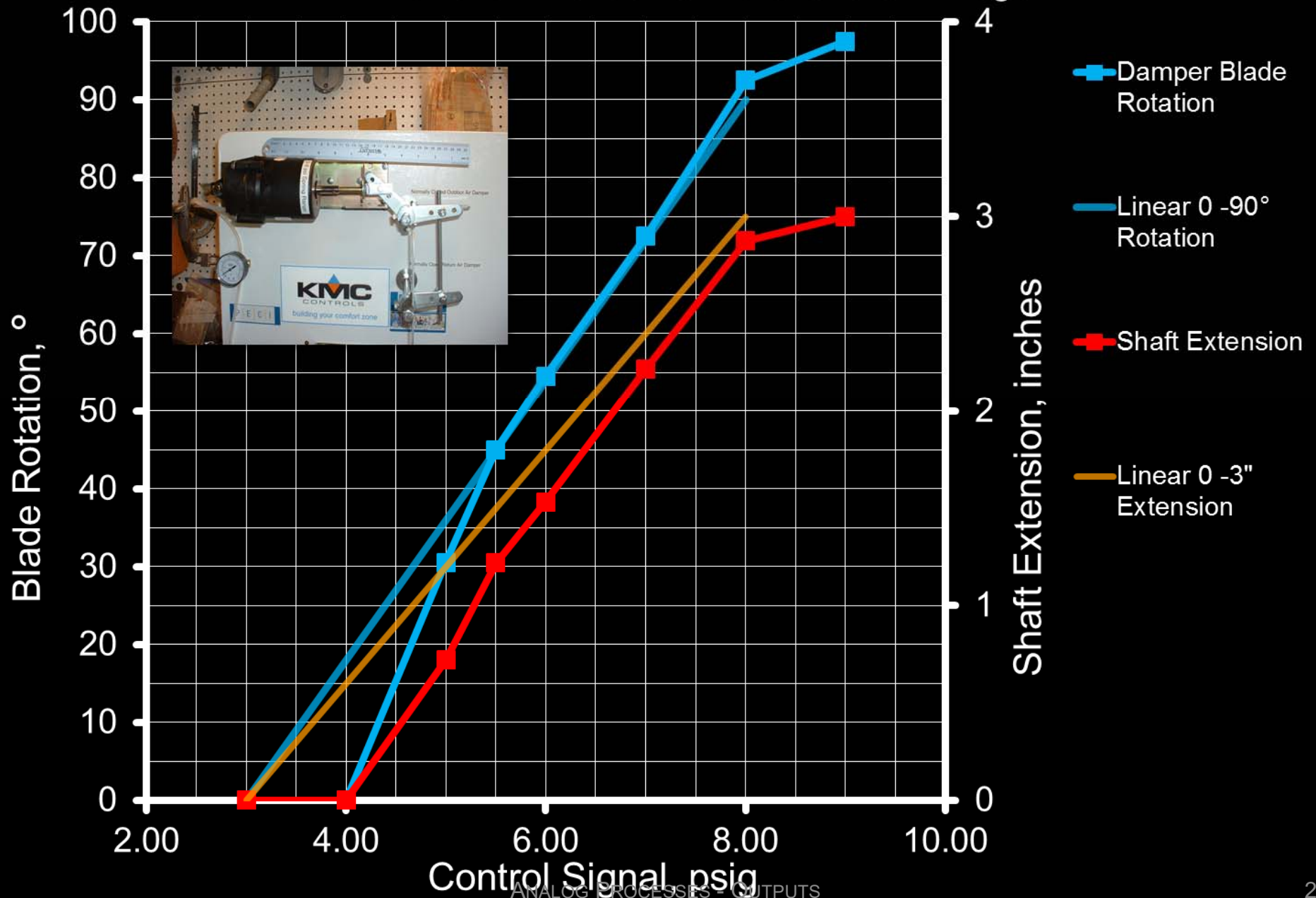
Positioners

Pneumatic and Electric

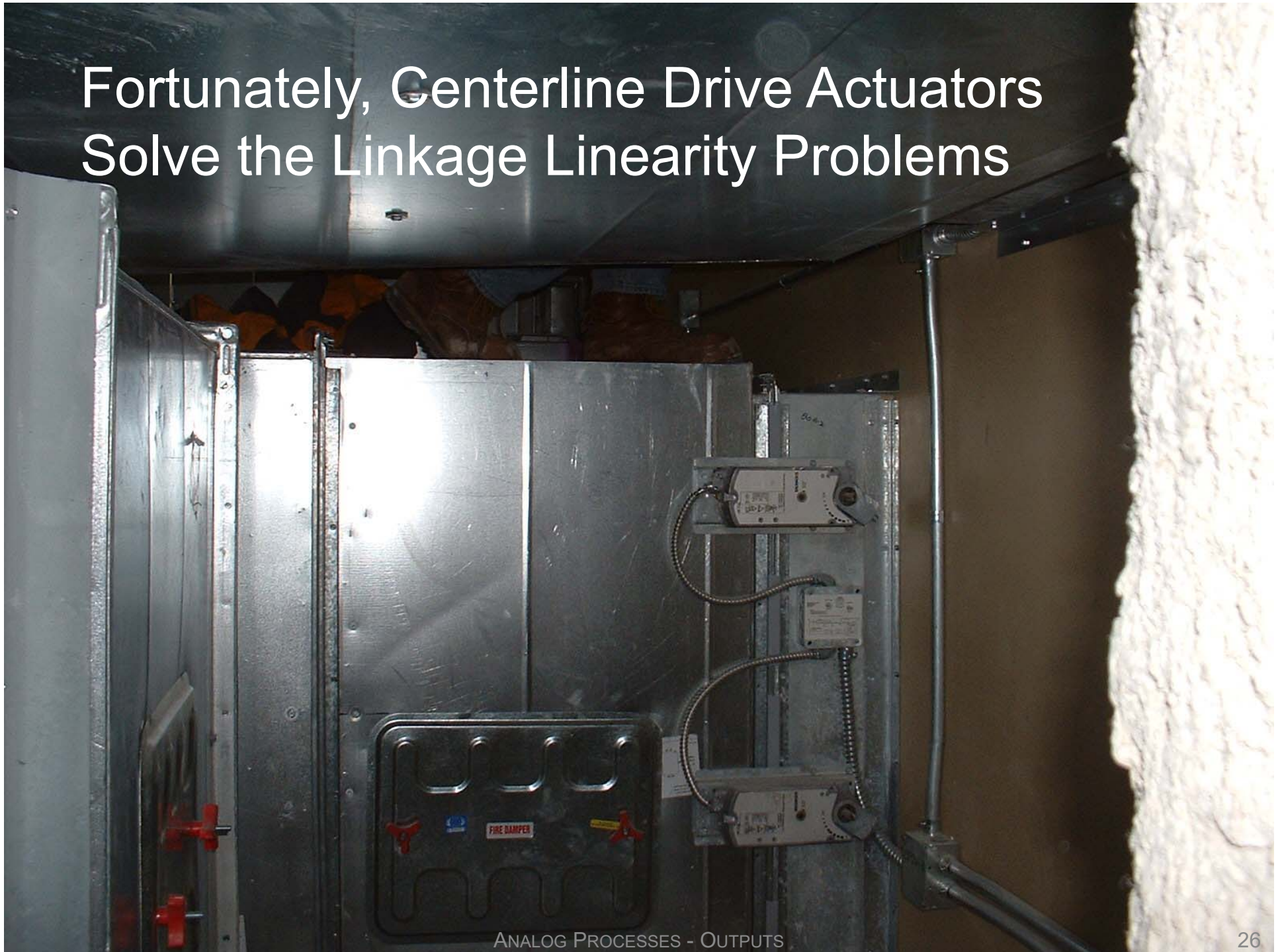
- Positive positioning relays ...
a.k.a “Positive positioners” a.k.a.
“Positioning relay” a.k.a.
“Positioners” ...
monitor actuator motion ...
... and compare it to the desired
actuator position
... and “do what ever it takes” to
get the actuator to move an
amount that corresponds to the
input change
- An independent energy source
provides the power to do this



Shaft Extension and Blade Rotation for a Rigid Mount Actuator with a Swivel Joint in the Shaft Linkage



Fortunately, Centerline Drive Actuators Solve the Linkage Linearity Problems



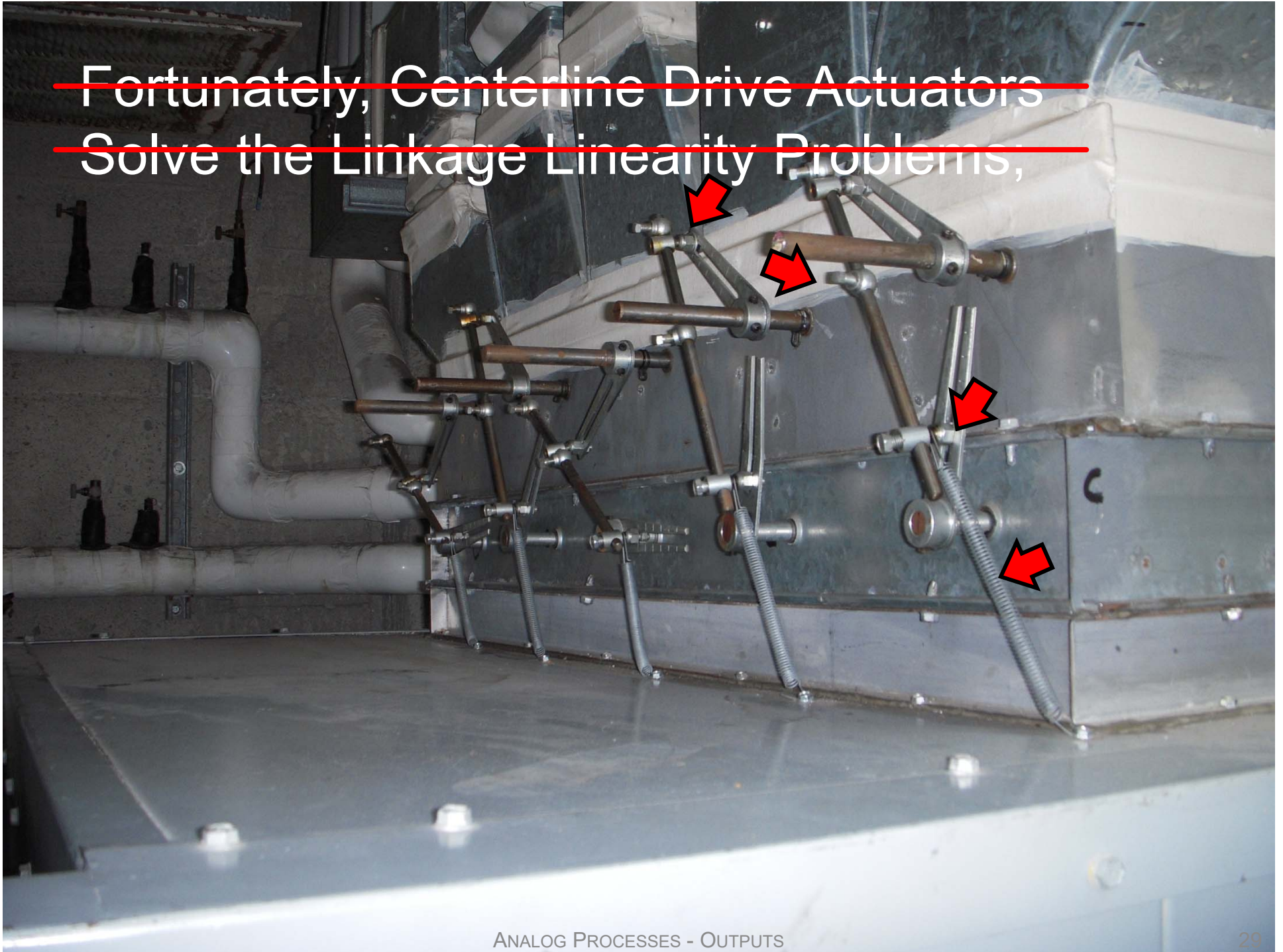
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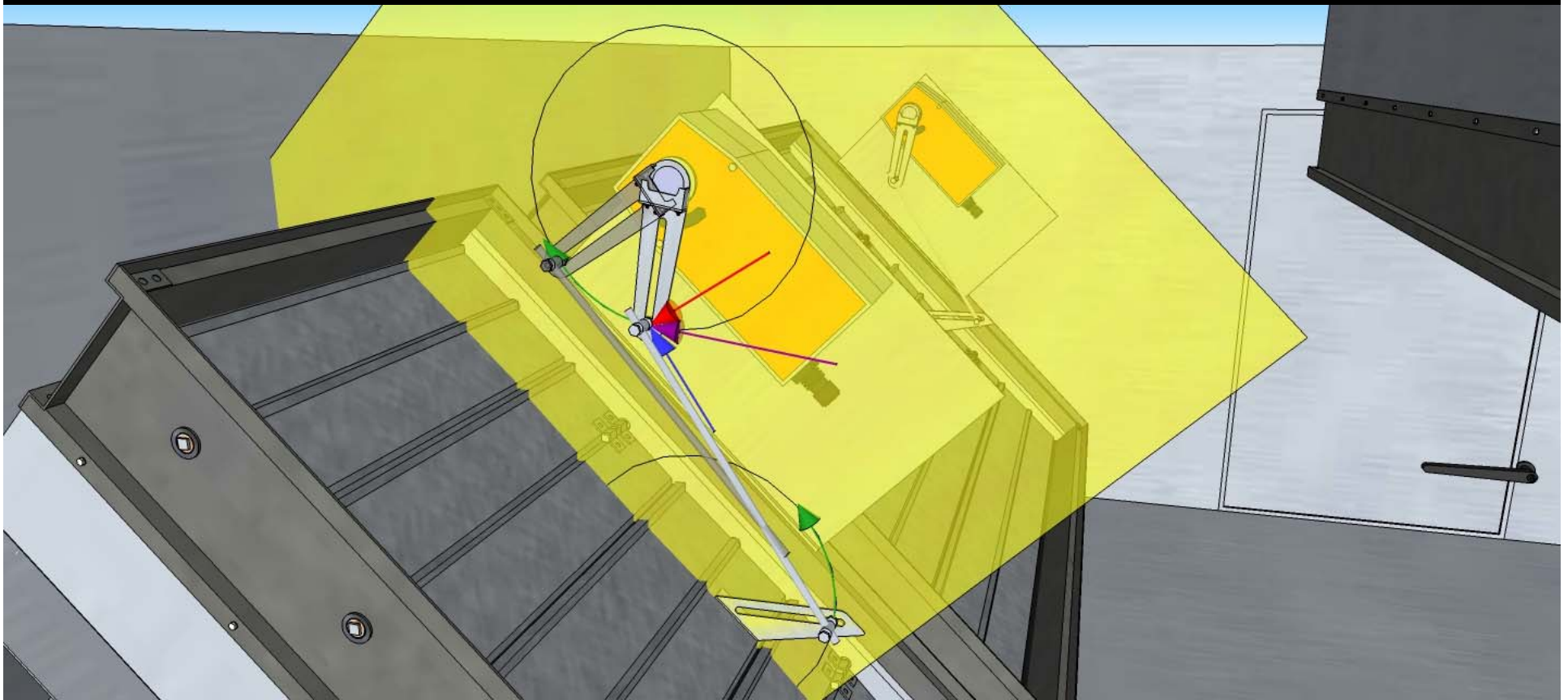
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Kinematics

The Physics of Motion

Kinematics can play a huge role in terms of actuator to damper linearity and actuator torque available vs. useful force at the damper blade



Actuators Actuate Final Control Elements

- Examples
 - Valves
 - Dampers
 - Variable Speed Drives
 - Centrifugal Machines
- Final control elements need to be properly applied
- Valves and dampers must be properly sized

